



Welding Residual Stress Distribution of U-Rib Stiffened Plates of Steel Box Girders and its Influence on Structural Natural Frequencies

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This paper mainly explores the distribution of welding residual stress of U-rib stiffened plates of steel box girders and examines the influence of welding residual stress over the natural frequencies of these plates. The research focuses on two aspects, namely, the distribution of welding residual stress, and the dynamic behavior of U-rib stiffened plates in the context of that distribution. Firstly, we build two finite element models to analyse the welding process and study the dynamic behavior. Next, the authors discussed how different geometric parameters, as well as welding parameters, affect the residual stress distribution. In addition, the welding residual stress curves were fitted for steel box girders of varied sizes, and a simplified formula was deduced for the distribution of welding residual stress. After that, the simplified formula was applied to analyze the natural frequencies of stiffened plates in multiple sets of models. Through the analysis of natural frequency under different boundary conditions, it was learned that the welding residual stress has a remarkable influence on the natural frequencies of stiffened plates.

Keywords: steel box girder, U-rib stiffened plate, welding residual stress, numerical simulation, natural frequencies

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1 INTRODUCTION

Over the past few decades, steel box girders have been widely employed in long-span bridges. Each steel box girder mainly consists of stiffened plates that contain welding residual stresses. How residual stresses affect the stiffened plates has attracted extensive attention from researchers. The bridge deck, which rests on the girders, carry most of the various traffic loads on the bridge structure (Cui et al., 2018a). However, the existing studies have not fully grasped the natural frequencies of stiffened plates under complex conditions, and the experimental data are insufficient due to high measuring errors. It is necessary to consider the residual stress in welding process because of the welding deformation. Generally speaking, residual stress is considered to be detrimental to the structure, it will affect the fatigue of the structure, the yield limit of the material, the brittle failure of the metal, and the corrosion cracking. It is obvious that the study of residual stress is a key factor in studying the influence of the structure. Study on residual stress should be an important part in this paper.

In addition, it is difficult to predict the residual stresses accurately in the past. In the present study, machine learning algorithms have been used to predict residual stress and natural frequency of vibrations (Das et al., 2020a). Therefore, it is extremely important to examine the natural frequencies of stiffened plates in the context of residual stresses.

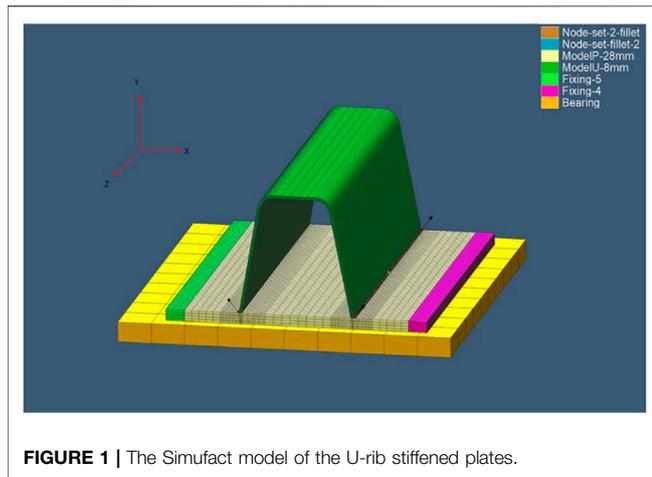


FIGURE 1 | The Simufact model of the U-rib stiffened plates.

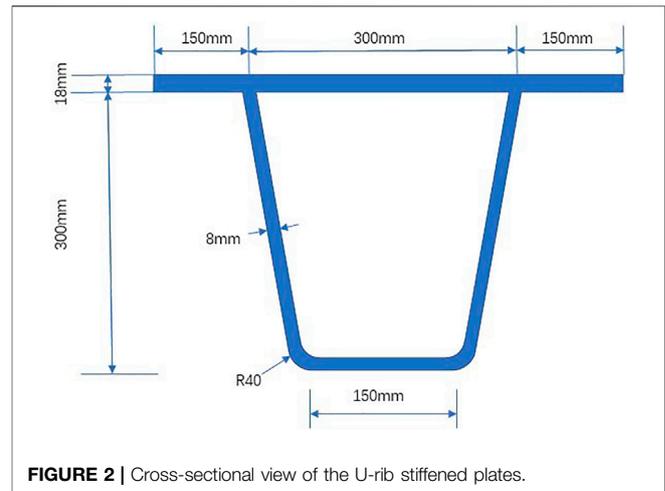


FIGURE 2 | Cross-sectional view of the U-rib stiffened plates.

Much research has been done on the influence of the welding process over the residual stress distribution in various welding structures (El-Axir, 2002; Zhao and Wu, 2012a; Lin and Yang, 2014; Wu and Kim, 2018; Zheng et al., 2018). But most of the published works are not deep enough into U-rib stiffened plates. The predictions on residual stress distribution are inconsistent with the actual situation. Ohashi et al. (1996) measured the residual stresses of the trapezoidal rib stiffened plates of Duodoro Bridge, and, on this basis, proposed a model to calculate the distribution of residual stresses.

