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*CORRESPONDENCE Aziz Ullah Awan, aziz.math@pu.edu.pk

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Linear and quadratic convection significance on the dynamics of MHD Maxwell fluid subject to stretched surface

Asia Ali Akbar¹, Aziz Ullah Awan¹*, Mutasem Z. Bani-Fwaz², ElSayed M. Tag-ElDin³, Kamel Guedri⁴, Mansour F. Yassen^{5,6} and Bagh Ali^{7,8}

¹Department of Mathematics, University of the Punjab, Lahore, Pakistan, ²Chemistry Department, College of Science, King Khalid University, Abha, Saudi Arabia, ³Faculty of Engineering and Technology, Future University in Egypt, New Cairo, Egypt, ⁴Mechanical Engineering Department, College of Engineering and Islamic Architecture, Umm Al-Qura University, Makkah, Saudi Arabia, ⁵Department of Mathematics, College of Science and Humanities in Al-Aflaj, Prince Sattam Bin Abdulaziz University, Al-Aflaj, Saudi Arabia, ⁶Department of Mathematics, Faculty of Science, Damietta University, New Damietta, Egypt, ⁷Department of Applied Mathematics, Northwestern Polytechnical University, Xi'an, China, ⁸Faculty of Computer Science and Information Technology, Superior University, Lahore, Pakistan

The heat transmission process is a prominent issue in current technology. It occurs when there is a temperature variation between physical processes. It has several uses in advanced industry and engineering, including power generation and nuclear reactor cooling. This study addresses Maxwell fluid's steady, twodimensional boundary layer stream across a linearly stretched sheet. The primary objective of this research is to investigate the impact of the non-Newtonian fluid parameter (Deborah number) on flow behavior. The secondary objective is to investigate the effect of linear and quadratic convection to check which model gives higher heat transfer. The flow is caused by the surface stretching. The mathematical model containing the underlying partial differential equations (PDEs) is built using the boundary layer estimations. The governing boundary layer equations are modified to a set of nonlinear ordinary differential equations (ODEs) using similarity variables. The bvp4c approach is employed to tackle the transformed system mathematically. The impacts of numerous physical parameters like stretching coefficient, mixed convective parameter, heat source/sink coefficient, magnetic coefficient, variable thermal conductance, Prandtl number, and Deborah number over the dimensionless velocity and temperature curves are analyzed via graphs and calculated via tables. After confirming the similarity of the present findings with several earlier studies, a great symmetry is shown. The findings show that the linear convection model gains more heat transport rate than the quadratic convection model, ultimately giving a larger thermal boundary layer thickness. Some numeric impacts illustrate that boosting the magnetic coefficient elevates the fluid's boundary layer motion, causing an opposite phenomenon of Lorentz force because the free stream velocity exceeds the stretched surface velocity.

KEYWORDS

magnetohydrodynamics, non-Newtonian fluid, stretching surface, quadratic convection, bvp4c technique

Introduction

Background study

Magnetohydrodynamics (MHD) is a subfield of fluid mechanics that analysis how an electrically conductive liquid moves in the vicinity of magnetism. Liao [1] inspected the exact solutions for the MHD stream of non-Newtonian fluid over an enlarging sheet. Datti et al. [2] explored the MHD stream and thermal transmission in a visco-elastic liquid across a nonisothermal expanding surface. Ishak et al. [3] analyzed the MHD stagnating point stream across an expanding sheet. Ahmad et al. [4] examined the MHD stream of a viscous liquid in a permeable medium across an exponentially stretched layer. Hayat et al. [5] provided an analytical solution for the MHD stream of a micropolar liquid caused by a curved elongating surface under the impact of homogeneousheterogeneous reactions. The transient MHD transportation of spinning Maxwell nanofluid stream over an extending surface with doubled diffusion and activating potential was exposed by Ali et al. [6]. Dawar et al. [7] explored the Lorentz forces and their effects on the Jaffrey nano liquid stream across a convectively warmed plain surface. Numerous researchers [8-11] have probed the MHD influence on fluid flow behavior.

