

Precision Compensation Control of a Piezo-Scanning Mechanism

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In this study, a controller based on two compensators is developed to support a two-axis piezo-scanning mechanism for tracking control. The hysteresis compensator is designed based on a Prandtl–Ishlinskii hysteresis model in order to reduce the nonlinear hysteresis effect of a piezoelectric actuator. Furthermore, to compensate for uncertainties due to parametric variations, hysteresis-compensated error, and un-modeled dynamics, the uncertainties compensator based on the neural network disturbance observer is proposed. The developed controller is verified with regard to control performance by experiment. Those two observers are used to complete hysteresis compensation and disturbance compensation, which will not reduce the stability and bandwidth of the system and improve the control accuracy. Experimental results show that the proposed hybrid controller can overcome the mentioned nonlinearity and uncertainty efficiently and preserve good positioning accuracy with high-bandwidth varying frequencies (1–150 Hz).

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1 INTRODUCTION

Ultra-precision positioning is quite demanding for many manufacturing processes and inspection techniques. For example, the ultra-precision machining, micro-nano manufacturing, ultra-large-scale integrated circuit manufacturing, diamond turning and grinding, and scanning probe microscopy require high position accuracy (Sang et al., 2007; Faa-Jeng Lin et al., 2009; Cheng et al., 2015; Zhu and Rui, 2017). Different from the traditional positioning system using ball screw, the piezoelectric actuator (PA) has been successfully implemented in many ultra-precision positioning mechanisms due to its advantages of fast response, small size, large output force range, and ultra-high resolution (Zhu and Rui, 2016).

Unfortunately, since the material of PA is usually ferroelectric, its responses to applied voltage exhibit hysteresis-characteristic nonlinearity (Gu et al., 2016). The hysteresis nonlinearity can degrade the overall control system performance and reduce tracking accuracies (Wang et al., 2019).

In order to restrain the hysteresis, the feedforward compensation control based on hysteresis models is usually used (Yangmin Li and Li, 2013). The main idea is to develop a mathematical model to represent the hysteresis. Then, the inverse hysteresis model is cascaded with the piezo-drive mechanism to linearize the response. In this regard, performance of the feedforward control depends heavily on the exactness of the hysteresis model. Here, the exactness of the model is in the sense that the latter can describe the system behavior correctly. Many hysteresis models have been proposed in the literature. These include the Preisach model (Truong and Ahn, 2013; Tang and Li, 2015), the Prandtl–Ishlinskii (P-I) model (Liu et al., 2013; Yang and Zhu, 2020), the Bouc–Wen model (Ikhouane et al., 2007; Zhu et al., 2017), Maxwell's slip model (Quant et al., 2009), the LuGre model (Bashash and Jalili, 2008), and the polynomial model (Huang and Chiu, 2009). However, uncertainties due to parametric variations, hysteresis-compensated error, and unmodeled

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dynamics will lead to large control error. Previous studies generally used the closed-loop error control mode to improve the control accuracy, which would reduce the system bandwidth and lead to large high-frequency positioning errors (Mercorelli and Werner, 2014; Mercorelli and Werner, 2016). In this study, a hysteresis compensator is designed based on a P-I hysteresis model and an uncertainties compensator based on a neural network disturbance observer is adopted to compensate for those uncertainties.

This article is organized as follows. First, the P-I hysteresis model is proposed in **Section 2**. Next, the hybrid control method is presented in **Section 3**. Then, in **Section 4**, the experiments are carried out to validate the proposed control method. Finally, the conclusion is made in **Section 5**.

2 MODELING OF THE TWO-AXIS PIEZO-SCANNING MECHANISM

2.1 Structure of the Mechanism

Figure 1 shows the piezo-scanning mechanism (P16. XY20, input voltage range: 0–120V, *X* and *Y* axis output displacement range: $\pm 8 \,\mu$ m, no-load resonant frequency: 2000 Hz, produced by the Harbin Core Tomorrow Co., Ltd., China). In the mechanism, the load platform is connected to two PAs through flexible hinges. The identical kinematic chains, flexible hinges, and PAs in two axes guarantee the uniform dynamic characteristics within the workspace.

2.2 P-I Hysteresis Model of the Mechanism

A discrete play operator in the P-I model of the *X*-axis is defined as follows:

$$x(kT) = H_r[u, x_0](kT) = \max\{u(t) - r, \min[u(t) + r, x(kT - T)]\}$$
(1)

$$x(0) = \max\{u(0) - r, \min[u(0) + r, x_0]\}$$
(2)

where *r* is the input threshold sometimes called the magnitude of the backlash, *T* is the sampling period, *k* indicates the sampled time instant, *u* is the input, *x* represents the mechanism response, and x_0 is the initial state and is usually initialized to 0.

