

# **Component-Based Reduced Order Modeling of Large-Scale Complex Systems**

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Large-scale engineering systems, such as propulsive engines, ship structures, and wind farms, feature complex, multi-scale interactions between multiple physical phenomena. Characterizing the operation and performance of such systems requires detailed computational models. Even with advances in modern computational capabilities, however, high-fidelity (e.g., large eddy) simulations of such a system remain out of reach. In this work, we develop a reduced-order modeling framework to enable accurate predictions of large-scale systems. We target engineering systems which are difficult to simulate at a high-enough level of fidelity, but are decomposable into different components. These components can be modeled using a combination of strategies, such as reduced-order models (ROM) or reduced-fidelity full-order models (RF-FOM). Component-based training strategies are developed to construct ROMs for each individual component. These ROMs are then integrated to represent the full system. Notably, this approach only requires high-fidelity simulations of a much smaller computational domain. System-level responses are mimicked via external boundary forcing during training. Model reduction is accomplished using model-form preserving least-squares projections with variable transformation (MP-LSVT) (Huang et al., Journal of Computational Physics, 2022, 448: 110742). Predictive capabilities are greatly enhanced by developing adaptive bases which are locally linear in time. The trained ROMs are then coupled and integrated into the framework to model the full large-scale system. We apply the methodology to extremely complex flow physics involving combustion dynamics. With the use of the adaptive basis, the framework is demonstrated to accurately predict local pressure oscillations, time-averaged and RMS fields of target state variables, even with geometric changes.

Keywords: reduced order modeling, domain decomposition, model reduction, turbulent reacting flows, adaptive basis

# **1 INTRODUCTION**

Rapid advancements in computing technologies are enabling high-fidelity simulations of complex, multi-scale physics (e.g., turbulence [1] and combustion [2,3]) observed in real engineering systems. These simulations provide insight into the underlying physics, which cannot be quantitatively accessed through experiments. This insight is useful in improving the performance of engineering

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systems and reducing failures. However, the high computational costs of high-fidelity simulations prohibit their integration into design and analysis of full-scale systems, which require repeated simulations to explore parameter spaces. One popular approach to address such challenges is through model order reduction (MOR), a common approach being projection-based reducedorder models (ROM) [4-6], which have demonstrated success in many applications such as flow control [7-9], aeroelasticity [10,11], hypersonics [12], and combustion [13,14]. Typically, the construction of ROMs involves three stages: 1) an offline training stage that performs high-fidelity simulations of the target systems for multiple parameters; 2) offline construction of reduced basis and projections on low-dimensional manifolds; and 3) online execution of ROMs by projecting the governing equations on the low-dimensional manifold. Despite the many successful examples of MOR, their direct applications in many practical large-scale engineering systems remain infeasible because the systems are so complex that high-fidelity simulations are completely inaccessible. Using an example from rocket combustion, a coarse-mesh ("low"fidelity) large eddy simulation (LES) of a small-scale rocket engine [15] requires  $O(10^7)$  CPU-hours, which even makes a single finemesh (high-fidelity) LES of this type of problems inaccessible, (estimated to require  $>O(10^9)$  CPU-hours), let alone the highfidelity LES of a large-scale rocket engine [16], a computation that would require > 10 times the resources of the small-scale problem.

To address this specific challenge of the lack of full-order model (FOM) data for large-scale systems, researchers have formulated domain-decomposition methods [17,18], or component-based methods [19] to develop a network of ROMs to model the target system. In addition to ROM applications, such ideas have been commonly used for computational fluid dynamics [20,21], porthamiltonian system [22-24] etc. For consistency, we refer to this family of approaches as component-based reduced-order modeling (CBROM) methods in the current paper. CBROM methods leverage the fact that many large-scale engineering systems can be decomposed into components of identical features and the offline training of the ROMs can be performed based on each individual component for multiple parameters, which significantly reduces the cost of the offline training. The trained component-based ROMs can then be used for the identical components and coupled together to model different configurations of the large-scale systems.

To date, the majority of the success of CBROM methods has been in problems governed by linear PDEs. Willcox et al. [25] demonstrated the feasibility of constructing low-order models of blade row unsteady aerodynamics in a compressor. Maday and Ronquist [26] formulated a reduced basis element method and applied it to a thermal fin problem. Iapichino et al. [27] proposed a reduced basis hybrid method to solve the steady Stokes problem with applications to cardiovascular networks. Adopting the staticcondensation reduced-basis-element method [19,28,29], Kapteyn et al. [30] demonstrated the development of a digital twin for a 12-ft wingspan unmanned aerial vehicle via a library of componentbased ROMs. More recently, McBane and Choi [31] leveraged the static-condensation reduced-basis-element method and demonstrated a 1000× speedup with relative error < 1% for lattice-type structure design using component-wise reduced-order modeling.

In addition, some applications of component-based ROMs on nonlinear PDEs can also be found in the literature. One group of studies incorporate the FOM to model a subset of components in the target system while applying ROMs to the other components. Lucia et al. [32] demonstrated a combination of ROMs and FOM by domain-decomposition for modeling two-dimensional high-speed flows with moving shock waves, by applying the FOM for the shockcontaining domain and ROMs for the other domains. Buffoni et al. [33] demonstrated similar ideas by partitioning the computational domain into two subdomains (one modeled using FOM while the other by ROM), and presented different approaches to couple ROM with FOM. Baiges et al. [34] demonstrated the improvement in predicting flow configurations that are not present in the training snapshots by integrating the FOM into the component-based modeling framework. Ahmed et al. [35] presented a hybrid analysis and modeling approach combining a physics-based FOM and a data-driven non-intrusive ROM towards predictive digital twin technologies. Another group of investigations aim at incorporating only ROMs rather than hybrid FOMs/ROMs in the componentbased modeling framework. Hoang et al. [36] proposed the domaindecomposition least-squares Petrov-Galerkin (DD-LSPG) modelreduction method for parameterized systems of nonlinear algebraic equations. Xiao et al. [37-39] developed a domaindecomposition non-intrusive reduced-order model (DDNIROM) for turbulent flows. The current authors demonstrated the integration of component-based ROMs with a FOM in a quasi-1D Euler problem [40,41].

In the present work, we develop a component-based modeling framework that can flexibly adopt either reduced-order models (ROM) or full-order models (FOM) for different components of the target system based on the corresponding requirements of modeling accuracy and efficiency with the goal of enabling:

- (1) Accurate simulations of large-scale systems, which cannot be directly accessed using high-fidelity simulations;
- (2) Parametric studies of the large-scale system targeting manyquery applications.

It is notable that in the current work, we choose rocket engine as the target system. This application involves compressible, reacting, chaotic flows and thus introduces complex challenges for reduced order modeling. We establish a component-based training strategy for ROM development, which only requires the high-fidelity simulations of the individual components, rather than the entire system. The trained ROMs are then coupled together (either with each other or with FOM) via a direct flux matching method for information transfer between components. The ROM formulation leverages model reduction techniques using model-form preserving least-squares projections with variable transformation (MP-LSVT) with physical realizability enforced on both temperature and species mass fractions [14] to achieve both global and local stabilization. Furthermore, the MP-LSVT ROM is incorporated with basis adaptation to achieve significant enhancement in modeling accuracy. Since our interests are focused on engineering systems involving combustion, we use extremely challenging turbulent reacting flow examples, relevant to rocket applications, to motivate and evaluate our framework. But it should be highlighted that our

component-based modeling framework is applicable to many other, unrelated disciplines, featuring systems that can be decomposed into different components.

The remainder of the paper is organized as follows. Section 2 presents the full-order model (FOM) and time discretization. Section 3 reviews the procedure for model reduction *via* MP-LSVT formulation. Section 4 discusses basis-adaptation algorithms for ROM enhancement. Section 5 presents the domain-decomposition framework, including both the component-based ROM training strategy and integration method in full system. Section 6 presents numerical results based on single- and multi-injector model rocket combustor configurations with detailed assessment on the accuracy of the framework. In Section 7, we provide concluding remarks and perspectives.

## 2 FULL-ORDER MODEL

We define the physical domain  $\Omega$  with boundary  $\partial \Omega$ , and then represent the governing equations of the full-order model (FOM) for  $\Omega$  as a generic dynamical system

$$\frac{d\mathbf{q}(\mathbf{q}_p)}{dt} = \mathbf{f}(\mathbf{q}_p, \mathbf{t}) \text{ in } \Omega,$$
  
with  $\mathbf{u}(\mathbf{q}_p) = \mathbf{u}_{BC} \text{ on } \partial\Omega,$   
and  $\mathbf{q}_p(\mathbf{t}=0) = \mathbf{q}_p^{0},$  (1)

where  $t \in [0, T]$  is the solution time, which spans the time interval from 0 to T,  $\mathbf{q}_p$ :  $[0, T] \to \mathbb{R}^N$  is the vector of solution (or state) variables,  $\mathbf{u}_{BC}$ :  $[0,T] \to \mathbb{R}^{N_b}$  is the vector of states to be enforced at the boundary  $\partial \Omega$  (i.e., boundary conditions),  $\mathbf{q}_p^0 \in \mathbb{R}^N$  is the vector of states to be specified as the initial conditions at t = 0,  $\mathbf{q}: \mathbb{R}^N \to \mathbb{R}^N, \ \mathbf{f}: \mathbb{R}^N \times [0,T] \to \mathbb{R}^N, \ \text{and} \ \mathbf{u}: \mathbb{R}^N \to \mathbb{R}^{N_b} \ \text{are}$ (typically highly non-linear) functions of  $\mathbf{q}_p$ . N is the total number of degrees of freedom in the system (e.g., for finite volume/element method,  $N = N_{elem} \times N_{var}$ , where  $N_{elem}$  is the total number of elements and  $N_{var}$  is the number of state variables in each element).  $N_b$  is the total number of degrees of freedom associated with the boundary  $\partial \Omega$  and  $N_b = N_{elem,BC} \times N_{var}$ , where  $N_{elem,BC}$  is the number of elements adjacent to the boundary  $\partial \Omega$ . For a FOM based on conservation laws, the function, q, represents the conservative state. The function, f, represents surface fluxes, source terms, and body forces arising from the spatial discretization of the governing equations. u represents the boundary condition state, and  $\mathbf{u}_{BC}$  denotes the values of the state to be satisfied at the boundary.

