



# Development of an Elongated Ellipsoid Heat Source Model to Reduce Computation Time for Directed Energy Deposition Process

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Directed Energy Deposition (DED) Additive Manufacturing process for metallic parts are becoming increasingly popular and widely accepted due to their potential of fabricating parts of large dimensions. The complex thermal cycles obtained due to the process physics results in accumulation of residual stress and distortion. However, to accurately model metal deposition heat transfer for large parts, numerical model leads to impractical computation time. In this work, a 3D transient finite element model with Quiet/Active element activation is developed for modeling metal deposition heat transfer analysis of DED process. To accurately model moving heat source, Goldak's double ellipsoid model is implemented with small enough simulation time increment such that laser moves a distance of its radius over the course of each increment. Considering thin build-wall of Stainless Steel 316L fabricated with different process parameters, numerical results obtained with COMSOL 5.6 Multi-Physics software are successfully validated with experiment temperature data recorded at the substrate during the fabrication of 20 layers. To reduce the computation time, elongated ellipsoid heat input model that averages the heat source over its entire path is implemented. It has been found that by taking such large time increments, numerical model gives inaccurate results. Therefore, the track is divided into several sub-tracks, each of which is applied in one simulation increment. In this work, an investigation is done to find out the correct simulation time increment or sub-track size that leads to reduction in computation time (5–10 times) but still yields sufficiently accurate results (below 10% of relative error on temperature). Also, a Correction factor is introduced that further reduces computation error of elongated heat source. Finally, a new correlation is also established in finding out the correct time increment size and correction factor value to reduce the computation time yielding accurate results.

**Keywords:** DED, modeling—HT, simulation, heat transfer, equivalent heat source, metal deposition, additive manufacturing, elongated ellipsoid heat source

## INTRODUCTION

Directed Energy Deposition (DED) is an additive manufacturing process in which focused thermal energy is used to fuse materials by melting as they are being deposited. “Focused thermal energy” means that an energy source (e.g., laser LDED) is focused to melt the material being deposited (Milewski, 2017). As compared to powder bed techniques, material addition rate is much higher in LDED process [up-to 300 cm<sup>3</sup>/h (Herzog et al., 2016)], hence leading to the possibility of fabricating large parts. Unfortunately, because of the process physics, involving numerous heating and cooling cycles, it leads to generation of unwanted distortion and residual stresses. Researchers have employed Finite Element Method (FEM) to study the thermal gradient induced deformation and stress (Nickel et al., 2001; Zhang et al., 2004; Alimardani et al., 2007; Mukherjee et al., 2017).

FEM techniques to model LDED process is inspired by prior research done on multi-pass weld modeling (Brickstad and Josefson, 1998; Lindgren et al., 1999; Deng and Murakawa, 2006; Bézi and Szávai, 2014; Bonnaud and Gunnars, 2015), because welding process, that has been studied in depth is quite similar to LDED process (Lindgren, 2001a; Lindgren, 2001b; Lindgren, 2001c). In the last few years, modeling techniques from multi-pass welding is applied to Additive Manufacturing (AM) process (Lundbäck and Lindgren, 2011), (Lindgren et al., 2016). But there is a strong difference between welding, LDED and other AM processes, notably in terms of quantity of filler material i.e., deposited material. In welding, filler material volume is relatively low to substrate that requires fewer processing times and computation times. In contrast, in LDED process filler material is much larger in quantity relative to substrate, hence leading to higher processing times. This in turn, requires large computation times (days or months) especially when the same modeling techniques are applied from welding to LDED process. This sort of computation time is not feasible or practical. Because of the large quantity of deposition material in LDED, researchers have focused on different numerical modeling techniques to accurately represent the material addition that leads to reduction in the computation time as well.

Conventionally, researchers have used different numerical techniques to model the material addition for LDED process. A lot of researchers have used the “quiet/active” method that is easy to implement and does not require equation re-numbering but can be computationally expensive (Wang et al., 2008; Chew et al., 2015; Denlinger et al., 2015; Yang et al., 2016; Biegler et al., 2018a; Johnson et al., 2018; Ren et al., 2019). Some researchers have employed “Element Birth” technique that is difficult to implement as it requires solver initialization and equation re-numbering at every simulation, time step elements are activated, but can be computationally faster (Labudovic et al., 2003; Farahmand and Kovacevic, 2014; Biegler et al., 2018b). “Hybrid Activation” method that takes the advantage of both methods of Quiet Activation and Element Birth has been used as well (Heigel et al., 2015; Denlinger and Michaleris, 2016; Biegler et al., 2020). A detailed explanation of all these metal deposition models is well presented in the work (Michaleris, 2014). Another

approach that is used by researchers to represent material addition consists of using dynamic mesh based on Arbitrary Lagrangian Eulerian Method (ALE) (Morville et al., 2012; Peyre and Dal, 2017; Morville, 2021).