By multiplying some such operators H_r with different thresholds and weights, the P-I model is obtained as follows:

$$x(kT) = \Gamma(u(kT)) = \boldsymbol{\omega}_h^T \mathbf{H}_r[u, \mathbf{x}_0](kT)$$
(3)

where $\boldsymbol{\omega}_{h}^{T} = [\omega_{h0} \cdots \omega_{hN}]$ representing the weight vector of the play operators, $\mathbf{H}_{r}[u, \mathbf{x}_{0}](kT) = [H_{r}[u, x_{00}](kT), \cdots, H_{r}[u, x_{p0N}](kT)]^{T}$ with the threshold vector $\mathbf{r} = [r_{0} \cdots r_{N}]^{T}$, where $0 = r_{0} < \cdots < r_{N}$.

Similarly, the P-I model of the Y-axis can be obtained.

3 HYBRID CONTROL METHOD

Obviously, due to the existence of the hysteresis, the relationship between the output displacement and input voltage of the piezo-scanning mechanism is multi-valued non-linear. In order to improve the accuracy in the motion control, it is necessary to compensate for the hysteresis. Therefore, a feedforward loop is used to compensate and reduce the hysteretic effect, and a feedback control based on a neural network disturbance observer is developed to compensate for the lumped uncertainty due to parametric variations, hysteresis-compensated error, and unmodeled dynamics. The block diagram of the hybrid control is presented in **Figure 2**.

3.1 Hysteresis Compensator

The inverse P-I hysteresis model of the *X*-axis can be obtained as follows (Bashash and Jalili, 2008):

$$u_{\mathrm{PI}x}\left(x\left(kT\right)\right) = \Gamma^{-1}\left(x\left(kT\right)\right) = \boldsymbol{\omega}_{h}^{'T}\mathbf{H}_{r}\left[\mathbf{x},\mathbf{x}_{0}^{'}\right]\left(kT\right)$$
(4)





where

$$\boldsymbol{\omega}_{h}^{'T} = \begin{bmatrix} \boldsymbol{\omega}_{h0}^{'} \cdots \boldsymbol{\omega}_{hN}^{'} \end{bmatrix}$$
(5)

$$\omega_{h0}^{'} = \frac{1}{\omega_{h0}}, \omega_{hi}^{'} = \frac{\omega_{hi}}{\left(\omega_{1} + \sum_{j=1}^{i} \omega_{hj}\right) \left(\sum_{j=1}^{i-1} \omega_{hj}\right)} \quad i = 1, 2, \cdots, N \quad (6)$$

$$r'_{i} = \sum_{j=0}^{i} \omega_{hj} (r_{i} - r_{j}) \quad i = 0, 1, 2, \cdots, N$$
 (7)

$$x_{0i}^{'} = \sum_{j=0}^{i} \omega_{hj} x_{0i}^{'} + \sum_{j=i+1}^{N} \omega_{hj} x_{0j}^{'} \quad i = 0, 1, 2, \cdots, N$$
(8)

Similarly, the inverse P-I model of the Y-axis can be obtained.

From the inverse P-I hysteresis model given by **Eqs 4–8**, the feedforward control can be designed. The schematic representation of the feedforward control is shown in **Figure 2**.



3.2 Uncertainties Compensator

The uncertainties compensator control is established relying on an uncertainties observer in this section. The uncertainties observer based on the radial basis function neural network (NN) is developed to observe the uncertainties. The radial basis function NN observer is presented in **Figure 3**. The inputs are $e_x(t)$, $e_y(t)$, $\dot{e}_x(t)$, and $\dot{e}_y(t)$, and the output observers are the observer uncertainties \hat{u}_{fx} and \hat{u}_{fy} .

The output of the input layer is as follows:

$$\boldsymbol{O}^{(1)} = \boldsymbol{E} \tag{9}$$

where $\boldsymbol{E} = [\boldsymbol{e}_x, \boldsymbol{e}_y, \dot{\boldsymbol{e}}_x, \dot{\boldsymbol{e}}_y]^T$.