Different time-discretization methods can be introduced to solve Eq. 1 (e.g., linear multi-step, or Runge-Kutta methods [42]). For all the numerical examples presented in the current paper, we use linear multi-step methods for both FOM and ROM calculations and refer the reader to [14] for details. An *l*-step version of linear multi-step methods can be expressed as

$$\mathbf{r}(\mathbf{q}_{p}^{n}) \triangleq \mathbf{q}(\mathbf{q}_{p}^{n}) + \sum_{j=1}^{l} \alpha_{j} \mathbf{q}(\mathbf{q}_{p}^{n-j}) - \Delta t \beta_{0} \mathbf{f}(\mathbf{q}_{p}^{n}, \mathbf{t}^{n}) - \Delta t \sum_{j=1}^{l} \beta_{j} \mathbf{f}(\mathbf{q}_{p}^{n-j}, \mathbf{t}^{n-j}) = 0 \quad (n \ge l),$$
  
with  $\mathbf{u}(\mathbf{q}_{p}^{n}) = \mathbf{u}_{BC}^{n}$  on  $\partial \Omega.$  (2)

where  $\Delta t \in \mathbb{R}^+$  is the physical time step for the numerical solution, and the coefficients  $\alpha_j$ ,  $\beta_j \in \mathbb{R}$  are determined based on *l*. If  $\beta_0 = 0$ , the method is explicit; otherwise, the method is implicit. **r**:  $\mathbb{R}^N \to \mathbb{R}^N$  is defined as the FOM equation residual. The state variables,  $\mathbf{q}_p^n$ , are solved for at each time step so that  $\mathbf{r}(\mathbf{q}_p^n) = \mathbf{0}$ .

# 3 MODEL-FORM PRESERVING MODEL REDUCTION FOR TRANSFORMED SOLUTION VARIABLES

For problems involving multiscale phenomena with strong convection and non-linear effects, it is well-recognized that ROM robustness can be a major issue. To address this challenge, we pursue the model-form preserving leastsquares with variable transformation (MP-LSVT) formulation to construct the reduced-order model (ROM). This methodology is described below—we refer the reader to ref [14] for further details.

## 3.1 Construction of Proper Orthogonal Decomposition Bases for Solution Variables

The state  $\mathbf{q}_p$  in **Eq. 1** can be expressed in a trial space  $\mathcal{V}_p \triangleq \operatorname{Range}(\mathbf{V}_p)$ , where  $\mathbf{V}_p \in \mathbb{R}^{N \times n_p}$  is the trial basis matrix. Define  $\mathbf{q}'_p(t) \triangleq \mathbf{q}_p(t) - \mathbf{q}_{p,\text{ref}}$ , where  $\mathbf{q}_{p,\text{ref}}$  is a reference state. Possible reference states include the initial FOM solution,  $\mathbf{q}_{p,\text{ref}} = \mathbf{q}_p(t) = \mathbf{t}_0$ , or the time-averaged FOM solution,  $\mathbf{q}_{p,\text{ref}} = \frac{1}{t_1-t_0} \int_{t_1}^{t_1} \mathbf{q}_p(t) dt$ . We then seek a representation  $\tilde{\mathbf{q}}_p$ :  $[0,T] \to \mathcal{V}_p$  such that

$$\mathbf{H}\left(\tilde{\mathbf{q}}_{p}-\mathbf{q}_{p,\text{ref}}\right)=\mathbf{V}_{p}\mathbf{q}_{r}$$
(3)

where  $\mathbf{q}_r: [0, T] \to \mathbb{R}^{n_p}$  is the reduced state with  $n_p$  representing the number of trial basis modes. In this work,  $\mathbf{V}_p$  is computed *via* the proper orthogonal decomposition (POD) [4] from the singular value decomposition (SVD), which is a solution to

$$\min_{\mathbf{V}_p \in \mathbb{R}^{N \times n_p}} \| \mathbf{Q} - \mathbf{V}_p \mathbf{V}_p^T \mathbf{Q} \|_F \quad s.t. \quad \left( \mathbf{V}_p \right)^T \mathbf{V}_p = \mathbf{I},$$
(4)

where  $\mathbf{Q}$  is a data matrix in which each column is a snapshot of the solution  $\mathbf{q}'_p$  at different time instances. A scaling matrix  $\mathbf{H} \in \mathbb{R}^{N \times N}$  must be applied to  $\mathbf{q}'_p$  such that the variables corresponding to different physical quantities in the data matrix  $\mathbf{Q}$  have similar orders of magnitude. Otherwise,  $\mathbf{Q}$  may be biased by physical quantities of higher magnitudes (e.g., total energy). In this work, we normalize all quantities by their  $L^2$ norm, as proposed by Lumley and Poje [4].

$$\mathbf{H} = diag(\mathbf{H}_1, \dots, \mathbf{H}_i, \dots, \mathbf{H}_{N_{elem}}), \tag{5}$$

where  $\mathbf{H}_i = diag(\phi_{1,norm}^{-1}, \dots, \phi_{N_{var},norm}^{-1})$ . Here,  $\phi_{v,norm}$  represents the *v*th state variable and

$$\phi_{\nu,norm} = \frac{1}{t_1 - t_0} \int_{t_0}^{t_1} \frac{1}{\Omega} \int_{\Omega} \phi_{\nu}^{'2}(\mathbf{x}, t) \, d\mathbf{x} \, dt.$$
(6)

# **3.2 Least-Squares With Variable Transformation**

Leveraging the least-squares Petrov-Galerkin (LSPG) projection formulation proposed by Carlberg et al. [43], we develop a modelform preserving least-squares formulation for the FOM in **Eq. 1**. Our objective is to minimize the fully-discrete FOM equation residual **r**, defined in **Eq. 2**, on the physical domain  $\Omega$  with respect to the reduced state,  $\mathbf{q}_r$ 

$$\mathbf{q}_{r}^{n} \triangleq \arg\min_{\mathbf{q}_{r} \in \mathbb{R}^{n_{p}}} \left\| \mathbf{Pr}(\tilde{\mathbf{q}}_{p}) \right\|_{2}^{2},$$
with  $\mathbf{u}(\tilde{\mathbf{q}}_{p}^{n}) = \mathbf{u}_{BC}^{n}$  on  $\partial\Omega$ 
and  $\tilde{\mathbf{q}}_{p}^{0} = \mathbf{q}_{p,\text{ref}} + \mathbf{H}^{-1}\mathbf{V}_{p}(\mathbf{V}_{p})^{T}\mathbf{q}_{p}^{0}$ 
(7)

where the approximate solution variables,  $\tilde{\mathbf{q}}_p = \mathbf{q}_{p,ref} + \mathbf{H}^{-1}\mathbf{V}_p\mathbf{q}_r$ . The equation residual, **r**, is scaled by **P** using the  $L^2$ -norm, similar to the scaling matrix **H** in **Eq. 5** 

$$\mathbf{P} = diag(\mathbf{P}_1, \dots, \mathbf{P}_i, \dots, \mathbf{P}_{N_{elem}}), \tag{8}$$

where  $\mathbf{P}_i = diag(\varphi_{1,norm}^{-1}, \dots, \varphi_{N_{var},norm}^{-1})$ . Here,  $\varphi_{v,norm}$  represents the *v*th evaluated quantity of  $\mathbf{q}(\mathbf{q}_p)$ 

$$\varphi_{\nu,norm} = \frac{1}{t_1 - t_0} \int_{t_0}^{t_1} \frac{1}{\Omega} \int_{\Omega} \varphi_{\nu}^{\prime 2}(\mathbf{x}, \mathbf{t}) \, d\mathbf{x} \, d\mathbf{t}, \tag{9}$$

such that each equation in **r** has similar contributions to the minimization problem in **Eq. 7**. It is worth pointing out that the treatment of boundary conditions in the MP-LSVT ROM is fully consistent with the FOM in **Eq. 2**, which guarantees that the boundary conditions are satisfied in the ROM and serves as an important building block in FOM/ROM coupling in the component-based domain-decomposition framework in **Section 5**.