Besides the material addition modeling, heat source model also influences the computation time. An Elongated Ellipsoid Heat Source Model is developed and demonstrated to reduce the computation time for Selective Laser Melting (SLM) process (Irwin and Michaleris, 2016). The proposed model significantly reduces the problem of temporal discretization originated due to moving heat source and leads to lesser computation time (Irwin and Michaleris, 2016). Also, a dimensionless number is presented that is used to define and optimise the size of computation increment (time-step) that in turn measures and activates the length of the linear segment over which elongated ellipsoid is spread. It was proved that with an increase in size of computation increment, computation time is reduced but it leads to stretch of Elongated Ellipsoid heat Source over longer deposition lengths that results in an increase of computation errors (Irwin and Michaleris, 2016). The model was demonstrated and validated for Ti-6Al-4V and Inconel 625.

Till date, to the best of the knowledge of the author, elongated ellipsoid heat source is not tested and validated for LDED process for any material. Because SLM and LDED processes are different in every aspect related to the physical phenomena and scale, they have therefore different computation requirements due to the size of the laser spot (Heat Source) and build dimensions.

Despite all the contributions and valuable results of the developed models in the past, there is still a need to reduce the computation time for the Numerical Model dedicated to LDED process as the modeling strategy suggested in the literature will lead to impractical computation time. Since, LDED is a material addition process that involves material deposition of track size in hundreds of meters for large part, material deposition model can therefore be an important feature to reduce the computation time of the Numerical Model for LDED process.

This work serves to validate a strategy by introducing efficient material activation by utilising Elongated Ellipsoid Heat Source model in order to reduce the computation time of FEM for LDED process. First, a transient thermal model using quiet activation strategy that employs Double Ellipsoid Heat Source Model (DE) is validated against *in situ* temperature measurements recorded during the deposition of 20-layer high, Stainless Steel 316L wall builds with varying inter-layer dwell times and laser powers. Next, a transient thermal model with proposed efficient material activation method employing Elongated Ellipsoid Heat Source Model (EE) is used to model the thermal behaviour that leads to drastic decrease of computation time. The computational accuracy and time are then compared with experiment results and Double Ellipsoid Model. A correction factor is introduced to compensate for computation error that increase due to the increase of size of simulation time increment or sub-track segment. At last, an analytical co-relation is developed between simulation time increment/sub-track size, computation accuracy and computation time that is helpful in finding the optimised value of simulation time increments that results in achieving the objective of computation error of less than 10%.

**TABLE 1** | Temperature dependent material properties of SS 316L (Mills and Mills, 2002).

T (°C)	k (W.m <sup>-1</sup> .°C <sup>-1</sup> )	C <sub>p</sub> (J.kg <sup>-1</sup> .°C <sup>-1</sup> )	ρ (kg.m <sup>-3</sup> )
25	13.4	470	7950
100	15.5	490	7921
200	17.6	520	7880
400	21.8	560	7785
600	24.5	590	7681
800	27.2	630	7575
1000	29.1	660	7462
1200	30.9	700	7361
1300	31.1	710	7311

## MODELING APPROACH

The methodology in the proposed numerical model focus on the thermal fields, simplifies the melt-pool fluid and powder dynamics to reduce computational cost. The developed LDED model discretizes the continuous physical process of laser metal deposition into a combination of simulation steps, in which laser travel is considered sequential step-by-step. During each time step, the thermal analysis is performed and the resulting temperature field is saved. The proposed numerical model architecture can be applied to simulate any multi-bead or multi-layer parts.