The applications of non-Newtonian fluids cover a wide range of fundamental challenges in pharmaceutical, crude oil, polymer, and food production industries. Viscoelastic fluids are non-Newtonian fluids in which the imposed tension distribution is non-linearly related to velocity gradient. The Navier-Stokes equations and Newton's law of viscosity are insufficient for characterizing non-Newtonian liquids. Generally, the mathematical issues in non-Newtonian liquids are more complex than in Newtonian fluids because of severe nonlinearity and higher-order differential equations. As a result, several modern scientists are exploring various non-Newtonian liquid models under varied flow conditions. Exact solutions for the rotational stream of a modified Burgers fluid over cylindrical regions were proposed by Jamil and Fetecau [12]. Kamran et al. [13] disclosed the unstable rotating stream of fractional Oldroyd-B liquid between two infinite concentric cylinders. Ali et al. [14] performed the numerical assessment of MHD boundary layer stream and heat transport in micropolar liquids across a shrinkable sheet. Rehman et al. [15] probed the numerical solutions for a stream of Casson fluid caused by the rotating disk. Riaz et al. [16-19] have studied numerous non-Newtonian liquids and a detailed analysis of entropy production considering different flow configurations. Ali et al. [20] discussed Maxwell's unsteady (3-D) stream and tangentially hyperbolic nano liquid across a stretchy surface in the rotational frame.

Boundary layer flows across the stretchable sheet are essential for various applications, including glass blowing, cord depiction, copper spiraling, paper manufacture, glass blowing, extrusion, warm progressing, thermal conductivity of heat sinks, and melts of high molecular weight polymers. Sakiadis [21] illustrated the boundary layer stream at a continuous elongated surface having consistent speed. Wang [22] explored the viscid stream produced by a horizontally extended surface, considering suction and sliding phenomena. Khan et al. [23] scrutinized the twodimensional stream of Prandtl liquid across a stretchy surface and the influence of homogeneous-heterogeneous reactions. In the vicinity of a heat source, the MHD boundary layer stream and heating flux of nano liquid across a vertically expanded sheet was scrutinized by Ali et al. [24]. Awan et al. [25] inspected secondgrade liquid's MHD stagnation point stream over the oscillating extending sheet. Ali et al. [26] inspected the influence of bioconvection and chemical reaction over MHD nano liquid stream caused by exponentially enlarging surface. Awan et al. [27] performed the thermal inspection of an oblique stagnation point stream of second-grade liquid over an elongating surface with slip constraints.

The general form of convection is mixed (combined) convection, which is a composite of natural and driven convection in which a stream is governed concurrently by both an outer driving system and interior volumetric (body) stresses. They are used in various applications, including climatic boundary-layer streams, thermal radiators, solar cells, nuclear plants, and auto-electronic devices. Earlier researchers assumed a linear density change in the buoyancy force factor, which is typically appropriate with minor temperature variations. The interplay of forced and natural convection is most noticeable where the driven flow rate is low and temperature variations are enormous. Karwe and Jaluria [28] disclosed the transportation of fluid stream and combined convection from a movable plate in sliding and erosion processes. Vajravelu et al. [29] revealed the heat transmission from a dike using nonlinear convection across a permeable flat plate. Kumar and Sood [30] disclosed the combined impact of MHD and nonlinear convection in the stagnation point stream across a shrinkable surface. Hayat et al. [31] investigated the MHD influence on (3-D) nonlinear buoyancy stream of Oldroyd-B nano liquid across an extending sheet. Sampath Kumar et al. [32] inspected the stream of Jaffrey nano liquid, considering the impact of quadratic convection, MHD, nonlinear radiations, and multiple convected constraints. Patil and Kulkarni [33] probed the impact of surface ruggedness on MHD nonlinear convective nano liquid stream above a vertically moving plate. The comparative analysis for the linear and nonlinear buoyancy stream of hybrid nano liquid was explored by Kumar et al. [34].