Following Eq. 7, a reduced non-linear system of dimension  $n_p$  can then be obtained and viewed as the result of a Petrov-Galerkin projection

$$\begin{pmatrix} \mathbf{W}_{p}^{n} \end{pmatrix}^{T} \mathbf{Pr}(\tilde{\mathbf{q}}_{p}^{n}) = \mathbf{0},$$
with  $\mathbf{u}(\tilde{\mathbf{q}}_{p}^{n}) = \mathbf{u}_{BC}^{n}$  on  $\partial \Omega$  (10)
and  $\tilde{\mathbf{q}}_{p}^{0} = \mathbf{q}_{p,\text{ref}} + \mathbf{H}^{-1} \mathbf{V}_{p} (\mathbf{V}_{p})^{T} \mathbf{q}_{p}^{0}$ 

where  $\mathbf{W}_p$  is the test basis

$$\mathbf{W}_{p}^{n} = \frac{\partial \mathbf{Pr}(\tilde{\mathbf{q}}_{p}^{n})}{\partial \mathbf{q}_{n}^{n}} = \mathbf{P}(\tilde{\mathbf{\Gamma}}^{n} - \Delta t \beta_{0} \tilde{\mathbf{J}}^{n} \tilde{\mathbf{\Gamma}}^{n}) \mathbf{H}^{-1} \mathbf{V}_{p}, \qquad (11)$$

with 
$$\tilde{\mathbf{J}}^n = [\partial \mathbf{f}/\partial \mathbf{q}]_{\mathbf{q}_p = \tilde{\mathbf{q}}_p}^n$$
 and  $\tilde{\boldsymbol{\Gamma}}^n = [\partial \mathbf{q}/\partial \mathbf{q}_p]_{\mathbf{q}_p = \tilde{\mathbf{q}}_p}^n$ .

# 4 REDUCED-ORDER MODELS ENHANCEMENT *VIA* BASIS ADAPTATION

While the MP-LSVT method improves the robustness and accuracy of the ROM, predictive capabilities (e.g., future-state prediction) are still restricted by the use of linear static basis, which has been shown to be inadequate for predictions in problems with slow Kolmogorov N-width decay [14,44]. Several remedies have been proposed to address this challenge

through, for example, localized linear bases [45,46], nonlinear bases [47,48], and online basis adaptation [49–51] etc. In the current work, we focus on online basis-adaption methods, which aim to update the trial basis  $\mathbf{V}_p$  during the online ROM calculation (**Eq. 10**) such that

$$\mathbf{V}_{p}^{n} \triangleq \underset{\mathbf{V}_{p}^{n} \in \mathbb{R}^{N \times n_{p}}}{\operatorname{small{eq:prod}}} \left\| \mathbf{Pr}(\tilde{\mathbf{q}}_{p}^{n}) \right\|_{2}^{2}, \tag{12}$$

where **r** is the fully-discrete FOM equation residual defined in **Eqs. 2**,  $\tilde{\mathbf{q}}_p^{\ n} = \mathbf{q}_{p,\text{ref}} + \mathbf{H}^{-1}\mathbf{V}_p^n \mathbf{q}_r^n$ , and  $\tilde{\mathbf{q}}_p^{\ n-j} = \mathbf{q}_{p,\text{ref}} + \mathbf{H}^{-1}\mathbf{V}_p^{n-j}\mathbf{q}_r^{n-j}$  while  $\mathbf{q}_r^n$  and  $\mathbf{q}_r^{\ n-j}$  are solutions to **Eq. 10**. This minimization problem can be solved exactly *via* the update

$$\mathbf{V}_{p}^{n} = \mathbf{V}_{p}^{n-1} + \delta \mathbf{V}_{p}, \tag{13}$$

where the basis at time-step n - 1 is adapted to n, through an increment,  $\delta \mathbf{V}_p \in \mathbb{R}^{N \times n_p}$ , given by

$$\delta \mathbf{V}_{p} = \frac{\left(\hat{\mathbf{q}}_{p}^{n} - \tilde{\mathbf{q}}_{p}^{n}\right)\left(\mathbf{q}_{r}^{n}\right)^{T}}{\|\mathbf{q}_{r}^{n}\|_{2}^{2}},$$
(14)

where  $\hat{\mathbf{q}}_p^n \in \mathbb{R}^N$  represents the full-state information, which can be evaluated based on the FOM equation residual as follows

$$\mathbf{q}(\hat{\mathbf{q}}_{p}^{n}) + \sum_{j=1}^{l} \alpha_{j} \mathbf{q}(\tilde{\mathbf{q}}_{p}^{n-j}) - \Delta t \beta_{0} \mathbf{f}(\tilde{\mathbf{q}}_{p}^{n}, t^{n}) - \Delta t \sum_{j=1}^{l} \beta_{j} \mathbf{f}(\tilde{\mathbf{q}}_{p}^{n-j}, t^{n-j}) = 0,$$
  
or 
$$\mathbf{q}(\hat{\mathbf{q}}_{p}^{n}) + \sum_{j=1}^{l} \alpha_{j} \mathbf{q}(\tilde{\mathbf{q}}_{p}^{n-j}) - \Delta t \beta_{0} \mathbf{f}(\hat{\mathbf{q}}_{p}^{n}, t^{n}) - \Delta t \sum_{j=1}^{l} \beta_{j} \mathbf{f}(\tilde{\mathbf{q}}_{p}^{n-j}, t^{n-j}) = 0.$$
  
(15)

here, we adopt an alternate formulation compared to [50] by updating the basis based on the full-state information evaluated at the current time step, n,  $\hat{\mathbf{q}}_p^n$ , instead of collecting at multiple time steps, which is similar to the work done by Zimmermann et al. [51]. We refer to this formulation as the one-step adaptive-basis approach. The rate of basis adaptation is empirically determined for the target applications and in the current work, we choose to adapt the basis at each time step.

To achieve gains in computational efficiency for the projection-based ROMs introduced in Section 3, hyperreduction is required to obtain an approximation of the non-linear function (e.g., f in Eq. 1) based on a small number of sampled elements-for example, it can be achieved by the discrete empirical interpolation method (DEIM) [52], or its least-squares regression analogue, gappy POD Everson and Sirovich [53]. In addition, the full-state information evaluation (Eq. 15) in basis adaptation can be computationally expensive and also requires hyper-reduction for efficiency gain so that the evaluation is only needed at a small number of sampled elements. This can be achieved by incorporating the recently developed adaptive sampling techniques [50], which update the selection of sampled elements based on the basis adaptation. However, the current work mainly focuses on the development and demonstration of the component-based ROM framework. Therefore for conciseness, we are not including ROM hyper-reduction in the current paper since in principle, its presence or absence will not impact the validity of the



framework. We will incorporate the hyper-reduced ROMs in the framework in future work.

# 5 COMPONENT-BASED DOMAIN-DECOMPOSITION FRAMEWORK

In this section, we introduce the component-based domaindecomposition framework for modeling large-scale engineering systems. Because our research has been primarily motivated by applications to propulsion systems for aerospace applications, we use a multi-injector model rocket combustor to assist in the description of the framework formulation. Figure 1 presents a representative geometry composed of seven injector elements through each of which fuel and oxidizer in separate channel feed a downstream combustion chamber. The physical domain has been separated into eight components with seven for the injector elements ( $\Omega_k$ , etc.) and one for the downstream combustor and nozzle ( $\Omega_8$ ). A set of similar configurations are also included in our numerical examples in Section 6.2. Even though the configuration in Figure 1 represents a complicated engineering system, most of the components share identical geometric features, an attribute that is common in many engineering systems (e.g., compressors, gas turbine engines, and wind farms, etc.). The interior components  $\Omega_k$ , where  $k = 2, \ldots, 6$ , are identical to each other geometrically, the two outer components  $\Omega_1$  and  $\Omega_7$  mirror each other in geometrical configuration (i.e., symmetric about the center axis), while  $\Omega_8$  does not resemble any other components. Therefore, the representation of the multi-injector rocket combustor in Figure 1 can be simplified as the combination of three representative components: I  $(\Omega_I)$ —an interior injector element (e.g.,  $\Omega_3$ ); II ( $\overline{\Omega}_{II}$ )—an outer injector element (e.g.,  $\Omega_7$ ); and III—the downstream combustor and nozzle  $(\Omega_8)$ , which enables the applications of the component-based domaindecomposition framework without accessing the expensive high-fidelity FOM of the full system. Furthermore, we denote the representative components that can be repetitively used in the full system (e.g.,  $\overline{\Omega}_{I}$  and  $\overline{\Omega}_{II}$ ) as reference components. Based on a priori knowledge of the dynamics in the system [54,55], different modeling strategies can be adopted for different components (or subdomains)  $\Omega_k$ , each of which is bounded by the physical boundaries,  $\partial \Omega_k$  (e.g., inlets, outlets, walls surrounding  $\Omega_k$ , represented by the solid lines in Figure 1), and interface boundaries (dashed lines in Figure 1) shared between components (e.g., k and m),  $\partial \Omega_{km} \triangleq \Omega_k \cup \Omega_m$ .