Owing to high cost and large data dispersion, it is a challenge to capture the true distribution of residual stresses through experiments (Deng and Murakawa, 2006; Qiang et al., 2018; Rikken et al., 2018; Salerno et al., 2018). The computer-aided technique provides an economical and effective way to simulate the welding residual stresses (Fujikubo and Yao, 1999; Gannon et al., 2010; Cui et al., 2018b; Gadallah et al., 2018; Jiang et al., 2018). Zhao and Wu (Zhao and Wu, 2012b) simulated the welding temperature field and stress field, calculated the residual stress of the U-rib stiffened plates, and derived the residual stress distribution related to U-rib thickness and plate thickness.

In 1974, Herrmann and Brunelle demonstrated that the residual stresses affect the stiffness, which in turn impacts the dynamic behavior of isotropic welded plates. Since then, many researchers have investigated the effects of residual stress on various structures, and even on micro structures (Olfatnia et al., 2010; Wang and Wang, 2012; Yang et al., 2019). F. Wu (Wu, 1999) theorized that thermal residual stresses significantly affect the natural frequencies, and thermal residual stresses can be tailored to increase natural frequencies. Das, D and Pratihari, D.K (Das et al., 2020b) find a correlation between the measured welding stress and natural frequency of vibration.

Lieven and Greening (2001); Ballestra et al. (2008) illustrated that the changes in resonance behavior resulting from pre-stress offer exciting prospects in model analysis, and experimentally proved that the natural frequencies are affected by the residual stress.

In actual engineering, the diversity of welding parameters adds difficulty to the implementation of the accurate distribution law

of residual stress of the welding structures. In most cases, the residual stress distribution is very complicated. To solve these issues, this paper simulates the welding process of U-rib stiffened plates with changing welding parameters. After determining the stress field, the authors plotted the fitting curve, and presented a simplified distribution formula. Taking the Hong Kong-Zhuhai-Macao Bridge as the engineering background, this paper relies on the simplified formula and shell model analysis to investigate how welding residual stress influences the natural frequencies of stiffened plates of steel box girders.

2 THERMAL ANALYSIS

2.1 Modeling

2.1.1 Welding Process of U-Rib Stiffened Plates

Welding is the most common technique to join two components together. As is known to all, the finite-element analysis of U-rib stiffened plates usually adopts shell elements. However, the shell elements are not suitable for analyzing the process of thermal welding. To solve the problem, this paper proposes a 3D solid element type in the welding model to investigate the welding process, and thereby extract the distribution of residual stresses. **Figure 1** shows the Simufact model of the U-rib stiffened plates, which are on the thick yellow platform prepared for welding. Note that fixing-4 and fixing-5 are fixed boundary conditions, both of which are constrained in the x -direction. At the beginning of the welding, the U-rib stiffened plates were laid down on the bearing. All the boundary conditions were kept constant until the end of the cooling process. After setting all the geometries and material properties, two welding robots began to work along the weld line. The robots weld independently of each other, with a 200 s cooling period between them. The total working time of each robot is the same. The entire process lasted 3,000s, until the temperature dropped to 20°C.

2.1.2 Model Description

Figure 2 details the cross-section of the U-rib stiffened plates. The width, i.e., the x -direction dimension in the finite-element

TABLE 1 | Welding parameters.

Current	Voltage	Speed	Efficiency
280 A	28 V	55 cm/min	80%

TABLE 2 | Heat source parameters.

b	d	a _r	a _r
5 mm	8.53 mm	1.6 mm	4.2 mm

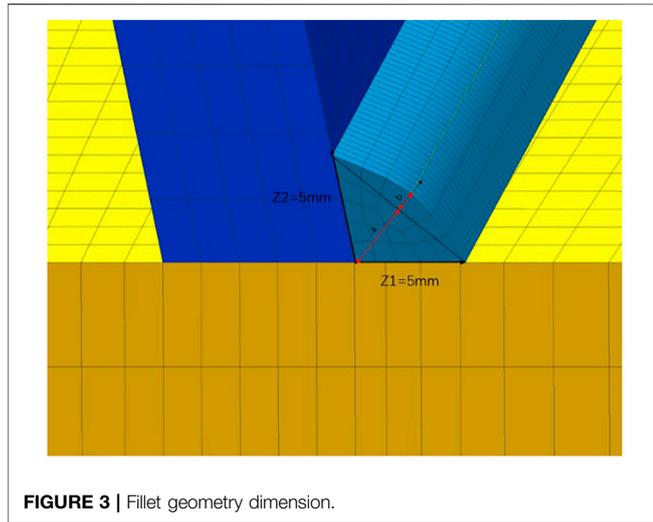


FIGURE 3 | Fillet geometry dimension.

TABLE 3 | Fillet geometry.

a	b	Z1	Z2
3.078 mm	-2 mm	5 mm	5

model, and the height, i.e., the *y*-direction dimension in the finite-element model, are 600 and 300 mm, respectively. The U-rib thickness varies from 6 to 24 mm, while the master plate thickness varies from 18 to 24 mm. It is assumed that each U-rib stiffened plate is made of the same material, with the welding progressing from the purple side to the light green side. The density, elastic modulus, and Poisson's ratio are denoted by ρ , E , and μ , respectively, the thickness of the U-rib as t_1 , the thickness of the master plate as t_2 , and the width of the model as t_3 . In actual welding of U-rib stiffened plates, partial penetration is often adopted. Our simulation considers only one-bead pass.