The output of the hidden layer can be written as follows:

$$\boldsymbol{O}^{(2)} = \boldsymbol{R}(\boldsymbol{E})$$

$$= \left[\exp\left(-\left\|\frac{\boldsymbol{E}-\boldsymbol{c}_{1}}{\boldsymbol{d}_{1}}\right\|^{2}\right), \cdots, \exp\left(-\left\|\frac{\boldsymbol{E}-\boldsymbol{c}_{j}}{\boldsymbol{d}_{j}}\right\|^{2}\right), \cdots, \exp\left(-\left\|\frac{\boldsymbol{E}-\boldsymbol{c}_{m}}{\boldsymbol{d}_{m}}\right\|^{2}\right) \right]^{T}$$
(10)

where $\mathbf{R}(\mathbf{E}) = [\mathbf{R}_1(\mathbf{E}), \dots, \mathbf{R}_j(\mathbf{E}), \dots, \mathbf{R}_m(\mathbf{E})]^T$ is the Gaussian basis function; $\mathbf{c}_j = [c_{j1} \ c_{j2} \ c_{j3} \ c_{j4}]^T$ and $\mathbf{d}_j =$



 $\begin{bmatrix} d_{j1} & d_{j2} & d_{j3} & d_{j4} \end{bmatrix}^T$ are the center and width vector of the *j*th basis function, respectively.

The output of the compensator can be given by the following:

$$O^{(3)} = \begin{bmatrix} \hat{u}_{fx}, \, \hat{u}_{fy} \end{bmatrix}^{\mathrm{T}} = \hat{u}_{f} (E, w, b) = w^{\mathrm{T}} R(E) + b \qquad (11)$$

where $w = \begin{bmatrix} w_{11} & w_{12} \\ w_{21} & w_{22} \\ \vdots & \vdots \\ w_{m1} & w_{m2} \end{bmatrix}, \, b = \begin{bmatrix} b_{1} & b_{2} \end{bmatrix}^{\mathrm{T}}.$

4 EXPERIMENT AND DISCUSSION

4.1 Hysteresis Modeling Results

First of all, a harmonic applied voltage is employed to motive the piezo-scanning mechanism. The practical and estimated responses are illustrated in **Figure 4**. It can be seen from **Figure 4** that a good consistence between the practical and estimated result is observed. Good modeling accuracy using the P-I hysteresis model suggests that the proposed model can closely represent the practical hysteresis in PA.

4.2 Hysteresis Compensation Results

Then, the control performance of the hysteresis compensator is verified by experiment. The experimental result is shown in **Figure 5**. The blue line in **Figure 5A** represents the original hysteresis curve without the compensator. From **Figure 5B**, the maximum linearization error is $0.253 \,\mu\text{m}$. The curves between the displacement and input voltage of the PA without the compensator present apparent hysteresis. The relationships between the displacement and applied voltages of the PA with the compensator based on the P-I model are almost linear. That is to say, the hysteresis compensator based on the P-I model has the ability to accurately linearize PA. It should be pointed out that this error is mainly caused by uncertainties including parametric variations, hysteresiscompensated error, and unmodeled dynamics.

4.3 Tracking Control Results

Furthermore, to validate the positioning performance of the proposed hybrid control method, a harmonic positioning task is performed. The control result is illustrated in **Figure 6**, which indicates the proposed hybrid control method can obtain a high positioning accuracy and the maximum absolute control error is $0.124 \,\mu\text{m}$.

Finally, a complex tracking task with varying amplitude and varying frequencies is implemented. The frequency bandwidth of



 $\ensuremath{\textit{FIGURE 6}}\xspace$ | Tracking control result of the hybrid control method with the harmonic tracking signal.

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the task is 1–150 Hz as shown in **Figure 7A**, and the sampling frequency of the displacement sensors is 10 kHz. The tracking control result of the hybrid control method is shown in **Figures 7B,C**. It can be observed in **Figure 7** that the proposed controller can also accurately track the complex trajectory.

These validate the proposed control method.

5 CONCLUSION

In this study, in order to effectively control the two-axis piezoscanning mechanism, the hybrid controller is proposed and presented. The controller is composed of two compensators. The hysteresis compensator is designed based on the P-I hysteresis model in order to reduce the nonlinear hysteresis effect of PA. The uncertainties compensator based on the neural network disturbance observer is adopted to compensate for uncertainties including parametric variations, hysteresis-compensated error, and

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unmodeled dynamics. Experimental results show that the proposed hybrid controller can overcome the mentioned nonlinearity and uncertainty efficiently and preserve good positioning accuracy.

DATA AVAILABILITY STATEMENT

The raw data supporting the conclusion of this article will be made available by the authors, without undue reservation.

AUTHOR CONTRIBUTIONS

HJ, SL, and XH contributed to conception and design of the study. JS organized the database. CS performed the statistical analysis. All authors contributed to manuscript revision and read and approved the submitted version.

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