First, we introduce a general description of the domain-decomposition formulation

$$\mathbf{B}_{k}\mathbf{r}(\bar{\mathbf{q}}_{p,k}^{n}) = \mathbf{0},$$
with  $\mathbf{u}_{k}(\bar{\mathbf{q}}_{p,k}^{n}) = \mathbf{u}_{BC,k}^{n}$  on  $\partial\Omega_{k}$ , (16)
and  $\mathbf{v}_{km}(\bar{\mathbf{q}}_{p,k}^{n}) = \mathbf{v}_{km}(\bar{\mathbf{q}}_{p,m}^{n})$  on  $\partial\Omega_{km}$ ,

where k denotes the numbering of the sub-components in the formulation,  $\mathbf{B}_k \in \mathbb{R}^{n_{B,k} \times N_k}$  denotes a matrix that enables the component to adopt either FOM or ROM for the corresponding kth subdomain  $(\Omega_k)$ , and  $N_k$  is the total number of degrees of freedom in  $\Omega_k$  and  $N_k = N_{elem,k} \times N_{var}$ , where  $N_{elem,k}$  is the total number of elements in  $\Omega_k$ . For FOM,  $n_{B,k} = N_k$ , and  $\mathbf{B}_k = \mathbf{I}$ , similar to **Eq. 2**, and for ROM,  $n_{B,k} = n_p$ , and  $\mathbf{B}_k = (\mathbf{W}_{p,k}^{n})^T \mathbf{P}_k$ , consistent with Eq. 10.  $\mathbf{u}_k : \mathbb{R}^{N_k} \to \mathbb{R}^{N_{b,k}}$ represents the physical boundary condition state to be satisfied as  $\mathbf{u}_{BC,k}$  on  $\partial \Omega_k$ , where  $N_{b,k}$  is the total number of degrees of freedom associated with the boundary  $\partial \Omega_k$ , and  $N_{b,k} = N_{elem,BC,k} \times$  $N_{var}$ , with  $N_{elem,BC,k}$  as the number of elements adjacent to the boundary  $\partial \Omega_k$ . In addition, a (non-linear) function,  $\mathbf{v}_{km} \colon \mathbb{R}^{N_k} \to \mathbb{R}^{N_{Interf,k}}$ , is introduced to match the interface condition between subdomains  $\Omega_k$  and  $\Omega_m$ , where  $N_{Interf,k}$  is the total number of degrees of freedom associated with the boundary  $\partial \Omega_k$ , and  $N_{Interf,k} = N_{elem,Interf,k} \times N_{var}$ , with  $N_{elem,Interf,k}$  as the number of elements adjacent to the interface boundary  $\partial \Omega_{km}$ . Additional details are provided in Section 5.2.

Given the complexity and scale of the physics, small-scale local components with identical features (e.g., the interior and outer injector elements,  $\Omega_3$  and  $\Omega_7$ , in **Figure 1**, the nozzle element in a gas turbine, the rotor blade in a compressor, or the wind turbine in a wind farm) often require high-fidelity modeling to achieve satisfying accuracy in many-query engineering applications. Therefore, ROMs can be an ideal candidate from the viewpoint of satisfying efficiency and accuracy requirements. On the other hand, large-scale system-level components (e.g., the downstream combustor and nozzle,  $\Omega_8$ , in **Figure 1**) are



usually governed by physics that is less demanding in numerical resolution. This makes the reduced-fidelity full-order model (RF-FOM) a good candidate for modeling of the large-scale systemlevel components—for example, coarse-mesh LES, nonlinear Euler model, unsteady Reynolds Averaged Navier Stokes models.

# 5.1 Component-Based Reduced-Order Model Training

To enable modeling of the full system (e.g., Figure 1), the FOM of which is not directly accessible, we develop a component-based ROM training method in the current section, that requires highfidelity FOM simulations for only the reference components identified in the system. The method aims to generate a rich training dataset that contains representative dynamics of the components when integrated in the full system of various configurations (e.g., different numbers of injector elements), thus enabling the generation of predictive component-based ROMs, which is analogous to the localized ROM strategy developed for finite element method with representative work by Henning and Peterseim [56]; Eftang and Patera [28]; and Smetana and Patera [29]. To achieve this, we introduce unsteady perturbations at interface boundaries in the FOM simulations of the reference components, following Eq. 1, to fabricate the effects of system-level responses, as demonstrated in Figure 2. Most importantly, by enriching the functions used for the boundary perturbations, the dynamics of different full-system configurations can be embedded within the ROMs of the individual components. For example, the effects of systemlevel acoustics can be accounted for by imposing different pressure perturbations at the boundary

$$p(t) = p_{ref} \left[ 1 + \sum_{i=1}^{n_f} A_i \sin(2\pi f_i t) \right].$$
(17)

where  $f_i$  is the frequency included in the boundary perturbations with  $A_i$  denoting the associated amplitude, reflecting the anticipated full-system responses, and  $n_f$  is the total number of frequencies, an indicator of the richness of the excited dynamics. Alternatively, velocity oscillations can be enforced at the boundaries following similar function in Eq. 17 to mimic the effects of large-scale flow dynamics in the full system.

It is important to ensure that the imposed boundary conditions (dashed lines in Figure 2) do not imprint the dimensions of the individual component upon the combustion dynamics. All internal domains are subject to acoustic resonance at scales determined by their geometry, but the dimensions in the individual component are not representative of those of the full system and so must not appear in the training dataset. Accordingly, it is critical that the boundary conditions on the reference component be chosen such that its geometry does not impact the FOM solutions of the dynamics upon which the ROM is based. An effective way to accomplish this is by applying nonreflective boundary conditions through pertinent Riemann invariants. We adopt the formulation that proved to be effective for multi-dimensional reacting flow simulations [57]. It should be mentioned that the ROM training method in Figure 2 requires a level of prior knowledge of the essential physics in the full-system (e.g., acoustics) to ensure that pertinent physics are included in the ROM training. For example, in rocket combustor design such as Figure 1, system-level acoustic frequencies can be estimated a priori based on the full-system configuration. Correspondingly, a multi-frequency perturbation can be imposed at the boundary conditions for ROM training, the frequency band of which covers the target full-system acoustic frequencies and therefore excites the essential dynamics anticipated within the component when integrated in the fullsystem. A similar idea has been demonstrated in simple 1D problems by the current authors Huang et al. [40]; Xu et al. [41]. To account for complex dynamics at the component interfaces, instead of directly imposing the perturbations at the interfaces, auxiliary domains can be introduced for the ROM training, which is discussed and demonstrated in Section 6.

# 5.2 Integration in Full System Simulations

Once the component-based models are constructed following the strategies in Section 5.1, the effective integration of these models in the domain-decomposition framework is another determining factor for the success of the framework. In this section, we use the example in Figure 1 to illustrate the integration of the component-based models. Following the method in Section 5.1, ROMs are trained on the reference components, the trained ROMs can be used repetitively to model identical components in the full system to enable geometric variations (Figure 3A with  $(\cdot)$  indicating that the left element mirrors the right one) based on the premise that the injector elements in the full system share identical geometries as the reference components used for ROM training. As discussed above, the RF-FOM can be adopted to efficiently model the geometrically flexible components (e.g., the downstream combustor and nozzle) that vary with the full system configurations, given the less demanding requirements on modeling accuracy. Alternatively, ROMs could also be developed for the geometrically flexible components although ROM developments for geometric variations remains an open area of research.



To achieve accurate predictions of the system-level response governed by the component interactions, it is important to ensure that the essential information is transferred between components. We adopt a direct flux matching method *via* ghost cell assignment to couple the components at the interface with no overlap

$$\mathbf{v}_{km}\left(\bar{\mathbf{q}}_{p,k}^{n},\bar{\mathbf{q}}_{p,m}^{n}\right) = \mathbf{v}_{km}\left(\bar{\mathbf{q}}_{p,m}^{n},\bar{\mathbf{q}}_{p,k}^{n}\right) \quad \text{on} \quad \partial\Omega_{km}, \tag{18}$$

a schematic of which is shown in Figure 3B, describing the coupling between domain  $\Omega_k$  and  $\Omega_m$  through the interface  $\partial \Omega_{km}$ . The two adjacent mesh cells of  $\partial \Omega_{km}$  are indicated by the shaded areas. When performing calculations based on Eq. 16 at time step *n* at a cell with solution variables  $\bar{\mathbf{q}}_{p,k}$  near  $\partial \Omega_{km}$  in  $\Omega_k$ , solution variables at the adjacent (or ghost) cell, denoted as  $\bar{\mathbf{q}}_{p,m}$ , are assigned by the corresponding neighboring domain  $\Omega_m$ . The combination of  $\bar{\mathbf{q}}_{p,k}$  and  $\bar{\mathbf{q}}_{p,m}$  is then used to calculate the interface condition state, denoted as  $\mathbf{v}_{km}(\bar{\mathbf{q}}_{p,k}^{n}, \bar{\mathbf{q}}_{p,m}^{n})$ , and vice versa, thus guaranteeing the interface condition state is matched as posed in **Eq. 16**—i.e.,  $\mathbf{v}_{km}(\bar{\mathbf{q}}_{p,k}^{n}) = \mathbf{v}_{km}(\bar{\mathbf{q}}_{p,m}^{n})$ . Specifically, we set the interface condition state function  $\mathbf{v}_{km}$  to be the numerical fluxes (both inviscid and viscous) to better suit the finite volume scheme of the numerical solver used for the current work [58]. We remark that the interface method remains the same regardless of whether the domain is represented by FOM or ROM.

The major benefit of the direct-flux-matching interfacing method is that it inherently accounts for changes in flow

characteristics at the interface and therefore important phenomena such as reverse flows are naturally supported. More importantly it makes the training of the componentbased ROMs relatively independent of their coupling with other components in the framework, which allows more flexibility in the ROM training strategy. For example, auxiliary domains (e.g., adjacent injector elements) can be introduced in the component-based ROM training stage in **Figure 2** to better emulate interactions between injector elements in the training dataset. These aspects are demonstrated in the numerical results in **Section 6**.

# **6 NUMERICAL RESULTS**

To assess the capabilities of component-based domaindecomposition modeling framework in predicting multi-scale multi-physics problems (e.g., reacting flows), two model rocket combustors are considered. The first configuration is a twodimensional representation of a generic laboratory-scale single injector configuration [59]. This case is used to a) assess the component-based ROM training strategy and the interfacing method between components in the framework, and b) to explore the feasibility of using the framework to predict dynamics on different full-system geometries. The second configuration is a two-dimensional representation of a



multi-injector model rocket combustor [60], and is used to evaluate the capabilities of the framework to model different geometric configurations of the full system as demonstrated in **Figure 3**.