2.2 Simulation Parameters

2.2.1 Simulation Theory

The numerical simulation of welding begun in the 1970s, when Japanese researchers Ueda and Yamakawa proposed the thermoplastic finite-element method (FEM). Considering the

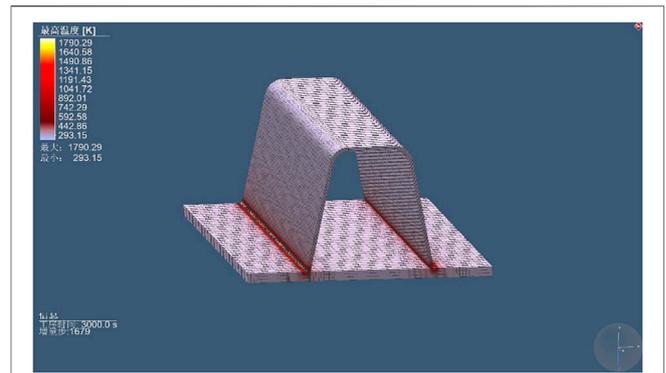


FIGURE 4 | Model of temperature field.

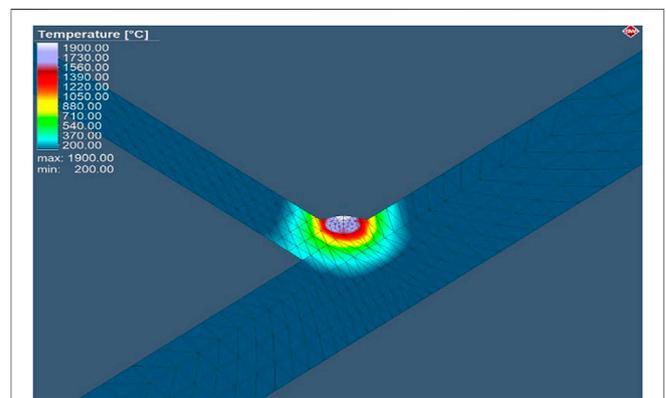


FIGURE 5 | Melting pool of the model.

variation of materials with temperatures, the basic theory of dynamic residual stress in welding (Ueda and Yamakawa, 1971) can be described as:

$$dF = \int_V B^T D d\epsilon dV - \int_V B^T C dT dV \quad (1)$$

where, dF is the node load; V is the integral domain.

If the material is in an elastic phase, then

$$d\epsilon = d\epsilon^e + d\epsilon^T = d\epsilon^e + \alpha dT \quad (2)$$

where, $d\epsilon$ is the strain matrix; D and C are the matrices related to the constitutive relationship and temperature, respectively; dT is the temperature increment; α is the expansion coefficient.

The value of $d\epsilon^e$ is the sum of $d\epsilon^{e'}$ and $d\epsilon^{T'}$:

$$d\epsilon^e = d\epsilon^{e'} + d\epsilon^{T'} = D^{-1} d\epsilon^e + S dT \quad (3)$$

where, $S = D^{-1} \frac{\partial D}{\partial T} D^{-1} \sigma$.

Multiplying both sides of Eq. 3 by D , and substituting the result into Eq. 2:

$$d\sigma = D d\epsilon - D(\alpha + S) dT \quad (4)$$

TABLE 4 | Welding models.

Model name	U-rib thickness (t1)	Master plate thickness (t2)
Case 1 (mm)	6	18 mm
Case 2		24 mm
Case 3		28 mm
Case 4	8	18 mm
Case 5		24 mm
Case 6		28 mm
Case 7	10	18 mm
Case 8		24 mm
Case 9		28 mm

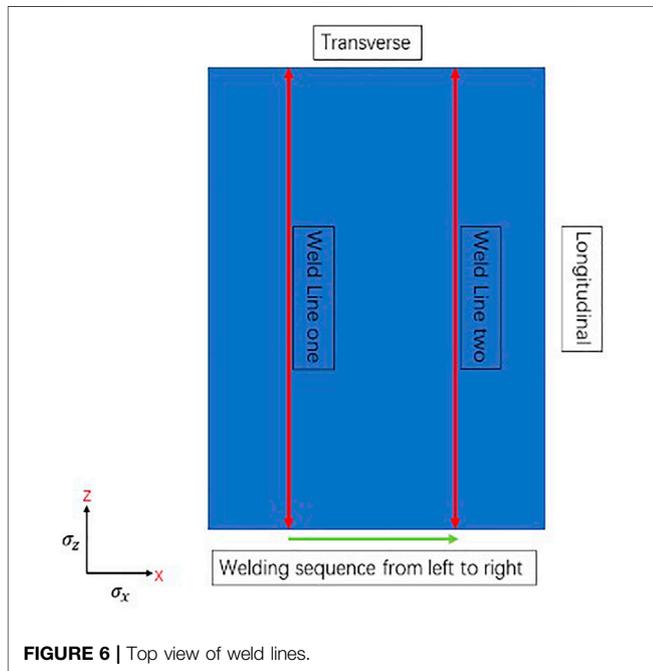


FIGURE 6 | Top view of weld lines.