After the aforementioned literature review, it is observed that no initiative has been taken to explore the energy and mass transport behavior of nonlinear mixed convection flow with nonhomogeneous material features and linear heat production around the boundary layer of a stretchy surface. Furthermore, nonlinear convection occurs due to nonlinear density temperature differences in the buoyant force term, which substantially influences flow behavior. Motivated by the

Bayones et al. [35], the present analysis is carried out to 1) analyze the rheology attributes of Maxwell fluid, 2) the significance of variable thermal conductance on the fluid temperature, 3) incorporate the influence of MHD, nonlinear convection, and heat absorption/generation. The underlying problem is solved numerically with the bvp4c technique in MATLAB. For linear and quadratic convection and varying temperature-dependent attributes, the velocity and temperature distributions and the coefficient of skin friction and heat flux are graphically explored. The validity of the numerical technique is probed by comparative tables with the existing research work. It is found that flow velocity enhances quadratic convection versus linear convection. On the other hand, the thermal boundary layer size and heat transport rate are higher for linear convection than in the quadratic model. The lower temperature for the quadratic convection model versus the linear model offers applications in manufacturing improved lubricants to minimize machinery degradation. Moreover, thermal designers can employ the current numeric outcomes in the industry because varying thermal conductance achieves higher efficiency than uniform thermal conductance.

Mathematical model

A time-independent, incompressible, two-dimensional, laminar, MHD flow of the Maxwell fluid across the stretched surface has been analyzed in the current flow problem. The impacts of varying thermal conductance, quadratic convection, and heat source/sink have also been incorporated. The cartesian coordinates are chosen with the horizontal axis pointing in the direction of the stretched surface and the vertical axis normal to the stretchy surface. The flow's composition is portrayed in Figure 1. To stabilize the boundary layer flow, a constant magnetic field B_0 is provided perpendicular to the sheet. The produced magnetic field is ignored since the magnetic Reynolds number is considered to be quite small. The enlarging velocity is supposed to be $U_w(\tilde{x}) = b\tilde{x}$ with b > 0, and the stream velocity is taken as $U_n(\tilde{x}) = a\tilde{x}$ with a > 0. The governing equations for the considered problem following [34–36] are stated as:

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0, \qquad (1)$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = v \frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + U_n \frac{dU_n}{d\tilde{x}} + (g\beta_1) (\tilde{T} - \tilde{T}_\infty) + (g\beta_2) (\tilde{T} - \tilde{T}_\infty)^2$$
$$-\lambda_1 \left[\tilde{u}^2 \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} + \tilde{v}^2 \frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + 2\tilde{u}\tilde{v} \frac{\partial^2 \tilde{u}}{\partial \tilde{x} \partial \tilde{y}} \right] - \frac{\sigma B_o^2}{\rho} (\tilde{u} - U_n),$$
(2)

$$\tilde{u} \ \frac{\partial \tilde{T}}{\partial \tilde{x}} + \tilde{v} \ \frac{\partial \tilde{T}}{\partial \tilde{y}} = \frac{1}{\left(\rho C_p\right)} \left[\frac{\partial}{\partial \tilde{y}} \left(k \left(\tilde{T} \right) \frac{\partial \tilde{T}}{\partial \tilde{y}} \right) \right] + \frac{Q_o}{\left(\rho C_p\right)} \left(\tilde{T} - \tilde{T}_{\infty} \right).$$
(3)



The suitable boundary constraints are stated as follows:

$$\begin{split} &\tilde{u}\left(\tilde{x},\tilde{y}\right) = U_{w}\left(\tilde{x}\right), \quad \tilde{T}\left(\tilde{x},\tilde{y}\right) = \tilde{T}_{w}\left(\tilde{x}\right) = \tilde{T}_{\infty} + c_{1}\tilde{x}, \quad \tilde{v}\left(\tilde{x},\tilde{y}\right) = 0, \quad \text{for} \quad \tilde{y} = 0, \\ &\tilde{u}\left(\tilde{x},\tilde{y}\right) \to U_{n}\left(\tilde{x}\right), \quad \tilde{T}\left(\tilde{x},\tilde{y}\right) \to \tilde{T}_{\infty} \quad \text{for} \quad \tilde{y} \to \infty \;. \end{split}$$

Where the velocity coefficients along \tilde{x} and \tilde{y} axes are indicated by \tilde{u} and \tilde{v} , respectively. The density of fluid is ρ , B_o is the intensity of applied magnetism, λ_1 is the fluid's relaxation time, σ is electrical conductance of fluid, (β_1, β_2) depict the coefficients of thermal expansion, the fluid's temperature is \tilde{T} , gstands for gravity, the heat diffusivity is $\hat{\alpha}$, (ρC_p) is the volumetric thermal capability, and Q_o reflect the heat generating coefficient. The ambient temperature and surface temperature are symbolized by \tilde{T}_{∞} and \tilde{T}_w respectively. The variable thermal conductance is defines as follows:

$$k(\tilde{T}) = \left(1 + \Lambda \frac{\tilde{T}_{\infty} - \tilde{T}}{\tilde{T}_{\infty} - \tilde{T}_{w}}\right)k.$$
(5)