The computational infrastructure used for the full- and reduced-order models solves conservation equations for mass, momentum, energy and species transport [58,61] in a fully coupled manner, which has been used to model a variety of



complex, practical reacting flow problems. More details of the FOM equations can be found in Appendix B of [14]. The FOM employs a cell-centered second-order accurate finite volume method for spatial discretization and uses the direct flux matching method as described in **Section 5.2** for parallel computation. The Roe scheme [62] is used to evaluate the inviscid fluxes and a Green-Gauss gradient reconstruction procedure [63] is used to compute the face gradients and viscous fluxes. A gradient limiter by Barth and Jespersen [64] is used to preserve monotonicity for flow fields with strong gradients. A ghost cell formulation is used for treatment of boundary conditions. Time integration for all FOM simulations uses the implicit second-order accurate backwards differentiation formula with dual time-stepping.

# 6.1 Single-Injector Model Rocket Combustor

First, we explore and demonstrate the component-based domaindecomposition framework on a 2D-planar representation of a generic laboratory-scale rocket combustor designed to study combustion dynamics [59]. The configuration is shown in **Figure 4A** and consists of a shear coaxial injector with an outer passage,  $T_1$ , that introduces fuel near the downstream end of the coaxial inner passage,  $T_2$ , which in turn feeds oxidizer to the combustion chamber with a choked nozzle downstream, resulting in combustion-driven acoustics to be sustained. Operating conditions in this single-injector combustor are maintained with an adiabatic flame temperature of approximately 2,700 K and a mean chamber pressure of 0.95 MPa. The  $T_1$  stream contains gaseous methane (100% CH<sub>4</sub>) at 300 K. The  $T_2$  stream is 42% gaseous O<sub>2</sub> by mass and 58% gaseous H<sub>2</sub>O by mass at 660 K. The  $T_1$  and  $T_2$  streams are maintained at constant mass flow rates, 0.46 kg/s and 5.40 kg/s, respectively. Combustion is represented by the flamelet progress variable (FPV) model [65] with GRI-1.2 [66] chemical kinetics, which consists of 32 species and 177 chemical reactions. The chemical species are treated as thermally perfect gases. Note that although 32 chemical species are modeled, the FPV model only solves transport equations for three scalar quantities: the mean mixture fraction ( $Z_{mean}$ ), the mixture fraction variance ( $Z''^2$ ), and the reaction progress variable ( $C_{mean}$ ) [65]. Individual chemical species mass fractions are looked up from pre-computed flamelet manifolds [67].

As shown in Figure 4A, the single-injector configuration consists of two components, the upstream injector element  $(\Omega_A)$  and the downstream combustor and nozzle  $(\Omega_B)$ . Three different lengths of the combustor  $(L_c)$  are investigated for this configuration by maintaining the upstream component  $(\Omega_A)$ while varying the length of the downstream component ( $\Omega_{\rm B}$ ), i.e., L<sub>B</sub>. This change in length leads to different dynamic behaviors as shown in **Figures 4B–D** when both  $\Omega_A$  and  $\Omega_B$  adopt FOM as their modeling strategy, denoted as FOM-FOM, the solutions from which are taken as the *truth* to evaluate the componentbased ROM framework. It can be readily seen from Figure 4B that by just varying the combustor length  $(L_c)$ , different pressure oscillations can be sustained. The longer combustor lengths tend to drive higher pressure oscillation amplitudes (> 30% peak-topeak) while the shorter length maintains lower amplitude, < 15%, (Figure 4B left). This features different frequencies as shown in the power spectral densities (PSD) in Figure 4B right. In addition, the combustion dynamics changes with  $L_c$  as seen in Figures 4C,D, which allows reverse flow at the component interface  $\partial \Omega_{AB}$  under high-amplitude pressure oscillations. Thus, this single-injector combustor is an appropriate testbed for the component-based domain-decomposition framework proposed in Section 5, which adopts a ROM for the upstream component ( $\Omega_A$ ), and a RF-FOM for the downstream component ( $\Omega_{\rm B}$ ).

### 6.1.1 Injector Element Reduced-Order Models Training

As mentioned in Section 5.1, the ROM training strategy is crucial to the success of the component-based framework. For the investigations on the single-injector configuration (Figure 4), two types of strategies are considered to train ROM for the upstream component ( $\Omega_A$ ): 1) system-based (only for framework verification), and 2) component-based methods as shown in Figure 5.

- The system-based approach (Figure 5A) simulates the complete geometry of interest (e.g.,  $L_c = 0.42$  m in Figure 4), trains the ROM based on the extracted snapshot solutions corresponding to the upstream injector-element component ( $\bar{\Omega}_A$ ), and only serves to verify the feasibility of the component-based ROM framework.
- The component-based approach (Figure 5B), as the primary focus of the current work, simulates only the



square (right) fields with different models (component-based ROM vs. system-based ROM vs. FOM) used for the upstream injector element ( $\Omega_A$ ) for the single-injector combustor.

injector-element for ROM training and produces essential dynamics by forcing the downstream boundary conditions. Further, component-based training is necessary for systems whose full-scale characteristics are beyond the current available computing capabilities. Instead of imposing the forcing directly at the downstream end of the injector element (i.e., the dashed line in **Figure 5B**), an auxiliary domain with exponentially stretched mesh in the axial direction is added downstream for the component-based ROM training approach. The addition of the auxiliary domain is necessary to represent large-scale motions

(e.g., vortex shedding) that are undamped in the downstream domain whereas small scale motions (e.g., chemical reaction) that would interfere with long-wavelength forcing are damped before the downstream boundary is reached. It inherently incorporates complex dynamics (e.g., reverse flow) at the component interface  $\partial \Omega_{AB}$  (e.g., similar to what has been observed in **Figures 4B,C** in the training snapshots for ROM, which cannot be accounted if boundary conditions are directly applied. In addition, it reduces the influence of the training geometry (i.e., injector element + auxiliary domain) on the dynamics

in the training dataset for ROM, especially on the longitudinal acoustics.

In either of the above approaches, the trained ROM is coupled with different downstream components to model the full system. The component-based approach provides flexibility in generating dynamics and substantial savings in computational cost for ROM training since it eliminates the need for computation of a complete configuration in **Figure 4**.

We consider  $L_c = 0.42$  m in **Figure 4** as the target full system, which exhibits a self-excited high-amplitude pressure oscillations at 1150 Hz as shown in **Figure 4A**. Both system-based and component-based methods are used to generate training snapshots for ROM development. To generate the essential dynamics during component-based training (**Figure 5B**), the Riemann invariant corresponding to backward characteristics ( $q_{u-c}$ ) is perturbed at the downstream boundary using the following forcing function:

$$q_{u-c} = q_{u-c,ref} [1 + A\sin(2\pi ft)],$$
(19)

where A = 0.1, f = 1150Hz to generate similar pressure oscillations observed in the full system, and  $q_{u-c,ref}$  represents the reference value of the Riemann variable of backward characteristics that maintains the nominal pressure. The training snapshots containing two acoustic cycles of information (i.e.,  $T_p = 2/f$ ) are used to generate the POD trial basis as described in **Section 3.1**, the characteristics of which are investigated to understand how well the POD trial basis represents the training dataset. The representation is evaluated using the POD residual energy:

POD Residual Energy
$$(n_p)$$
,  $\% = \left(1 - \frac{\sum_{i=1}^{n_p} \tilde{\sigma}_i^2}{\sum_{i=1}^{n_p total} \tilde{\sigma}_i^2}\right) \times 100$ , (20)

where  $\tilde{\sigma}_i$  is the *i*th singular value of the SVD used to compute the trial basis  $V_p$ . Again,  $n_p$  is the number of vectors retained in the POD trial basis, and  $n_{p,total}$  (= 1740) is the total number of snapshots in the dataset. The residual energy as a function of  $n_p$ , as shown in Figure 5C for both system-based and componentbased training methods, reveals the information excluded by the POD representation for a given number of modes. Overall, the two ROM training methods show very similar POD residual energy decay. The results show that, to recover approximately 99% of the total energy, 70 and 68 modes are needed for system-based and component-based methods, respectively, while approximately 150 modes are required to reach 99.9% for both methods. This slow energy decay is indicative of the significant complexity of the system dynamics. Many fundamental projection-based ROM methodologies are tested on relatively simple problems requiring only ~ 10 trial basis modes to achieve 99.9% POD energy [68-70]. ROMs for more practical engineering systems, however, generally require ~ 100 trial basis modes [43,71,72].

#### 6.1.2 Performance

Next, we couple the trained injector-element ROMs from Section 6.1.1 with the downstream combustor and nozzle ( $\Omega_B$  in Figure 4) *via* the interfacing method described in Section 5.2 to model the full configuration of  $L_c = 0.42$  m. Two acoustic cycles (1.74 ms) of snapshots (1,740 in total) are used to train the ROMs,

which are constructed with the number of POD modes capturing 99% of the total energy. To consistently evaluate the modeling capabilities of the resulting framework based on the FOM-FOM results (i.e., the true solutions) in **Figure 4**, we adopt a FOM, instead of RF-FOM, for the downstream component ( $\Omega_B$ ). The coupled ROM-FOM framework is then used to predict 20 ms of dynamics and compared against the FOM-FOM results.