During plastic molding, it is necessary to consider the plastic strain $d\epsilon^p$:

$$d\epsilon = d\epsilon^e + d\epsilon^T + d\epsilon^p \tag{5}$$

Considering the flow rule and the von Mises yield criterion, the constitutive relationship can be expressed as:

$$d\sigma = D^p d\epsilon - \left(D^p \alpha - D^p \frac{1}{E} \frac{dE}{dT} \epsilon^e - \frac{2}{3} \sigma_y \frac{d\sigma_y}{dT} N \right) dT \tag{6}$$

where, D^p is molding matrix; E is Young's modulus; σ_y is yield strength; N is a matrix related to stress component.

The welding materials are the same as the steel box girders of Hong Kong-Zhuhai-Macao Bridge. The primary material is Q345qD U-rib stiffened plates. The welding rods are of the material E5015. The material properties vary with temperatures. The thermodynamic property is an important parameter to predicting the distribution of residual stress.

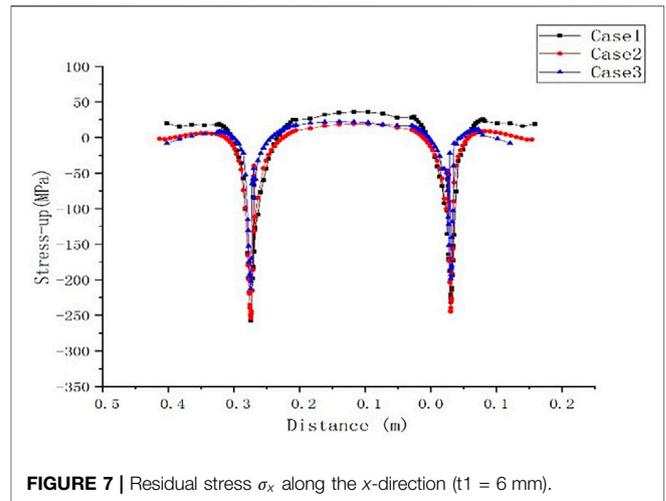


FIGURE 7 | Residual stress σ_x along the x-direction ($t1 = 6$ mm).

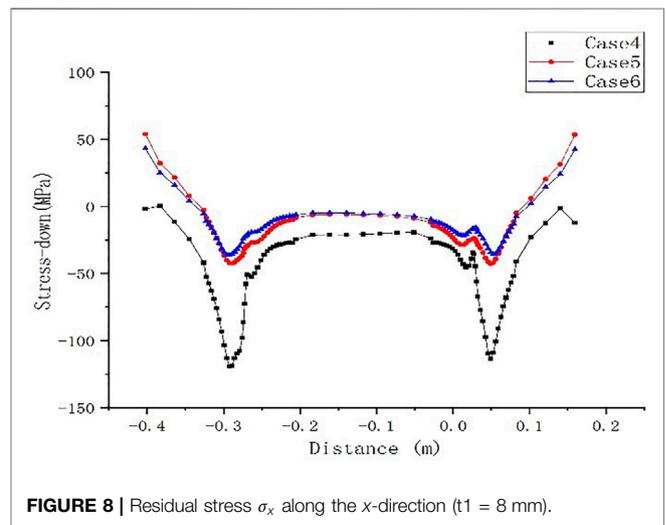


FIGURE 8 | Residual stress σ_x along the x-direction ($t1 = 8$ mm).

2.2.2 Thermal Analysis

Before predicting the residual stress, transient thermal analysis should be conducted to derive the temperature field during welding. The governing equation for the transient temperature field can be expressed as:

$$\rho C_p(T) = \frac{\partial T}{\partial t} \frac{\partial}{\partial x} \left(\lambda(T) \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(\lambda(T) \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial z} \left(\lambda(T) \frac{\partial T}{\partial z} \right) + Q(x, y, z, t) \tag{7}$$

where ρ is density; $C_p(T)$ is the specific heat varying with temperatures; $\lambda(T)$ is thermal conductivity varying with temperature; $x, y,$ and z are the three directions in the coordinate system; Q is the internal heat generation rate; t is the time.