### Thermal Analysis

Assuming a Lagrangian frame  $\Omega$  and a material point located by  $r$  ( $r \in \Omega$ ) as the reference, given thermal energy balance at time  $t$ , the governing equation can be formulated as follows:

$$\rho(T)C_p(T) \frac{\partial T(r, t)}{\partial t} = -\nabla \cdot q(r, t) + Q(r, t), \quad r \in \Omega \quad (1)$$

where  $\rho$  is the material density,  $C_p$  is the specific heat capacity,  $T$  is the temperature,  $t$  is the time,  $Q$  is the heat source, and  $q$  is the heat conduction flux vector, calculated as:

$$q = -k(T)\nabla T(r, t) \quad (2)$$

Where  $k$  is the thermal conductivity of the material.

**Table 1** presents the temperature dependent thermal properties for SS 316L (Mills and Mills, 2002). Linear interpolation is used to calculate the properties at any temperature.

### Heat Source Models

The Single Ellipsoid (SE) heat input model (Goldak) can be used to describe an equivalent to the laser heat source (Goldak et al., 1984) as:

$$Q_{SE} = \frac{6\sqrt{3}AP}{abc\pi\sqrt{\pi}} \exp\left(-\frac{3(x+v_s t)^2}{a^2} - \frac{3y^2}{b^2} - \frac{3z^2}{c^2}\right) \quad (3)$$

The laser power is  $P$  and the laser absorption efficiency is  $A$ . The value for laser power  $P$  is based on measurement, as will be discussed in *Experiment Set-Up Section*. The value of  $A$  is calibrated using the method of reverse calibration by

iteratively fitting the simulated temperature field at thermocouple location to match the experiment results described in Ref. (Denlinger et al., 2015).  $x$ ,  $y$  and  $z$  are the local coordinates with the origin centred at the ellipsoid where the heat source reaches the maximum intensity and with a moving velocity  $v_s$ . Parameters  $a$ ,  $b$  and  $c$  represent the respective length of the longitudinal, transverse and through the depth semi-axes of the ellipsoid parallel to the local  $x$ ,  $y$  and  $z$  axes. Generally,  $a$  is taken as the melt-pool length,  $b$  is taken as the half width of the deposition bead and  $c$  to the melt pool depth (Denlinger et al., 2015).

It has been shown previously that, using single ellipsoid heat source predicts a lower temperature gradient at the front and a higher temperature gradient at the trailing edge as compared to the experiments (Goldak et al., 1984). Therefore, it has been a common practise among the researchers to employ double ellipsoid heat source to be more computationally accurate. The double ellipsoid (DE) Heat Input model (Goldak) is used in the present work to describe the laser heat source (Goldak et al., 1984):

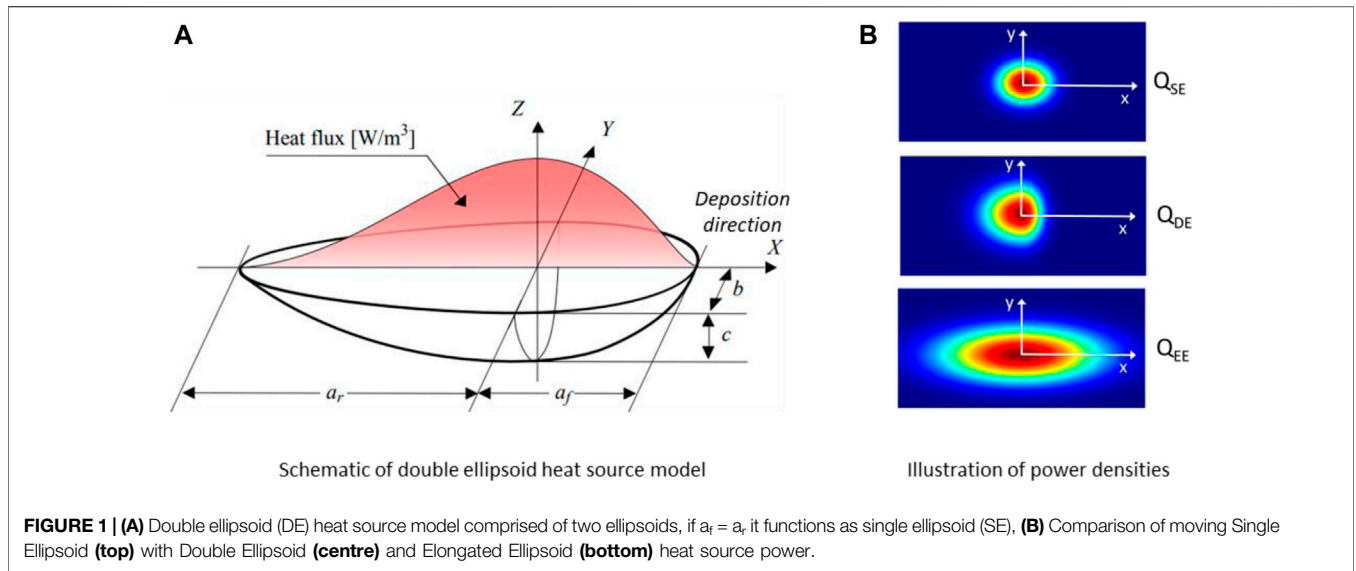
$$Q_{DE} = \frac{6\sqrt{3}APf_{f,r}}{a_{f,r}bc\pi\sqrt{\pi}} \exp\left(-\frac{3(x+v_s t)^2}{a_{f,r}^2} - \frac{3y^2}{b^2} - \frac{3z^2}{c^2}\right) \quad (4)$$