First, we evaluate the performance based on the local pressure signals measured at x = 0 in Figure 4, which has been often used as an important quantity of interest (QoI) to assess the accuracy of modeling tools in predicting combustion instability [58]. The predicted pressure signals, both time traces (left and middle) and PSDs (right), are compared in Figure 6A with different models used for the used for the upstream injector element ( $\Omega_A$ ) for the single-injector combustor. Furthermore, the peak-to-peak pressure oscillation amplitude  $(p'_{ptp})$  based on the root-meansquare (RMS) in Eq. 21) and the dominant acoustic frequency  $(f_a)$ are calculated based on the pressure time trace and PSD, respectively and compared against the FOM-FOM results for quantitative assessment, as summarized in Table 1. Overall, the ROM-FOM framework (either with system-based or componentbased ROM) is able to predict the pressure amplitude and frequency with reasonable accuracy ( < 10%). As shown in Figure 4, within the training period, the system-based ROM-FOM replicates the FOM-FOM results closely, as expected. On the other hand, the component-based ROM-FOM also represents the pressure oscillations reasonably well even though the essential downstream component is excluded in ROM training. Moreover, the comparisons in Figure 6A confirm that the component-based ROM training strategy is feasible by emulating feedback responses from the downstream component with boundary forcing to train ROM, as illustrated in Figure 5. More importantly, both approaches enable long-time predictions (e.g., 1.74 ms training vs. 20 ms prediction), which is not commonly reported in the literature for ROM applications relevant to compressible fluid flow problems.

Second, we assess the predictive capabilities of the ROM-FOM framework based on two other QoIs, time-averaged and rootmean-square (RMS) fields of the state variables, which serve as crucial determining factors in many engineering applications

$$\mathbf{\Phi}_{\text{average}} = \frac{1}{n_t} \sum_{n=1}^{n_t} \mathbf{\Phi}^n, \quad \mathbf{\Phi}_{\text{RMS}} = \sqrt{\frac{1}{n_t} \sum_{n=1}^{n_t} \left(\mathbf{\Phi}^n - \mathbf{\Phi}_{\text{average}}\right)^2}, \quad (21)$$

where  $n_t$  is the total number of snapshots included to calculate the QoI, and  $\Phi^n$  represents the state variable of interest, e.g., pressure (*P*), temperature (*T*), and axial (or streamwise) velocity (*U*), at time step *n*. In addition, the errors of the ROM-FOM framework in predicting  $\Phi_{\text{average}}$  and  $\Phi_{\text{RMS}}$  are further quantified as follows

$$\epsilon_{\Phi} = \frac{\|\Phi - \Phi_{\text{ref}}\|_2}{\|\Phi_{\text{ref}}\|_2},\tag{22}$$

where  $\Phi$  represents the QoIs (i.e., either the time-averaged or RMS field) for the error measurement, and  $\Phi_{ref}$  represents the QoIs calculated from the FOM-FOM framework. The errors, calculated based on Eq. 22, are summarized in Table 2. Though the ROM-FOM framework is able to provide reasonably accurate

TABLE 1   Comparisons of the dominant acoustic frequency (f <sub>a</sub> ) and the peak-to-peak pressure amplitudes (p <sub>dp</sub> ) with different models (component-based ROM vs.
system-based ROM vs. FOM) used for the upstream injector element for the single-injector combustor, corresponding to the results in Figure 6A.

Model for $\Omega_A$	f <sub>a</sub> , Hz	Error in <i>f<sub>a</sub></i> , %	p <sub>ptp</sub> ′, kPa	Error in p <sub>ptp</sub> ′, %
FOM	1,150	_	125.08	_
System-based ROM	1,050	8.70	116.25	7.06
Component-based ROM	1,100	4.35	112.93	9.71

**TABLE 2** Comparisons of the errors in predicting time-averaged and RMS fields of pressure (*P*), temperature(*T*), and axial velocity (*U*) corresponding to **Figures 6B-D** with different models (component-based ROM vs. system-based ROM) used for the upstream injector element for the single-injector combustor.

Qol	Model for $\Omega_A$	€р	ε <sub>T</sub>	€IJ
Time-averaged Fields	System-based ROM	$1.28 \times 10^{-2}$	$5.93 \times 10^{-2}$	6.06 × 10 <sup>-2</sup>
	Component-based ROM	$2.63 \times 10^{-2}$	$5.58 \times 10^{-2}$	$7.72 \times 10^{-2}$
RMS Fields	System-based ROM	$1.6 \times 10^{-1}$	$4.03 \times 10^{-1}$	2.32 × 10 <sup>-1</sup>
	Component-based ROM	$9.80 \times 10^{-2}$	$4.63 \times 10^{-1}$	$2.50 \times 10^{-1}$

predictions of the time-averaged fields (< 3% errors for P, and < 8% for T and U) with both system-based, and componentbased ROM used for  $\Omega_A$ , significant errors (>9% for P, >20%) for *U*, and > 40% for *T*) are observed in predicting the RMS fields, which are directly related to the unsteady dynamics. Specifically, while the predictions of pressure RMS fields are acceptably accurate, the ROM-FOM framework exhibits difficulties in representing the RMS fields of state variables featuring strong advection (i.e., T and U), which can be largely attributed to the chaotic nature of the dynamics as seen in Figures 4B,C. The pressure field exhibits organized dynamics due to strong selfexcited oscillations in the full system, which allows the ROM-FOM to provide reasonable predictions as the trial basis generated during the training stage is able to efficiently represent such organized dynamics. However, in turbulent reacting flows (characterized by transport of strong temperature gradients), chaotic and non-stationary features present a major challenge. The basis is unable to represent the unsteady features of T and U in the upstream component  $\Omega_A$ , therefore producing significant errors when coupled with the FOM for the downstream component  $\Omega_B$  in the resulting framework. This is not a flaw in the ROM formulation or the domain-decomposition framework, but rather a limitation of using a linear and static basis set to construct the ROM for the upstream component in the framework, which has also been discussed by the current authors in [14].

Such challenges and the limitation of using a linear and static basis set are further revealed by comparing the time-averaged and RMS fields between the FOM-FOM and ROM-FOM framework in **Figures 6B–D**, which shows significanly under-predicted magnitudes of the RMS fields by the ROM-FOM framework. More importantly, distinguishable mismatches between solutions in  $\Omega_A$  and  $\Omega_B$  are observed at the component interface  $\partial \Omega_{AB}$  in ROM-FOM results, featuring abrupt changes in numerical values in the regions adjacent to the interface (e.g.,  $P_{average}$ ,  $U_{average}$ ,  $U_{RMS}$  of system-based ROM-FOM and  $P_{RMS}$  of component-based ROM-FOM), which is absent in FOM-FOM results. The mismatches can be mainly attributed to the inconsistent orders of modeling accuracy between ROM in  $\Omega_A$ , restricted by the POD basis, and FOM in  $\Omega_{\rm B}$ , restricted by the mesh resolution. The use of a static linear basis for ROM in  $\Omega_A$ limits its predictive capabilities. It is pointed out that the interface mismatches are not unique to ROM/FOM coupling, but more general issues for finite-element [73] and finite-volume methods, especially with non-matching [74] grids (i.e., inconsistent orders of modeling accuracy)-for example, the coupling of low- and high-order CFD solvers (FOMs) may require-for example-an overset-mesh approach [21,75]. Though such methods can be effective in coupling of low- and high-order FOMs, it is not clear whether such methods can be applied directly to ROM and FOM coupling since ROM evolves on a reduced dimensional trajectory determined by the basis, while the FOM (either with low- or high-order numerical methods) solves the dynamical system on the full state space trajectory.

# 6.1.3 Performance Enhancement *via* Adaptive-Basis Reduced-Order Models

To address the above challenges, we seek to improve the ROM modeling accuracy via the one-step adaptive-basis approach introduced in Section 4. During the offline stage, 10 snapshots from the component-based ROM training demonstrated in Figure 5B are used to generate the initial POD basis  $V_{n}^{0}$ 5 POD modes, containing >99.9% of the total energy, are chosen to develop the ROM for the upstream injector element  $(\Omega_A)$ . Then the POD basis is adapted at each time step based on the algorithm in Eq. 13, which is then used to construct an updated ROM. It is noted that even though significant reduction in the offline training cost is enabled by the adaptive-basis approach, the additional costs required to evaluate the fullstate information in Eq. 15, can lead to an increase of the online-stage computational cost. On-line cost savings can be accomplished using hyper-reduction [52,53,76,77], which is not considered for the current work and will be included for future investigations. In the current work, we denote the ROM enhanced with basis adaptation as adaptive-basis ROM while in



TABLE 3   Comparisons of the dominant acoustic frequency (f) and the peak-to-peak pressure amplitudes ( $p_{olo}$ ) with different models (Adaptive-basis ROM vs. FOM) used
for the upstream injector element for the single-injector combustor with different combustor lengths (L <sub>c</sub> ), corresponding to the results in Figures 7A-C.

<i>L<sub>c</sub></i> , m	Model for	f, Hz	Error in	p <sub>ptp</sub> ′, kPa	Error in
	$\Omega_{A}$		f, %		p <sub>ptp</sub> ′, %
0.42	FOM	<i>f<sub>a</sub></i> : 1,150	_	125.08	_
	Adaptive-basis ROM	<i>f<sub>a</sub></i> : 1,200	4.35	123	1.67
0.35	FOM	f <sub>a</sub> : 1,350	_	128.28	_
	Adaptive-basis ROM	<i>f<sub>a</sub></i> : 1,300	3.70	114.95	10.39
0.35	FOM	f <sub>b</sub> : 500, f <sub>a</sub> : 1,500	_	67.87	_
	Adaptive-basis ROM	f <sub>b</sub> : 450, f <sub>a</sub> : 1,400	f <sub>b</sub> : 4.35, f <sub>a</sub> : 6.70	59.49	12.34

**TABLE 4** Comparisons of the errors in predicting time-averaged and RMS fields of pressure (*P*), temperature(*T*), and axial velocity (*U*) using the framework corresponding to **Figures 7D-H** with adaptive-basis ROMs for the single-injector combustor with different combustor lengths ( $L_c$ ).