The problem is simplified by using similarity variables following [35, 37, 38]:

$$\tilde{u} = f'(\xi)a\tilde{x}, \quad \tilde{v} = -\sqrt{a\nu}f(\xi), \quad \Theta(\xi) = \frac{\tilde{T} - \tilde{T}_{\infty}}{\tilde{T}_w - \tilde{T}_{\infty}}, \quad \xi = \tilde{y}\sqrt{\frac{a}{\nu}}.$$
(6)

where ξ is non-dimensional constraint, $f'(\xi)$ and $\Theta(\xi)$ are the dimensionless velocity and temperature distributions respectively. Using above mentioned transformation Eq. 1 is satisfied, and Eqs 2, 3 are altered into dimensionless ODEs.

$$f''' + 1 + f f'' - (f')^{2} + \beta (2f f' f'' - f^{2} f''') + \Omega (\Theta + Q_{c} \Theta^{2}) - M^{2} (f' - 1) = 0,$$
(7)

$$\Theta''(1 + \Lambda\Theta) + \Lambda(\Theta')^{2} + Prf\Theta' + PrS_{1}\Theta - Pr\Theta f' = 0.$$
(8)

The associated non-dimensional boundary conditions are stated as:

$$\begin{cases} f(\xi) = 0, & f'(\xi) = \varepsilon_1, \quad \Theta(\xi) = 1, \quad \text{for} \quad \xi \to 0, \\ f'(\xi) = 1, \quad \Theta(\xi) = 0, \quad \text{for} \quad \xi \to \infty. \end{cases}$$

$$(9)$$

In above mentioned equations, the prime (') denotes the differentiation with regard to the similarity variable ξ . Furthermore, M^2 denotes magnetic parameter, β is the Deborah number, Ω the mixed convection parameter, S_1 represents the heat generating/absorbing parameter, Pr is the Prandtl number, the heat conductance parameter is Λ , Q_c depicts quadratic convection parameter, and ε_1 is the velocity ratio coefficient. These parameters are stated as follows:

$$M^{2} = \frac{\sigma B_{o}^{2}}{\rho a} \quad \Omega = \frac{Gr_{\tilde{x}}}{\left(Re_{\tilde{x}}\right)^{2}}, \quad Gr_{\tilde{x}} = \frac{g\beta_{1}\left(\tilde{T}_{w} - \tilde{T}_{\infty}\right)\tilde{x}^{3}}{\nu^{2}}, \quad Re_{\tilde{x}} = \frac{a\left(\tilde{x}\right)^{2}}{\nu},$$
$$Q_{c} = \left(\frac{\beta_{2}}{\beta_{1}}\right)\left(\tilde{T}_{w} - \tilde{T}_{\infty}\right), \quad Pr = \frac{\mu C_{p}}{k}, \quad S_{1} = \frac{Q_{o}}{\left(\rho C_{p}\right)a}, \quad \varepsilon_{1} = \frac{b}{a}, \quad \beta = \lambda_{1}a.$$

It would be observed that $(Q_c > 0)$ reflects the non-linear convection, and $(Q_c = 0)$ represents linear mixed convection. Further, $(\varepsilon_1 > 0)$ for a stretching case, and $(\varepsilon_1 = 0)$ for the stationary surface. The skin frictional coefficient and Nusselt number are two important physical attributes. These quantities are stated as follows:

$$Cf_{\bar{x}} = \frac{\tilde{\tau}_w}{\rho U_n^2}, \qquad Nu_{\bar{x}} = \frac{\tilde{x}q_w}{k(\tilde{T})(\tilde{T}_w - \tilde{T}_\infty)}, \qquad (10)$$

where $\tilde{\tau}_w$ the wall shear stress, and q_w the wall heat influx are stated as:

$$q_w = -k \left(\frac{\partial \tilde{T}}{\partial \tilde{y}} \right) \bigg|_{\tilde{y}=0}, \qquad \tilde{\tau}_w = \mu \left(\frac{\partial \tilde{u}}{\partial \tilde{y}} \right) \bigg|_{\tilde{y}=0}.$$
(11)