$f$  is a weighting fraction, that determines the energy partition among the front and rear ellipsoid. Typically, different values are employed in the front and rear of the heat source for the longitudinal axis length  $a$  (Goldak et al., 1984). The orientation of the double ellipsoid (DE) heat source and coordinate system are depicted in **Figure 1A**. For the single ellipsoid (SE) heat source the lengths of two ellipsoids are equal ( $a_f = a_r$ ), hence depicting a single ellipsoid.

To accurately model the motion or movement of laser (input heat source), the simulation time increments should be small enough in such a way that the numerical heat source should move less than or equal to half of laser spot size i.e., spot size radius in one-time step (Irwin and Michaleris, 2016). Therefore, in the present work for all cases using Double Ellipsoid Heat Source, computation time-step does not exceed  $R/v_s$ , where  $R$  is laser spot size radius and  $v_s$  is laser source/travel speed. In LDED, track length is big and with the specified computation time increment, it can lead to high computation time. Therefore, to reduce the computation time, an elongated ellipsoid line heat input model can be used that averages the single ellipsoid heat source over its path (Irwin and Michaleris, 2016).

$$Q_{AVERAGE} = \frac{1}{\Delta t} \int_{t_0}^{t_0 + \Delta t} Q_{SE} dt \quad (5)$$

where  $t_0$  is the time at beginning of the increment and  $\Delta t$  is the duration of time increment over which SE model is averaged. But this averaging over its path will result in large thermal gradients at the edges of each segment especially when the linear segment is large ( $v_s \Delta t \gg a$ ) (Irwin and Michaleris, 2016). Therefore, to smooth out the discontinuities at the segment edges, elongated ellipsoid model is developed where peak value of the heat input is at the middle of the sub-track or segment. Power density of



elongated ellipsoid at the beginning and end of each segment is half of its peak value that results in smooth distribution over successive linear segments. Thus, Elongated Ellipsoid (EE) line heat input model is also used in present work to describe the laser heat source (Irwin and Michaleris, 2016):

$$Q_{EE} = \frac{6\sqrt{3}AP}{\hat{a}bc\pi\sqrt{\pi}} \exp\left(-\frac{3\left(x + v_s\left(t + \frac{1}{2}\Delta t\right)\right)^2}{\hat{a}^2} - \frac{3y^2}{b^2} - \frac{3z^2}{c^2}\right) \quad (6)$$

where the length of each elongated ellipsoid sub-track or segment  $\hat{a}$  is (Irwin and Michaleris, 2016):

$$\hat{a} = \frac{v_s\Delta t}{2} \sqrt{\frac{3}{\log 2}} \quad (7)$$

The main difference between Eq. 3 and Eq. 6 are the introduction of elongated length  $\hat{a}$  in place of  $a$ , that stretches the heat distribution in the local  $x$  direction and additional  $(1/2)\Delta t$  which shifts the peak heat input from the end to the middle of the linear segment or sub-track (Irwin and Michaleris, 2016). Also, the double ellipsoid heat source moves continuously along the path replicating laser movement in experiments with computation time step less than  $R/v_s$ , the elongated ellipsoid heat source also moves continuously along the path, but enables bigger time steps equal to  $\Delta t$  that results in reduction of computation time. A comparison of the energy distribution with three different heat source models SE, DE and EE is shown in Figure 1B. A detailed analysis of the Elongated Ellipsoid Heat Source and its functioning is the presented in the reference work (Irwin and Michaleris, 2016).