Qol	<i>L<sub>c</sub></i> , m	€р	εŢ	ε <sub>U</sub>
Time-averaged Fields	0.42	4.23 × 10 <sup>-3</sup>	1.95 × 10 <sup>-2</sup>	2.10 × 10 <sup>-2</sup>
	0.35	6.70 × 10 <sup>-3</sup>	1.94 × 10 <sup>-2</sup>	2.17 × 10 <sup>-2</sup>
	0.28	$2.20 \times 10^{-3}$	$2.50 \times 10^{-2}$	1.68 × 10 <sup>-2</sup>
RMS Fields	0.42	7.87 × 10 <sup>-2</sup>	9.70 × 10 <sup>-2</sup>	6.44 × 10 <sup>-2</sup>
	0.35	6.50 × 10 <sup>-2</sup>	1.084.38 × 10 <sup>-2</sup>	8.49 × 10 <sup>-2</sup>
	0.28	$1.08 \times 10^{-1}$	$1.07 \times 10^{-1}$	$8.30 \times 10^{-2}$

contrast, we denote the ROM of **Section 6.1.1** as the static-basis ROM. The adaptive-basis ROM is coupled with different downstream components ( $\Omega_B$ ) to model the full configuration with three different combustion lengths ( $L_c = 0.42$ , 0.35, and 0.28 m as shown in **Figure 4**), denoted as adaptive-basis framework.

Following similar evaluation procedures as in Section 6.1.2, local pressure signals measured at x = 0 are first compared with the FOM-FOM results in Figures 7A-C for different combustor lengths. The peak-to-peak pressure oscillation amplitude  $(p'_{ptp})$ and the dominant acoustic frequency ( $f_a$  and  $f_b$ ) are calculated based on the pressure time trace and PSD, respectively and compared against the FOM-FOM results for quantitative assessment, as summarized in Table 3. It can be readily seen that, compared to the static-basis results in Figure 6, the predictions of the pressure signals for  $L_c = 0.42 \text{ m}$  are improved with adaptive-basis, especially in predicting  $p'_{ptp}$ (< 2% with adaptive-basis ROM versus > 7% with static-basis ROM). In addition, incorporating basis adaptation in the ROM enables more accurate predictions of the high-frequency pressure responses comparing the pressure PSD in Figure 7A (right) to Figure 6A (right). More importantly, the framework can also be extended to model other full-system configurations by coupling the upstream adaptive-basis ROM with different downstream components as illustrated in Figure 4. As exhibited in Figures 7B,C, the adaptive-basis framework is able to predict the pressure characteristics changes due to variations in full-system geometric configurations reasonably well, which shows < 7% errors in f <12% errors in  $p'_{ptp}$ , as summarized in **Table 3**. Though the characteristics of the pressure field appear to be similar between the single-injector combustors with  $L_c = 0.42$  and 0.35 m, the pressure signals with  $L_c = 0.28$  m exhibit significantly different behaviors, featuring approximately 50% reduction in  $p'_{ptp}$  and an additional low-frequency acoustic mode ( $f_b$  in **Figure 7C**) appearing in the PSD analysis, in addition to the dominant acoustic mode ( $f_a$ ). Such distinguishable changes in QoIs (e.g., pressure characteristics) are well-captured using the adaptive-basis framework, which can provide important guidelines for engineering system design (e.g. to design a rocket combustor with reduced pressure oscillations).

Next, we extend the evaluations of adaptive-basis framework to the predictions of time-averaged and RMS fields defined in Eq. 21, the errors of which are calculated using Eq. 22 and summarized in **Table 4**. Significant improvement (e.g., approximately O(10) error reduction) in predicting the timeaveraged and RMS fields of P, U, and T can be readily seen comparing **Table 4** (rows corresponding to  $L_c = 0.42$  m) and **Table 2**. The time-averaged and RMS fields predicted using adaptive-basis framework are investigated further in **Figures 7D–F**, which shows excellent agreement with the FOM-FOM results with the magnitudes of the RMS fields predicted correctly and no distinguishable interface mismatches observed in **Figures 6B–D**.

Moreover, the adaptive-basis framework is demonstrated to be capable of predicting the time-averaged and RMS fields for different combustor lengths reasonably accurately as reflected in **Table 4**. Specifically, the time-averaged and RMS fields of temperature (*T*) are selected to further demonstrate the modeling capabilities of the adaptive-basis framework as shown in **Figures 7G,H** because the temperature dynamics is characterized by chaotic non-stationary and advection-dominated features as shown in **Figure 4**, which prove to be most challenging to represent with static-basis framework as shown in **Figure 6**. Overall, the adaptive-basis framework is able to represent the changes in the time-averaged and RMS temperature fields with variations in the combustor lengths.

# 6.2 Multi-Injector Model Rocket Combustor

Next, we proceed to demonstrate the component-based domaindecomposition framework on the multi-injector model rocket combustor configuration shown in **Figure 8A**, based on a laboratory rocket model engine [60], originally designed to study combustion instability of transverse acoustics. In **Figure 8A**, we take the five-injector configuration as an example for illustration and also consider the configurations with three and seven injectors to demonstrate the capabilities of the framework in the following sections. As seen in Figure 8A, this configuration consists of five shear coaxial injectors, each of which is similar to the single-injector geometry in Figure 4, and is featured with an outer passage,  $T_1$ , that introduces fuel near the downstream end of the coaxial inner passage,  $T_2$ , that feeds oxidizer to the combustion chamber. The operating conditions in all the multi-injector combustor configurations are maintained with an adiabatic flame temperature of approximately 2,700 K and a mean chamber pressure of 1.3 MPa. The  $T_1$  stream contains gaseous methane (100%  $CH_4$ ) at 300 K. The  $T_2$  stream is 42% gaseous O<sub>2</sub> by mass and 58% gaseous H<sub>2</sub>O by mass at 660 K. Both the  $T_1$  and  $T_2$  streams are fed with constant mass flow rates, 0.67 kg/s and 19.75 kg/s, respectively. A non-reflective boundary condition is imposed at the downstream end of the computational domain with the goal of suppressing longitudinal acoustics in the streamwise direction, which promotes the generation of transverse acoustic waves in the spanwise direction. Similar to the single-injector configuration in Section 6.1, combustion is represented by the flamelet progress variable (FPV) model with GRI-1.2 chemical kinetics.

As shown in **Figure 8A**, the five-injector model rocket combustor can be represented by two reference components, the interior injector element (e.g.,  $\Omega_3$ ) and the wall injector element (e.g.,  $\Omega_5$ ). Therefore, the multi-injector configuration can be modeled using two ROMs trained based on the reference components, denoted as an all-ROM framework, as illustrated in **Section 5**. With the ROM/FOM coupled framework demonstrated using the single-injector configuration, we use the multi-injector configuration to evaluate and demonstrate the ROM/ROM coupled framework.

When the FOM is adopted to model all five components, denoted as all-FOM framework, the resulting representative snapshots of the flow fields are shown in **Figure 8B**, which exhibits two major features that have not been observed in the single-injector case in **Section 6.1**: 1) stronger interactions between components, featured with large-scale vortex shedding approaching the downstream end of the domain; and 2) more complex dynamics at the component interfaces (i.e.,  $\partial\Omega_{12}$ ,  $\partial\Omega_{23}$ ,  $\partial\Omega_{34}$ , and  $\partial\Omega_{45}$ ), featured with both positive and negative spanwise velocity.

### 6.2.1 Injector-Element Reduced-Order Models Training and Framework Integration

In this section, we discuss the ROM training strategies based on the two different components (the interior and wall injector elements) identified above and how to integrate the two trained ROMs into the framework to model the multi-injector model rocket combustor. We focus on the component-based methods for ROM training as illustrated in Figures 9A,B corresponding to the interior and wall injector element respectively. To incorporate the strong interactions between injector elements observed in Figure 8B, two adjacent injector elements are included for interior injector-element ROM training (Figure 9A) while one additional adjacent injector element is added for wall injectorelement ROM training (Figure 9B), which cannot be easily accounted by imposing boundary conditions directly as conceptualized in Figure 2 considering the complexity of the dynamics at the component interfaces. In addition, similar to the single-injector configuration, auxiliary domains with exponentially stretched mesh elements in the spanwise direction are added next to the additional injector elements in ROM training (two for interior injector-element and one for wall injectorelement). This incorporates the complex dynamics at the component interfaces, especially for the abrupt changes in the directions of the flow characteristics (e.g., existence of both positive and negative spanwise velocity) observed in Figure 8B. Non-reflective boundary conditions are imposed at the downstream end for both the interior and wall injector element ROM training to be consistent with the target multi-injector configuration (Figure 8A). Forcing is imposed via non-reflective boundary conditions at the side boundaries with backward characteristics  $q_{u-c}$  perturbed using the same function in Eq. 19 to generate the essential





dynamics anticipated in the full system. Here, we choose to impose the forcing with A = 0 to mimic a broad-and response, which presumably contains rich responses in the frequency domain. The solution snapshots are extracted corresponding to the regions bounded by dashed lines in **Figures 8A,B**, which are then used to construct the interior injector-element ROM and the wall injector-element ROM, respectively. The resulting ROMs are coupled through the direct flux matching method at the component interfaces adopted to model all the interior injector elements ( $\Omega_2$ ,  $\Omega_3$ , and  $\Omega_4$  in **Figure 8A**) and  $\overline{\Omega}_{II}$  for all the wall injector elements ( $\Omega_1$  and  $\Omega_5$  in **Figure 8A**), resulting in the all-ROM framework as shown in **Figure 9C**.