By using Eqs 6, 11 in Eq. 10, the dimensionless forms of physical quantities are stated as:

$$Cf_{\tilde{x}}(Re_{\tilde{x}})^{1/2} = f''(0), \qquad Nu_{\tilde{x}}(Re_{\tilde{x}})^{-1/2} = -\frac{1}{(1+\Lambda\Theta)}\Theta'(0).$$
(12)

where $Re_{\tilde{x}} = \frac{a\tilde{x}^2}{v}$ depicts the local Reynolds number.

Numerical method

Due to the high non-linearity of the coupled ODEs (7–8), the analytical solution is not feasible. Various numerical methods are utilized in MATLAB to obtain a valid numerical solution. One of most significant method is bvp4c technique. The solution strategy is given as:

$$f(\xi) = Z_1, \ f'(\xi) = Z_2, \ f''(\xi) = Z_3, \ f'''(\xi) = Z_4, \ \Theta(\xi)$$
$$= Z_5, \ \Theta'(\xi) = Z_6, \ \Theta''(\xi) = Z_7,$$

Using the above substitution, the subsequent ODEs are defined as:

$$Z_4 = \frac{1}{(1 - \beta Z_1^2)} \left[(Z_2)^2 - Z_1 Z_3 + M^2 (Z_2 - 1) - 1 - \Omega (Z_5 - 1) \right]$$

$$+Q_c (Z_5)^2) - 2\beta Z_1 Z_2 Z_3],$$
(13)

$$Z_7 = \frac{1}{(1 + \Lambda Z_5)} \left[Pr(Z_5 Z_2 - Z_1 Z_6 - S_1 Z_5) - \Lambda (Z_6)^2 \right].$$
(14)

The boundary constraints are stated as:

$$Z_{1} = 0, \quad Z_{2} = \varepsilon_{1}, \quad Z_{5} = 1, \quad \text{for} \quad \xi \to 0, \\ Z_{2} = 1, \quad Z_{5} = 0, \quad \text{for} \quad \xi \to \infty.$$
(15)

Numerical findings and discussion

In current article, the flow phenomena arising due the linear stretching of expanding sheet for two-dimensional Maxwell fluid with variable thermal conductivity is studied. The numerical outcomes for the current study are calculated by solving the dimensionless ODEs (7-8) along with the boundary and initial constraints (9). The bvp4c method in MATLAB software is applied to obtain the numeric results. The estimation for the current analysis is settled by using the specified values for parameters: Pr = 2.0, M = 0.3, $\beta = 0.2$, $\Omega = 1.0$, $\varepsilon_1 = 0.3$, $S_1 =$ 0.2, Λ = 0.2. The ranges for the involved parameters are: $0.2 \le \beta \le$ $2.6, 0.2 \le \Lambda \le 2.2, 0.2 \le \varepsilon_1 \le 1.4, 0 \le M \le 8.6, -8 \le S_1 \le 2, 0.2 \le \Omega \le$ 1.4 and $0.7 \le \Pr \le 3$ are chosen to observe the smooth behavior of velocity and temperature curves. Tables 1-3 are constructed to compare our findings with already published papers. A notable coincidence has occurred, displaying the validity of the bvp4c method. Figures 2-8 are plotted to show the significance of distinct parameters over velocity $f'(\xi)$ and temperature $\Theta(\xi)$ profiles for two cases: (i) linear convection ($\Omega = 1, Q_c = 0$) (ii) quadratic convection ($\Omega = 1, Q_c = 2$).

Figure 2A,B reveal the influence of Deborah no. (β) on the dimensionless velocity $f'(\xi)$ and temperature $\Theta(\xi)$ profiles for both the linear and non-linear convection. Deborah no. is the quotient of fluids relaxation time and the time of observation. It is used to assess the fluidity of the substances under specified flow regimes. The case ($\beta = 0$) reduces to Newtonian fluid model. Raising the value of β enhances the velocity and thus the width of the boundary layer. The velocity fluctuates from zero to a maximal positive value, then gradually declines taking zero value outside the boundary layer. It is observed that the fluid accelerates more strongly for quadratic convection in comparison to the linear convection. An reverse effect is visualized for the temperature $\Theta(\xi)$, which descends slightly for the boosting β values. In the temperature boundary layer, it is depicted that the linear buoyancy $(Q_c = 0)$ provides more heat transfer than the quadratic buoyancy $(Q_c \neq 0)$.