The internal heat Q can be defined as:

$$Q = \frac{\eta UI}{v} \tag{8}$$

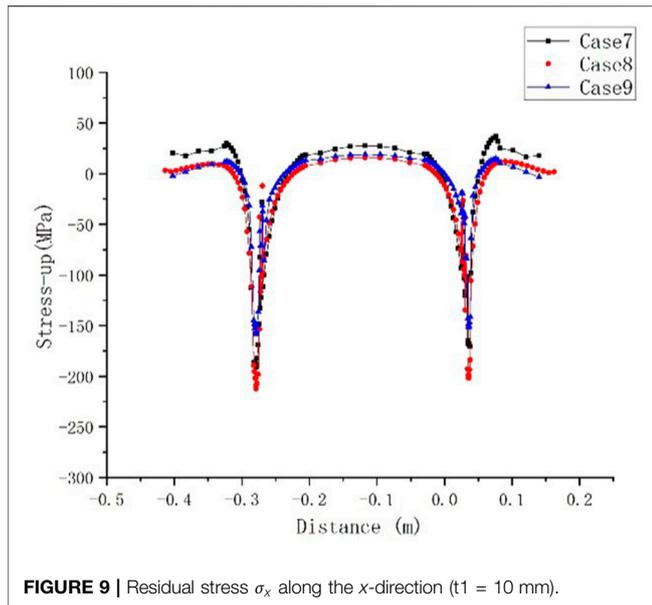


FIGURE 9 | Residual stress σ_x along the x-direction ($t_1 = 10$ mm).

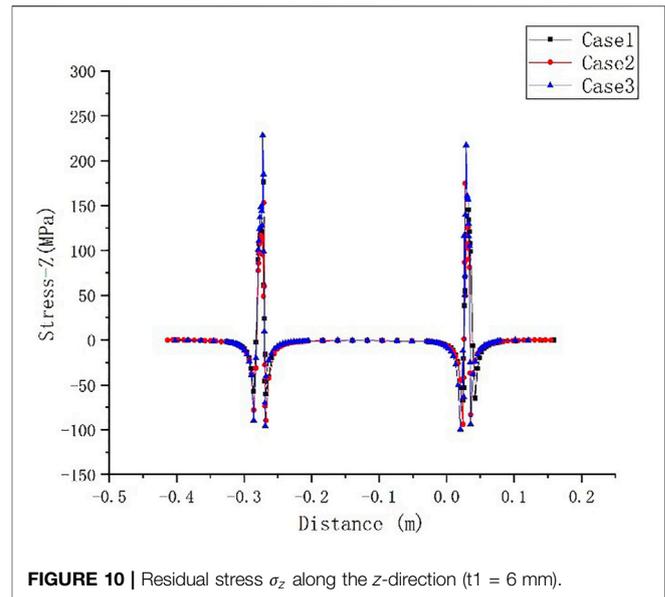


FIGURE 10 | Residual stress σ_z along the z-direction ($t_1 = 6$ mm).

where, U is the voltage; I is the current; η is the efficiency; v is the welding speed.

According to the Goldak heat source model, the distribution can be expressed as:

$$q_f = \frac{6\sqrt{3}f_1Q}{\pi a_f b d \sqrt{\pi}} \exp\left(-3\frac{x^2}{a_f^2}\right) \exp\left(-3\frac{y^2}{b^2}\right) \exp\left(-3\frac{z^2}{d^2}\right) \quad (9)$$

$$q_r = \frac{6\sqrt{3}f_2Q}{\pi a_r b d \sqrt{\pi}} \exp\left(-3\frac{x^2}{a_r^2}\right) \exp\left(-3\frac{y^2}{b^2}\right) \exp\left(-3\frac{z^2}{d^2}\right) \quad (10)$$

where, q_f is for the front heat source; q_r is for the rear heat source.

The concept of heat source aims to model the isothermal surface of the real melt pool. The software Simufact Welding provides a conventional model introduced by Goldak et al. (Goldak et al., 1984) for arc welding.

2.3 Material Parameters and FE Simulation

Material properties are the key in FE simulation of welding process. Q345qD is a widely used material among Chinese bridges, as per Structural Steel for Bridge(GB/T 714-2015). But the experimental data on the thermodynamic properties of Q345qD are rare and insufficient. JMatPro is a commercial material property software for thermal parameter calculation. It can provide data on thermodynamic properties regarding temperature change. According to the various components in Q345qD steel, the thermal-physical material properties can be obtained between 200°C and 2,000°C. This approach has been widely accepted in the steel industry, owing to its high efficiency, economy, and stability (Saunders et al., 2003). It can be learned that the materials in the master plate and U-ribs have the same properties. The phase transformation calculated by JMatPro is also crucial to the analysis on the material properties of Q345qD.

The welding parameters and heat source parameters are listed in Tables 1, 2. The fillet geometry is depicted in Figure 3; Table 3. After

obtaining the material properties, we can use them in 3D welding model. Because of the complexity of the welding process and the involvement of nine welding models, it is time-consuming to calculate each model in turn. Thanks to the symmetry of the welded U-rib stiffened plate, the residual stress distribution can be analyzed on a local scale. Without considering the multi-pass welding, the total length of the plate is 1.2 m. In total, the U-rib stiffened plate was meshed into 60,500 elements, while the master plate was divided into 28,160 elements. The element density was increased in the welded area. The model of the temperature field and its melting pool are illustrated in Figures 4, 5, respectively.

The welding lasted for around 2,000 s. After that, cooling was carried out until the temperature dropped to 20°C. The heat source plays a significant role in the welding process. The heat source parameters were configured based on the design drawings of the Hong Kong-Zhuhai-Macao Bridge. Considering the different plate sizes, the authors split the simulation data into nine groups. The heat source parameters employed in welding analysis are as follows: current = 280 A, voltage = 28 V, welding speed = 55 cm/min, and efficiency = 80%. The welding models share the same material parameters and heat source parameters, but differ in geometry. In addition, the geometric parameters were properly adjusted to create a realistic heat flow through the plate surface and the melting pool.