Elongated Ellipsoid heat source model allows the simulation of an entire heat source scan in one time increment instead of hundreds of time increment when using Double Ellipsoid or Single Ellipsoid Heat Source. However, taking large time

increments also leads to increase in errors as well (Irwin and Michaleris, 2016). Therefore, the track scan is divided into several linear segments (sub-tracks), for each of which, heat source is applied in one simulation time increment.

### Thermal Boundary Conditions

The thermal governing equations are supplemented by the initial condition and different types of boundary conditions:

$$T(\mathbf{r}, 0) = T_0(\mathbf{r}, 0) \quad \mathbf{r} \in \Omega \quad (8)$$

Thermal radiation  $q_{rad}$  is accounted for using the Stefan-Boltzmann law:

$$q_{rad} = \epsilon \sigma (T^4 - T_{amb}^4) \quad (9)$$

where  $\epsilon$  is the surface emissivity,  $\sigma$  is the Stefan-Boltzmann constant,  $T$  the surface temperature of the workpiece, and  $T_{amb}$  is the ambient temperature.

Newton's law of cooling describes the heat loss due to convection  $q_{conv}$ :

$$q_{conv} = h (T - T_{amb}) \quad (10)$$

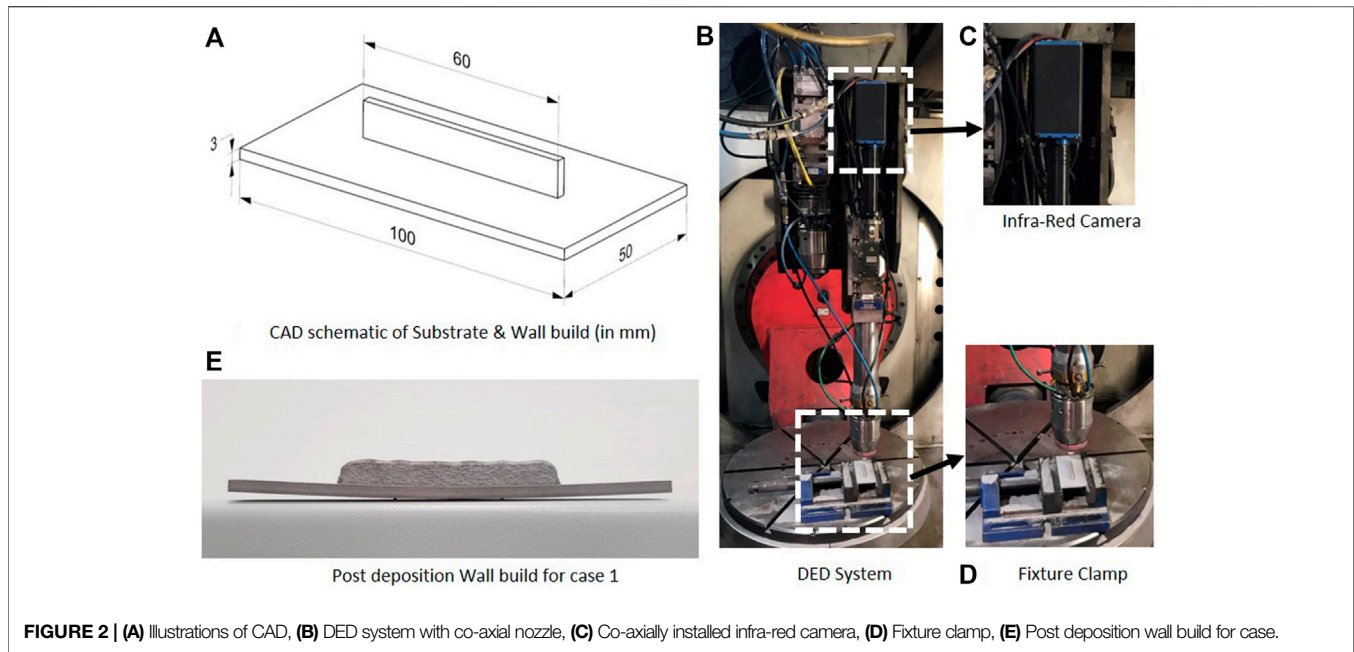
where  $h$  is the convective heat transfer coefficient.