	Hassanien and	Lok et al.	Ramachandran	Ali et al.	Our	
	Gorla [39]	[40]	et al. [41]	[37]	findings	
0.7	1.70632	1.7064	1.7063	1.7063	1.70632	
1.0	_	_	_	1.6754	1.67543	
7.0	_	1.5180	1.5179	1.5179	1.51791	
10	1.49284	_	_	1.4929	1.49284	
20	_	1.4486	1.4485	1.4485	1.44848	
40	_	1.4102	1.4101	1.4101	1.41006	
50	1.40686	_	_	1.3989	1.39893	
60	_	1.3903	1.3903	1.3903	1.39028	
80	_	1.3773	1.3774	1.3774	1.37739	
100	1.38471	1.3677	1.3680	1.3680	1.36804	

TABLE 1 Numerical evaluations for f''(0) with fluctuation in *Pr*.

Pr

 $\beta = M = \Lambda = \varepsilon_1 = Q_c = S_1 = 0, \ \Omega = 1$

TABLE 2 Numerical evaluations for $-\Theta'(0)$ with fluctuation in *Pr*.

Pr	$\beta = M = \Lambda = \varepsilon_1 = Q_0$	$\beta = M = \Lambda = \varepsilon_1 = Q_c = S_1 = 0, \ \Omega = 1$										
	Hassanien and	Lok et al.	Ramachandran	Ishak et al.	Our							
	Gorla [39]	[40]	et al. [41]	[38]	findings							
0.7	0.76406	0.7641	0.7641	0.7641	0.76406							
1.0	_	_	_	0.8708	0.87078							
7.0	_	1.7226	1.7224	1.7224	1.72238							
10	1.94461	_	_	1.9446	1.94462							
20	_	2.4577	2.4576	2.4576	2.45758							
40	_	3.1023	3.1011	3.1011	3.10109							
50	3.34882	_	_	3.3415	3.34146							
60	_	3.5560	3.5514	3.5514	3.55141							
80	_	3.9195	3.9095	3.9095	3.90950							
100	4.23372	4.2289	4.2116	4.2116	4.21168							

TABLE 3 Numerical evaluations for – $\Theta'(0)$ and $f^{\prime\prime}(0)$ with fluctuation in M.

β	=	Λ	=	Qc	=	0,	ε1	=	0.	5,	S ₁	=	1.0,	Pr	=	1.0), Ω	2 =	0.5	,
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	Bayones e	t al. [35]	Current results			
М	f''(0)	$-\mathbf{\Theta}'(0)$	f''(0)	- \Omega'(0)		
0.0	0.94096	0.5734	0.94091	0.5735		
1.0	1.06964	0.5887	1.06955	0.5888		
2.0	1.18340	0.5999	1.18341	0.6007		
4.0	1.38201	0.6186	1.38202	0.6184		
16	2.22152	0.6673	2.22157	0.6673		

Figures 3A,B exhibit the non-dimensional velocity and temperature curves for both the linear and quadratic convection terms for different values of variable thermal conductivity parameter Λ . The higher values of Λ improves the peak velocity of the fluid. Hence, the linear convection model performs as a lower bound. The thermal conductance of the liquid rises the non-dimensional temperature as a function of ξ , as illustrated in Figure 3B, for both linear and quadratic buoyancy terms. This is due to the fact that thermal conductivity measures the efficiency with which heat escapes from a fluid. As a result, heat from the sheet is estimated to be transmitted quickly into the primary stream due to an elevation in the thermal conductance value. The heat transmission at the boundary





layer is found to be greater for the linear convection than for the quadratic convection model.

Figures 4A,B depict the impact of velocity ratio coefficient ε_1 for both linear and non-linear buoyancy parameters. For larger values of ε_1 (when the stretching velocity U_w is greater than the free stream speed U_n), the fluid's speed decelerates in the boundary layers for both the linear and non-linear models. Figure 4B indicates the impact of ε_1 on $\Theta(\xi)$. It is noted that as ε_1 grows, the temperature curve and temperature boundary layer size for both linear and non-linear convection diminish. Also, linear buoyancy model generates more heat in comparison to the quadratic model.