2.4 Residual Stress Distribution

The multiple models (Table 4) were designed to reflect how the residual stress distribution varies with thicknesses. As shown in Figure 6, there were weld lines along with the weld bead in two directions. The welding proceeded from the left to the right. Although the residual stresses were mainly distributed in the longitudinal direction, the stresses in other directions cannot be ignored during the natural frequencies analysis.

The residual stress distribution was examined in two directions: σ_x (longitudinal stress) along the weld line, and σ_z

(transverse stress) perpendicular to the weld line. The residual stress distributions of the nine cases were compared in both directions. The stresses σ_y were ignored, because the stresses vary little in the y -direction. In addition, σ_x means stress in the x -direction, σ_y means stress in the y -direction, σ_z means stress in the z -direction.

It can be observed from **Figures 7–12** that the residual stresses in Cases 1, 4, and 7 (master plate thickness = 18 mm) were greater than those in other cases. In other words, the compressive residual stress is negatively correlated with master plate thickness. Apparently, the residual stress was not very much affected by the changing thickness of the master plate. In **Figures 10–12**, the positive stress was almost twice that of the negative stress in the same position of the master plate; the stress σ_z (longitudinal stress) ranged between 130 and 225 MPa. The σ_z value changed rapidly from tensile state to compressive state around the edge of the weld line.

The simulation reveals that the stress near the weld line changed rapidly in a very narrow range. The variation of residual stress should not be ignored, during the prediction of natural frequencies. Based on the simulation results, the residual stress curve was fitted, and used to initialize the shell elements for analysis. The fitting curve of residual stress is supposed to generate a standard formula. It would be very convenient to take initial stress by the standard formula in subsequent analysis of natural frequencies.

3 FITTING CURVES OF RESIDUAL STRESS

To further analyze with the residual stresses, a half model was selected for curve fitting, in which the U-rib is 6 mm thick, and the master plate is 18 mm thick. The curves were fitted with a cubic polynomial function, and the correlation coefficient surpassed 0.9. The fitting results are illustrated in **Figures 13, 14**.

A,B,C,D is the coefficient of polynomial, which is determined by fitting curves. l means the length of motherboard. The coefficient values of $f_{\sigma_x}(x)$ and $f_{\sigma_z}(x)$ are given in **Tables 5, 6**, respectively.

$f_{\sigma_x}(x)$ means stress equation of piecewise function in x -direction.

$f_{\sigma_z}(x)$ means stress equation of piecewise function in y -direction.

The first and second equations can be expressed as:

$$f_{\sigma_x}(x) = \begin{cases} A_1 + A_2 \times x^1 + A_3 \times x^2 + A_4 \times x^3, & \text{when } \begin{cases} 0 \leq x < \frac{l}{4} \\ \frac{l}{4} \leq x < \frac{l}{2} \end{cases} \end{cases}, \text{ when } \begin{cases} 0 \leq x < \frac{l}{4} \\ \frac{l}{4} \leq x < \frac{l}{2} \end{cases} \quad (11)$$

$$f_{\sigma_z}(x) = \begin{cases} C_1 + C_2 \times x^1 + C_3 \times x^2 + C_4 \times x^3 \\ D_1 + D_2 \times x^1 + D_3 \times x^2 + D_4 \times x^3 \\ E_1 + E_2 \times x^1 + E_3 \times x^2 + E_4 \times x^3 \end{cases}, \text{ when } \begin{cases} 0 \leq x < \frac{l}{4} \\ \frac{l}{4} \leq x < \frac{5l}{12} \\ \frac{5l}{12} \leq x < \frac{l}{2} \end{cases} \quad (12)$$

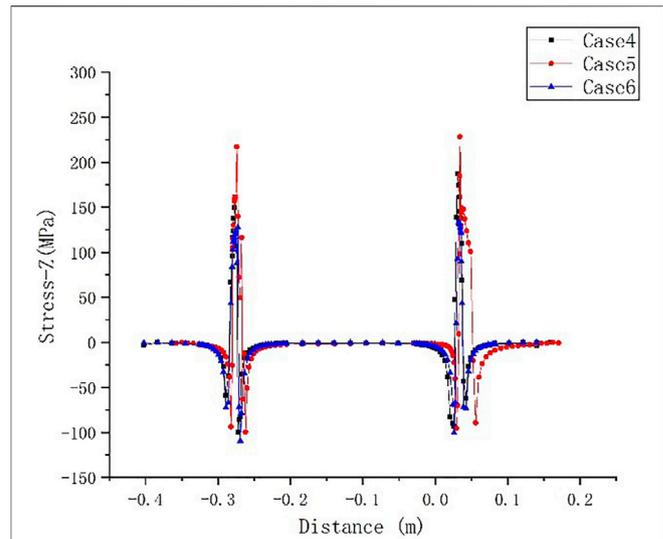


FIGURE 11 | Residual stress σ_z along the z -direction ($t_1 = 8$ mm).