### Latent Heat of Fusion and Marangoni Flow

The effect of latent heat of fusion during the melting and solidification process is accounted by modifying the specific heat capacity  $C_p^*$  as in Eq. 1, where  $L_f$  (J/kg) is latent heat of fusion, and  $T_m$  is melting temperature (Yan et al., 2017).

$$C_p^*(T) = \frac{L_f}{T_m - T_{amb}} + C_p(T) \quad (11)$$

In order to take the convective redistribution of heat in the melt pool due to the fluid flows into account, a higher value of thermal conductivity is considered, hence avoiding integrating complex analytic form in the model (Lampa et al., 1997). Thus,



**FIGURE 2 |** (A) Illustrations of CAD, (B) DED system with co-axial nozzle, (C) Co-axially installed infra-red camera, (D) Fixture clamp, (E) Post deposition wall build for case.

enhanced thermal conductivity factor  $k^*$  is assumed in Eq. 2 (Ren et al., 2019):

$$k^*(T) = \begin{cases} k(T), & T < T_m \\ 2.5 \times k(T), & T \geq T_m \end{cases} \quad (12)$$

## EXPERIMENT SET-UP

The modeling approach laid out in *Modeling Approach Section* is applied to simulate the experimental results acquired for SS 316 L wall builds done with different sets of experiment in an attempt to capture the thermal behaviour. A detailed explanation of the experiment is provided here. Single wall structures were fabricated on a 100 mm long, 50 mm wide, and 3 mm thick SS 316 L substrate (matching material) using a laser-directed energy deposition process. In-house developed MAGIC machine is used for DED system equipped with a 2 kW Diode laser by IPG laser system. Laser and powder are co-focussed at the substrate with laser having a top-hat intensity distribution and incoming powder having Gaussian distribution at the co-focussed point.

For all experiment cases, the depositions are performed at a scan speed of 1 m/min with zig-zag deposition strategy and a powder deposition rate of 13 g/min Stainless Steel 316 L powder feedstock (Oerlikon, grain size 45-106  $\mu\text{m}$ ). The laser beam spot size was measured to be 2.2 mm in diameter at the part surface. **Figure 2A** shows the schematic of substrate and planned wall build and **Figure 2B** shows DED system with nozzle **Figure 2C**, co-axial infra-red camera **Figure 2D** and fixture clamps and **Figure 2E** shows the wall build for case 1. Each wall build is 20 layers high, 1 bead wide, with a longitudinal zig-zag deposition strategy. For the first set of experiments, effect of waiting time between successive layers (dwell time) is studied by fixing laser

**TABLE 2 |** Description of the cases and process parameters used in the present work.

### Experiment Set 1: Effect of dwell time (scan speed 1 m/min)

Case	Dwell time (s)	Laser Power (W)
D1	0	800
D2	5	800
D3	10	800
D4	30	800

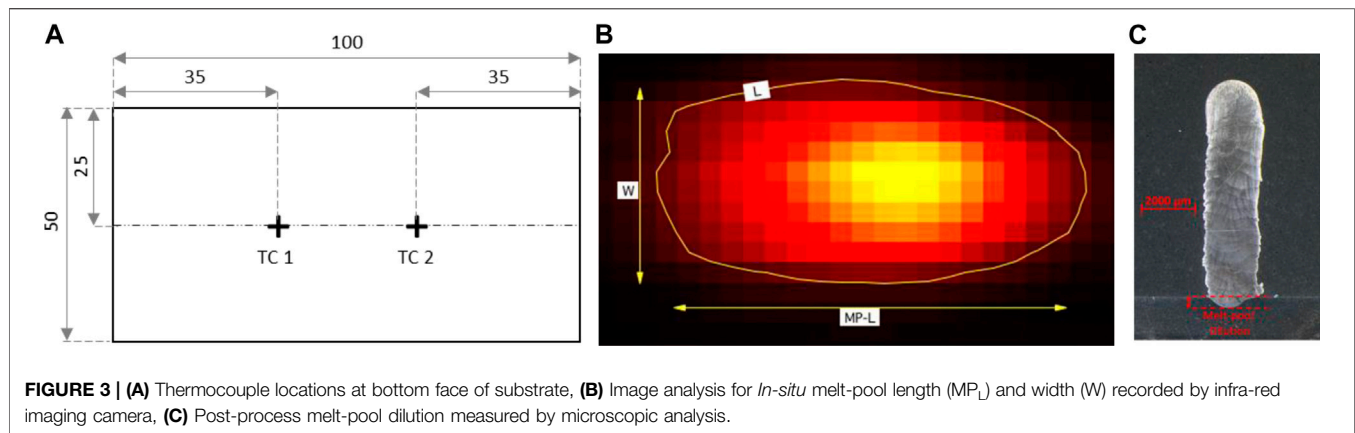
### Experiment Set 2: Effect of Laser Power (scan speed 1 m/min)