Figures 5A,B are displayed to highlight the effect of magnetic coefficient *M* over the $f'(\xi)$ and $\Theta(\xi)$ profiles for both linear and non-linear convection. This is probed that the velocity of liquid improves for progressing amounts of *M*, which holds for both the

linear and quadratic buoyancy models. However, in an electrically conductive liquid, the transverse magnetic flux leads to a retardation force termed as the Lorentz force, that restricts the motion of the liquid in boundary layer. The opposite impact of *M* over the velocity profile, on the other hand, indicates that the free stream velocity exceeds the stretched surface speed. The temperature curve $\Theta(\xi)$ declines for higher amounts of magnetic coefficient *M*. It is noted that quadratic buoyancy model works as the lower bound for the heat transport than the linear convection model.

Figures 6A,B capture the trend of velocity and temperature curves for heat generation/absorbtion coefficient S_1 for the quadratic and linear buoyancy terms. It is explored that enhancing the heat generating coefficient improves both the velocity and the heat propagation of the fluid. The heat generating coefficient enables fluid surface to absorb this heat





and so remain effective. This consequence illustrates that the energy of the heating source is rather high, and the maximum temperature of the liquid is measured on the surface. As depicted in Figure 6B, the participation of a heat sink ($S_1 < 0$) causes decline in the thermal efficiency of the fluid, resulting in the formation of smaller thermal boundary layers. Furthermore, the linear convection model acquires higher heat transport rate in comparison to the quadratic convection model.

Figures 7A,B are plotted to depict the behavior of $f'(\xi)$ and $\Theta(\xi)$ for both linear and non-linear buoyancy terms along with diverse values of mixed convective coefficient Ω . Increasing values of the combined convection coefficient Ω indicate a stronger buoyant force, which results in more kinetic energy and an elevation in fluid speed, as plotted in Figure 7A, the velocity has growing impacts. As the quadratic convective coefficient grows ($Q_c > 0$), the maximum velocity improves, and the linear

case ($Q_c = 0$) acts as the lowest limit. On the other hand, Figure 7B depicts a very minor reduction in temperature profile as the combined convection parameter is enhanced. Furthermore, the linear buoyancy model serves as a upper bound in case of heat transport as compared to the quadratic buoyancy.

The tendency of velocity and temperature curve for distinct values of Prandtl no. is described in Figures 8A,B for the linear and non-linear Boussinesq estimations. The velocity curve depicts a declining trend for higher Pr values. However, quadratic buoyancy model attains larger thickness of boundary layer in comparison to the linear convection. Physically, as the Prandtl number raises, the heat diffuseness reduces, resulting in a reduction in power capacity, which lessens the temperature boundary layer, as displayed in Figure 8B. Consequently, heat is dissipated more quickly, and hence, temperature boundary layer width and temperature of the





liquid both diminish. As a result, the Prandtl number is employed to enhance the cooling tendency of fluids. It is examined that linear convection gains more heat exchange rate than the quadratic convection model.

Numerical assessment of skin friction and nusselt number

Figures 9A,B are plotted to signify the importance of Deborah no. (β) and magnetic coefficient (M) on coefficient of skin friction $Cf_{\bar{x}}(Re_{\bar{x}})^{0.5}$ and Nusselt no. $Nu_{\bar{x}}(Re_{\bar{x}})^{-0.5}$. It is exposed that f''(0) and $-\Theta'(0)$ grow up for higher values of M and β . It is visualized that quadratic convection gains higher values of f''(0) and $-\Theta'(0)$ as compared to linear buoyancy model. Figures 10A,B characterize the behavior of $Cf_{\bar{x}}(Re_{\bar{x}})^{-0.5}$ for progressing values of Λ and ε_1 . It is probed

that coefficient of skin friction has descending trend for accelerating values of ε_1 and Λ for both the linear and quadratic convection. However, the linear buoyancy model acts as lower bound against higher values of ε_1 and Λ in comparison to the quadratic buoyancy model. Figure 10B depicts the increasing fashion of wall heat transfer coefficient for the enhancing values of ε_1 and Λ . The heat flux coefficient accelerates more quickly for quadratic convection as compared to linear convection.