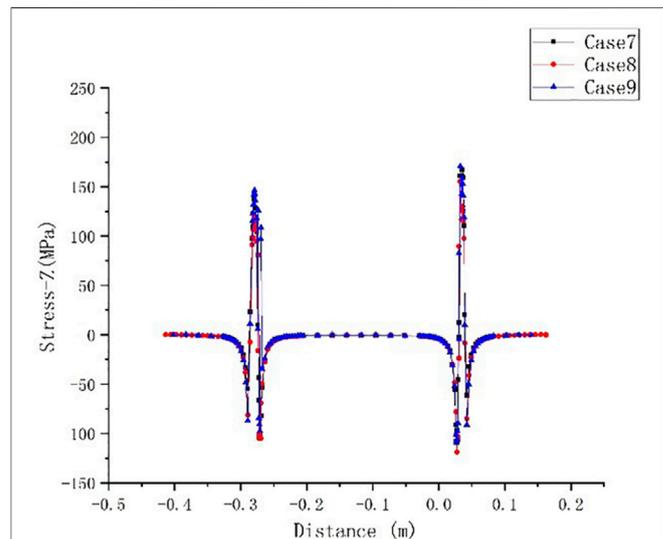


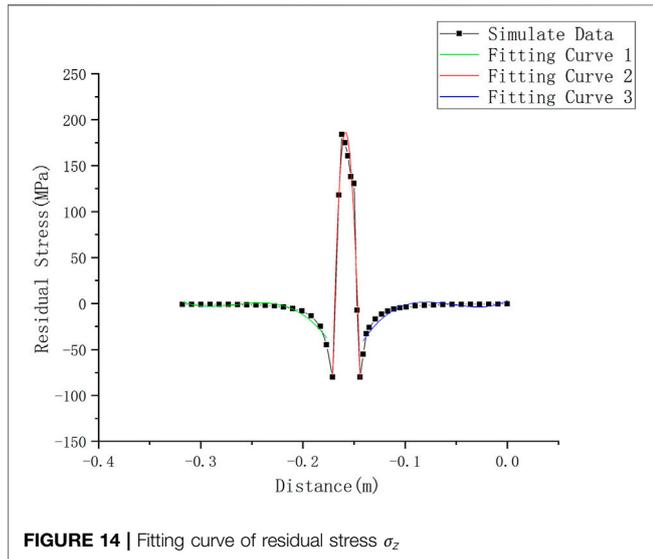
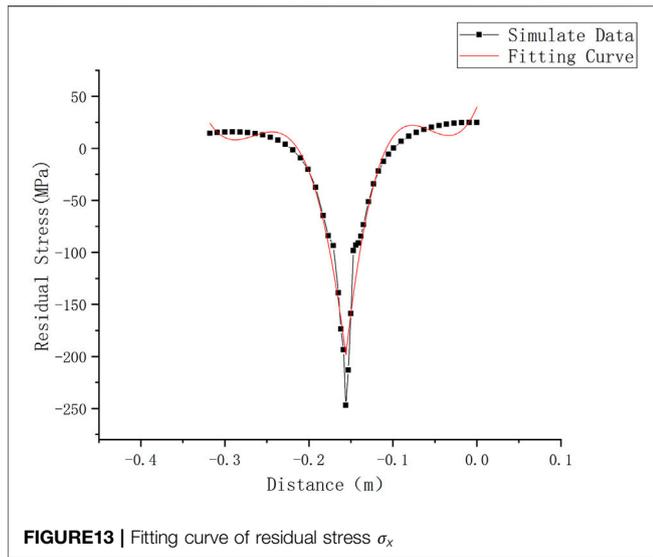
FIGURE 12 | Residual stress σ_z along the z -direction ($t_1 = 10$ mm).

The residual stress distribution at each thickness of the master plate can be derived with the equation parameters. Then, the residual stresses can be applied to the calculation model using the fitting curve.

4 MODEL ANALYSIS AND DISCUSSION

4.1 Applying Initial Stresses to the Shell Model

For analyzing the effect of residual stress, a three-dimensional thermodynamic finite element model is extracted through the SIMFACT welding model. Six residual stress parameters,



corresponding to s_{11} , s_{22} , s_{33} , s_{12} , s_{23} , s_{13} , respectively, are extracted and applied to the U-rib stiffened plate model simulated with shell 181. s_{11} , s_{22} , s_{33} represent the normal stresses in X, Y, Z direction and s_{12} , s_{23} , s_{13} represent the in-plane stresses in shell elements.

Since the plates are modeled by shell elements, only three parameters s_{11} , s_{22} , and s_{12} are applied to correctly simulate the analysis of the shell 181 elements. The analysis of the residual stress field in three directions considers the vibration model analysis caused by the initial stress field. Two boundary conditions have been compared in our dynamic analysis. In fixed boundary conditions, both sides of the edges have been locked in all directions. (The displacement $x, y, z = 0$ and rotation $U_x, U_y, U_z = 0$) In simply supported conditions, the displacement $x, y = 0$ and rotation $U_y, U_z = 0$. The displacement z and rotation U_x are not constrained in simply supported conditions.