Case	Dwell time (s)	Laser Power (W)
P1	10	800
P2	10	1000
P3	10	1200
P4	10	1400

power 800 W and changing the dwell time after the deposition of each layer. Dwell times of 0, 5, 10 and 30 s were used for each layer to expose the parts to different cooling time. In the second set of experiments, effect of laser power is studied by keeping a dwell time of 10 s and varying the laser power to 800, 1000, 1200 and 1400 W. Laser scan speed is set to 1 m/min in both tests. **Table 2** summarizes the sets of experiments and cases.

## Temperature Measurement

*In situ* temperature is measured at two different locations on the bottom face of substrate, as shown in **Figure 3A**, using  $\Omega$  GG-K-30 type K thermocouples of 250  $\mu\text{m}$  diameter. The thermocouples have a measurement uncertainty of  $\pm 0.75\%$ . TC 1 and TC 2 are located on the bottom surface of the substrate along the deposition path. The thermocouple signals are read by National Instruments modules 9213. The module record data in Signal Express at a sampling rate of 200 Hz.

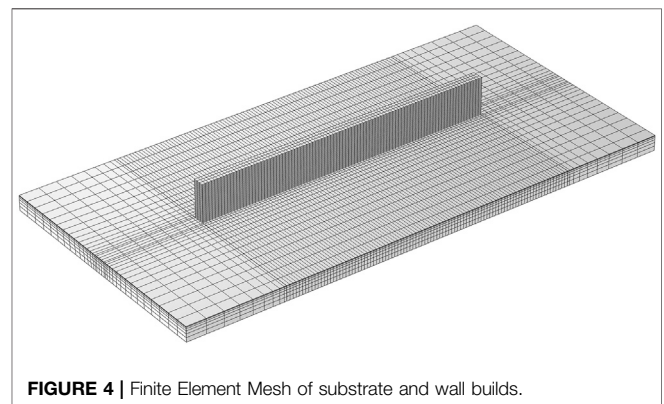


**TABLE 3 |** Wall geometries and Melt-Pool Dimensions.

Case	Track Width $W$ (mm)	Track Height $H$ (mm)	Melt-Pool Dilution $MP_D$ (mm)	Melt-Pool Length $MP_L$ (mm)
D1-D4	2.20	0.38	0.20	3.50
P1	2.20	0.38	0.20	3.50
P2	2.25	0.40	0.40	3.65
P3	2.40	0.41	0.52	3.80
P4	2.50	0.43	0.70	3.85

## Melt-Pool Measurement

An Infra-Red imaging camera (NIT system) is used to analyze the melt-pool stability and to estimate the melt-pool dimensions based on image analysis. Image acquisition was done at a sampling rate of 200 fps. Image analysis is performed with the Igor Pro software on these images and an average value of melt-pool width ( $W$ ) and length ( $MP_L$ ) is measured for each experiment case based on a specific contour level ( $L$ ), as shown in **Figure 3B**. The laser penetration depth was also measured for each case by sectioning as recommended in (Goldak et al., 1984) and illustrated in the macrograph shown in **Figure 3C**. The melt-pool dimensions values given in the **Table 3** are then used to calibrate the values of heat source as discussed in previous section.