Concluding remarks

This research has investigated Maxwell fluid's boundary layer stream and thermal efficiency across a linear stretchy surface along with MHD, nonlinear convection, and varying thermal conductance. The dimensionless nonlinear ODEs are tackled numerically in MATLAB utilizing the bvp4c technique. The numerical findings







for the non-dimensional velocity curve, temperature curve, skin friction, and Nusselt number are displayed via graphs and verified through tables. Some crucial outcomes are listed here:

- Increment in the magnetic coefficient (*M*), varying thermal conductance (Λ), and Deborah number (β) value amplified the axial velocity field for both the linear and quadratic convection. However, the fluid temperature diminishes for higher inputs of β and *M* but amplifies for larger values of Λ in both linear and nonlinear convection.
- When the velocity ratio coefficient ε₁ is raised, the extending velocity becomes greater than the free streaming speed, reducing the temperature and velocity fields.
- Higher inputs of Ω depict stronger buoyant force. Hence, the velocity field boosts, and temperature declines for larger inputs of Ω .
- As the heat-generating parameter S_1 is elevated, the velocity and thermal profiles increase for both linear and quadratic buoyancy.
- The thermal boundary layer width declines with increment in Pr no. values due to the reduction in thermal diffuseness for both linear and nonlinear buoyancy.
- The coefficient of skin friction $Cf_{\bar{x}} (Re_{\bar{x}})^{1/2}$ and Nusselt no. $Nu_{\bar{x}} (Re_{\bar{x}})^{-1/2}$ enhance for the larger inputs of *M* and β for both linear and quadratic convection. However, the elevating fashion of Λ and ε_1 has decreased the skin friction but increased the Nusselt number.
- The linear convection model attains a higher heat transport rate than the quadratic convection model.

Data availability statement

The original contributions presented in the study are included in the article/supplementary material, further inquiries can be directed to the corresponding author.

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Author contributions

All authors listed have made a substantial, direct, and intellectual contribution to the work and approved it for publication. AAA: Formal analysis, investigation and writing-original draft. AUA: Formal analysis and supervision. MZB-F: Funding acquisition, software and revision. EMT-E: Funding acquisition, validation and revision. KG: Funding acquisition, resources and revision. MFY: Funding acquisition, validation and revision. BA: Conceptualization, editing and revision.

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Conflict of interest

The authors declare that the research was conducted in the absence of any commercial or financial relationships that could be construed as a potential conflict of interest.

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Nomenclature

 \tilde{x}, \tilde{y} Cartesian coordinates (m) \tilde{u}, \tilde{v} Velocity components along x and y – axes (m/s) $U_n(\tilde{x})$ Free stream velocity (m/s) $U_w(\tilde{x})$ Stretching velocity at the surface (m/s) **g** Gravity acceleration (m/s^2) \tilde{T} Temperature of the fluid (K) \tilde{T}_w Temperature at wall (K) $ilde{T}_{\infty}$ Ambient temperature (K) Pr Prandtl no. (--) B_0 Applied magnetism ($kg/s^2.A$) **Q**₀ Heat source/sink (*J/K.m³.s*) $Nu_{\tilde{x}}$ Nusselt number (--) M² Magnetic parameter (--) Q_c Quadratic convection parameter (--) C_p Specific heat (J/kg.K) $Re_{\tilde{r}}$ Reynolds number (--) $Gr_{\tilde{x}}$ Grashof number (---)

 $Cf_{\tilde{x}}$ Coefficient of skin friction (--) f Non-dimensional velocity q_w Wall heat flux (W/m^2)

Greek symbols

μ Viscosity of fluid (kg/m.s) ρ Density of fluid (kg/m^3) $λ_1$ Fluid's relaxation time (s) $(β_1, β_2)$ Coefficients of thermal expansion (1/K) σ Electric conductivity $(A^2.s^3/kg.m^3)$ $\hat{α}$ Heat diffusivity (m^2/s) ξ Dimensionless variable (--) β Deborah number (--) $ε_1$ Velocity ratio parameter (--) Ω Mixed convection parameter (--) $\tilde{τ}_w$ Shear stress at wall $(kg/m.s^2)$ Θ Non-dimensional temperature (--)