In order to apply residual stress more easily, we can apply it in a piecewise function. The range of piecewise function is determined by the specific welding times and welding positions. The same structure size of welding plates (including the mother plate and stiffened plate) will generate a specific same trend fitting curve sample. We can find the fitting curve sample of the corresponding structure size and use the fitted piecewise function to apply for the actual mother plate. When actual structure size distance is measured, we put measured distances into the corresponding curve sample to predict the residual stress without using the physical experiment to measure the results.

4.2 Natural Frequencies Analysis

To verify the accuracy of the finite-element model of the plate and shell, the rectangular-rib stiffened plate in H.A.Xu's work (Xu et al., 2010) was selected as a comparison. According to the parameters of that stiffened plate, two stiffeners were set separately in the longitudinal and lateral directions. The finite-element model of the stiffened plate was built with shell 181 elements. In Abaqus, the residual stress can be applied as initial state stress in the predefined field. The dimensionless frequency parameters in work H.A. Xu's (Xu et al., 2010) were converted into frequency values, and compared with the values calculated by the Abaqus model. The comparison shows that the initial stress condition affects the model analysis of stiffened plates. Drawing on the literature (Xu et al., 2010; Gao et al., 2014), the initial stress condition also affects the natural frequencies of the U-rib stiffened plate. **Tables 7, 8** compare the residual stresses under two types of boundary conditions. The first 10 frequencies were selected for comparative analysis and the first four mode shapes were depicted.

TABLE 5 | Coefficient values for $f_{\sigma_x}(x)$.

Coefficient	Value	Coefficient	Value
A1	-3.18E + 03	B1	4.05E + 01
A2	-3.64E + 04	B2	1.99E + 03
A3	-1.37E + 05	B3	4.23E + 04
A4	-1.72E + 05	B4	2.53E + 05

TABLE 6 | Coefficient values for $f_{\sigma_z}(x)$.

Coefficient	Value	Coefficient	Value	Coefficient	Value
C1	-1.21E + 03	D1	1.40E + 04	E1	3.08E + 00
C2	-1.39E + 04	D2	4.99E + 05	E2	5.43E + 02
C3	-5.26E + 04	D3	4.65E + 06	E3	1.30E + 04
C4	-6.59E + 04	D4	1.30E + 07	E4	8.01E + 04

TABLE 7 | Natural frequencies with fixed boundary conditions (Hz).

Mode	Original state model	Initial stress model	Differences (%)
1	50.101	54.132	8.04
2	87.795	100.32	8.58
3	124.5	129.23	3.79
4	128.45	145.98	10.22
5	154.47	165.77	7.32
6	182.29	198.85	9.08
7	238.45	251.75	5.58
8	238.59	268.35	12.47
9	266.53	284.32	6.67
10	293.02	314.52	7.34

Tables 7 and 8 show that the stiffened plate had slightly lower natural frequencies under fixed boundary conditions than under simply supported conditions. When the initial stresses were considered, the natural frequencies of the stiffened plate under fixed boundary conditions was approximately 4–10% lower than those of the stiffened plate under simply supported conditions. This effect was so obvious that the initial stresses cannot be neglected for simply supported boundary conditions and fixed boundary conditions. The above results indicate that the effect of initial stresses on the natural frequencies of stiffened plates depends on actual conditions.

5 CONCLUSION

This paper intends to numerically analyze the natural frequencies of a stiffened plate with initial stresses. After obtaining all the calculation results from the welding process, the authors built a shell model in Abaqus to analyze the natural frequencies of stiffened plates. The effect of residual stresses was fully considered in the shell model-based simulation. Through the research, the following conclusions were drawn:

- 1) The results of the thermal simulation and shell models are consistent with the findings in the literature. This means the thermal simulation and shell model in Abaqus are feasible and reliable tools for vibration analysis of stiffened plates.
- 2) The analysis of residual stress distribution shows that the residual stress varied clearly with the thicknesses of the master plate. As the plate became thinner, the absolute value of residual stress increased, especially in the welding bead. It also means the peak residual stresses in the U-rib changes rapidly with thickness.

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TABLE 8 | Natural frequencies with simply supported conditions (Hz).

Mode	Original state model	Initial stress model	Differences (%)
1	36.44	39.993	9.74
2	73.46	76.641	4.33
3	104.6	115.71	10.62
4	120.96	123.96	2.48
5	132.4	143.51	8.38
6	170.72	181.25	6.17
7	215.28	234.33	8.85
8	221.32	236.83	7.01
9	238.86	260.42	9.03
10	270.94	292.74	8.05

- 3) The residual stress had an obvious effect on natural frequencies of stiffened plates under simply supported conditions and fixed supported condition. The effect of initial stresses on the natural frequencies of stiffened plates should be considered according to actual conditions.

DATA AVAILABILITY STATEMENT

The original contributions presented in the study are included in the article/Supplementary Material, further inquiries can be directed to the corresponding author.

AUTHOR CONTRIBUTIONS

LC: review and editing, Software TW: original draft preparation, methodology JP: Project administration, supervision NM: Validation, supervision RW: Funding acquisition, validation.

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