Since the two wall injector elements are geometrically identical (i.e., reflective symmetry) to each other,  $\overline{\Omega}_{II}$  is mirrored about the center axis to generate a reflective counterpart when adopted to model  $\Omega_1$ , reflected as  $\overline{\Omega}_{II}$ . Comparisons of the training domains in **Figures 9A,B** with the full five-element configuration in **Figure 9C**, suggest that the training costs in **Figures 9A,B** are similar to that of the full five-element configuration. The advantage of this approach is that the training costs is fixed for any number of elements (e.g., three, seven, nine or more) and

for larger systems, substantial cost savings can be realized. The current work serves as a first step toward modeling practical rocket engines which typically consist of hundreds of injector elements.

#### 6.2.2 Performance

Based on the investigations using the single-injector configuration in **Section 6.1**, we apply the adaptive-basis method, introduced in **Section 4**, to develop the two component-based ROMs,  $\overline{\Omega}_{I}$  and  $\overline{\Omega}_{II}$ . Similar to the single-injector case, 10 snapshots are collected from the offline ROM training stage for each component in **Figures 9A,B** to generate the initial two sets of trial basis  $\mathbf{V}_p^0$ . 5 POD modes covering > 99.9% of the total energy are selected to construct adaptive-basis ROMs *via* MP-LSVT formulation, respectively. The trial basis  $\mathbf{V}_p^k$  is adapted at each time step *k* based on the formulation in **Eq. 13** and following the schematics in **Figure 9C**, the adaptive-basis ROMs are then coupled to model 3 multi-injector configurations with three, five, and seven injector elements.

Next, we proceed to evaluate the performance of the alladaptive-basis-ROM framework based on the results from the



Qol	Number of Injectors	€р	€Ţ	€IJ
Time-averaged Fields	3	$4.02 \times 10^{-3}$	$3.57 \times 10^{-2}$	$3.50 \times 10^{-2}$
	5	$4.07 \times 10^{-3}$	$3.66 \times 10^{-2}$	$3.69 \times 10^{-2}$
	7	$5.37 \times 10^{-3}$	$3.25 \times 10^{-2}$	$3.89 \times 10^{-2}$
RMS Fields	3	1.42 × 10 <sup>-1</sup>	5.36 × 10 <sup>-2</sup>	8.16 × 10 <sup>-2</sup>
	5	$1.81 \times 10^{-1}$	$6.13 \times 10^{-2}$	$9.48 \times 10^{-2}$
	7	$1.45 \times 10^{-1}$	$5.63 \times 10^{-2}$	$9.32 \times 10^{-2}$

**TABLE 5** Comparisons of the errors in predicting time-averaged and RMS fields of pressure (*P*), temperature(*T*), and axial velocity (*U*) corresponding to **Figures 10D-E** using the all-adaptive-basis-ROM framework for the multi-injector combustor.

FOM, which is taken as the ground truth solution. **Figures 10A–C** compares the local PSD predicted using the FOM and with the all-adaptive-basis-ROM framework, which are measured at the corner of the left wall, i.e., (x, y) = (0, 0) in **Figure 8A** for all 3 multi-injector configurations. Different from the single-injector cases in **Section 6.1**, the true pressure signals from the all-FOM results in the multi-injector configurations do not exhibit distinguishable coherent oscillating patterns as in **Figure 4**. This aspect can be expected to be more challenging to be predicted by the ROM.

The predictions using the all-adaptive-basis-ROM framework show good agreement with the all-FOM framework results, especially in capturing the changes in PSD distributions due to the configuration variations with the number of injector elements increased from three to seven. For example, the wideband frequency peak between 2,500 and 3,000 Hz in the threeinjector combustor and the peak near 2,500 Hz for the seveninjector configuration are both well predicted by the alladaptive-basis-ROM framework. More importantly, the broad-band PSD distributions (i.e., no identifiable frequency peaks) in the five-injector configuration are also accurately captured.

Having successfully demonstrated the ability of the adaptive ROM to capture changes in pressure oscillations arising from configuration variations, we next look at their ability to predict time-averaged and RMS fields of target state variables (P, U, and T) is assessed. The accuracy of the framework is evaluated based on the errors defined in Eq. 22, summarized in Table 5. It can be readily seen that the all-adaptive-basis-ROM framework is able to accurately predict the time-averaged fields of selected state variables with errors below 4% while even for the generally more challenging RMS fields, the prediction errors are shown to be below 18%, given the complexity and chaotic features of the dynamics present in the multi-injector problems, indicated by the broad-band frequency distributions in Figures 10A-C. Specifically, the time-averaged, RMS, and representative unsteady fields of temperature (T) are selected to demonstrate the predictive capabilities of the all-adaptivebasis-ROM framework as shown in Figures 10D,E for the three different multi-injector configurations, which exhibits good agreement between all-adaptive-basis-ROM and all-FOM framework results. But it still need to be pointed out that the all-adaptive-basis-ROM framework predicts elongated hightemperature zones between injector elements compared to the all-FOM framework results, indicating that the current methodology may require further improvement.

# **7 CONCLUSION**

A component-based domain-decomposition framework is established for the modeling of large-scale systems that cannot be directly accessed using the high-fidelity simulations (e.g., a rocket engine, a wind farm, and a compressor). This approach decomposes the full system into different components, each of which can flexibly adopt different modeling strategies (e.g., ROM or FOM), balancing physical complexity with accuracy requirements. Under the premise that most of the components share identical features and can be represented by a few reference components, a component-based reduced-order model (ROM) training strategy is proposed and demonstrated, which requires only the high-fidelity simulations of the individual components. System-level feedback and responses in the training dataset is emulated by imposing boundary forcing. This leads to a significant saving in computational cost during ROM training. The model-form preserving least-squares with variable transformation (MP-LSVT) ROM formulation is pursued with enhancement through basis adaptation to construct the component-based ROMs. The trained ROMs can be adopted to model components with identical geometric features and coupled with either a reduced-fidelity full-order model (RF-FOM) or ROMs via a direct flux matching method to enable both accurate and efficient simulations of large-scale systems with different geometric configurations.

Detailed evaluations of the framework were first performed based on a planar single-injector model rocket configuration with varying combustor lengths, each of which exhibit different dynamic behaviors. The framework separates the singleinjector configuration into two components, the upstream injector element and the downstream combustor + nozzle, the former of which adopts MP-LSVT ROM for modeling, while the latter adopts a FOM. Two methods, (system-based and component-based) are used to train the injector-element ROM. It was demonstrated that the upstream-component ROMs from both training methods, when coupled with the downstream-component FOM, can produce reasonably accurate predictions of the pressure oscillations while the component-based method requires much less computational cost. However, the ROM/FOM framework encounters difficulties in representing the time-averaged and root-meansquare (RMS) fields of the target state variables while distinguishable solution mismatches are observed at the component interface. To address this limitation, basis adaptation is incorporated in the MP-LSVT formulation to

enhance ROM capabilities, which significantly improves the predictive accuracy of the framework and more importantly, is capable of representing changes in dynamic behaviors due to the variations in combustor length.

The framework was then extended to a 2D-planar multiinjector model rocket configuration with different number of injector elements, which can be represented by two reference injector-element components. High-fidelity training simulations are then conducted on the two reference components to develop the component-based ROMs *via* MP-LSVT formulation with basis adaptation. The framework is demonstrated to be capable of predicting all the quantities-of-interest (QoIs) accurately, including local pressure oscillations, time-averaged and RMS fields of target state variables for the multi-injector configuration with different number of injector elements.

Though preliminary and demonstrated for only 2D problems, the component-based domain-decomposition framework with adaptive-basis ROMs is directly applicable to 3D problems and mostly importantly serves as a stepping stone towards modeling practical large-scale engineering systems (e.g., a RD-170 rocket engine [16]). Before this framework can be adopted by engineers in many-query applications (such as design and uncertainty quantification) of the full system, two major aspects need to be considered: 1) efficiency—hyper-reduction has to be considered to enable more efficient ROM calculations as mentioned in **Section 4**; and 2), scalability—the ROMs must be amenable for execution on memory-restricted computers such as desktop workstations or embedded systems—i.e., they need to be load-balanced and scalable in terms of computational resources available.

Discussions are provided by the current authors [14] on constructing scalable, load-balanced, and hyper-reduced staticbasis ROMs while all these aspects remain to be further investigated for adaptive-basis ROM development. To address the remaining gaps, good avenue for future work can be on incorporating adaptive sparse sampling methods (e.g., [50]) in the adaptive-basis ROM to achieve computational efficiency

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enhancement, while exploring dynamic methods to achieve scalability when the sampling elements are getting adapted. In addition, the component-based framework is designed to be generally compatible with different types of ROM methods and hence instead of the intrusive ROM used in the current work, non-intrusive ROM methods [44,78] may also be considered for the future work.

## DATA AVAILABILITY STATEMENT

The raw data supporting the conclusion of this article will be made available by the authors upon request.

## **AUTHOR CONTRIBUTIONS**

CH, KD, and CM all contributed to formulation of the component-based reduced-order modeling framework and the component-based ROM training strategies. CH established the test problems, perform the numerical investigations, and wrote the first draft of the manuscript. KD formulated the adaptive-basis method provided in the manuscript and contributed to the method for ROM integration in the framework. CM formulated the component-based ROM framework and conceptualized its application for rocket combustion problems. All authors contributed to manuscript revision, read, and approved the submitted version.

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