## NUMERICAL IMPLEMENTATION

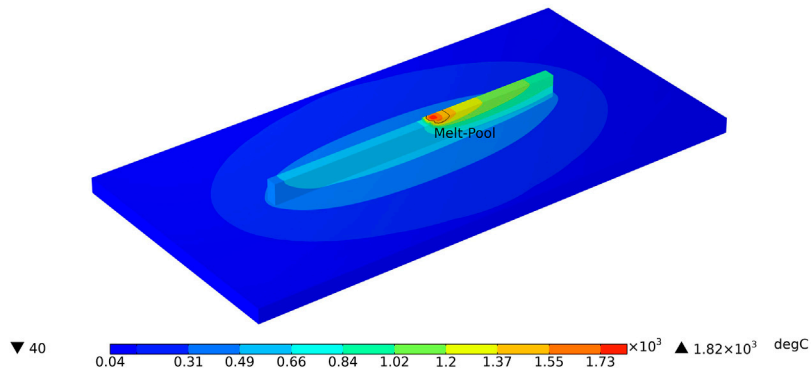
### FEA Solver

The FEM analysis is performed using COMSOL Multiphysics based solver (PARDISO) with the implicit Backward Differentiation Formula (BDF) time stepping method. Adaptive time stepping method is employed rather than strict formulation with maximum time step of  $R/v_s$  for DE and  $\Delta t$  for EE heat sources. The solver is further adapted specifically to model additive manufacturing technologies in the present work. All simulation cases are performed on an Intel Xeon W-2275, 16 cores, with 128 GB RAM workstation.

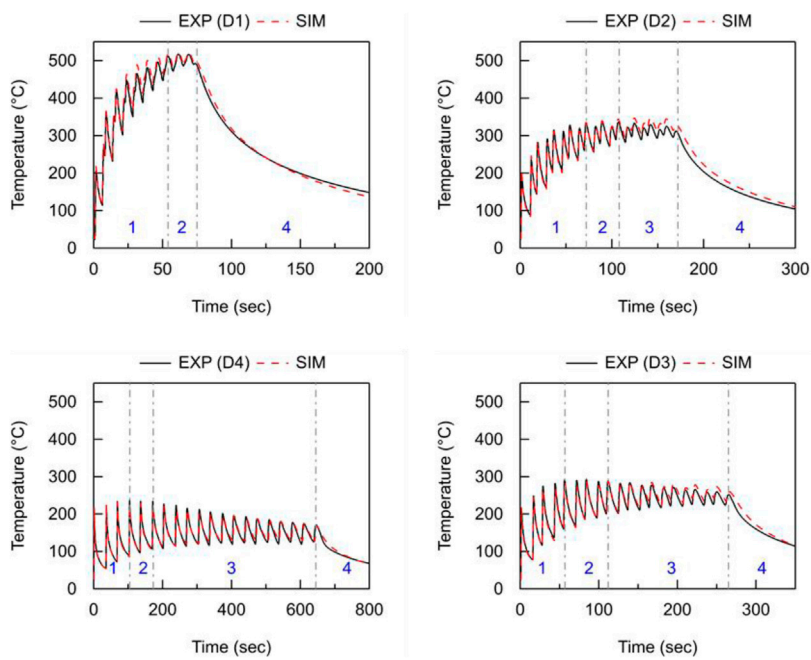
### FEM Mesh

**Figure 4** displays the three-dimensional finite element mesh, generated in COMSOL Multiphysics, used for the thermal model.

The same mesh is used for all cases and for different Heat Source models used in the present work. The mesh contains 33,852 Hex-8 elements and 49,790 nodes. Hex-8 elements were chosen because it has been proved that they give more accurate results as compared to tetrahedral elements for the plastic deformation (Benzley et al., 1995), therefore to have same mesh for thermal and mechanical analysis in future work, Hex-8 elements were used. The elements for the deposited material are allotted as 3 per laser spot size and 2 per deposition thickness, making the elements  $0.75 \times 0.75 \times 0.19$  mm in volume for experiment case D1, but varies for different experiment cases as track geometry changes. The mesh is coarsened at the substrate as it moves away from the wall builds. A mesh convergence study was done using three different mesh strategies to confirm the accuracy of thermal analysis.



**FIGURE 5** | Temperature distribution during deposition of 12th layer utilising Quiet/Active material activation.



**FIGURE 6** | Experiments v/s Simulation using Double Ellipsoid (DE) heat source for experiment cases D1-D4 (Effect of dwell time).

### Material Deposition Modeling

The “quiet” element activation method is used to simulate the deposition of material during the DED process. The elements that represent the wall builds are pre-existing at the beginning of the analysis as shown in **Figure 4**. But their material properties are rescaled in such a way that it does not affect the computational analysis. In the present work, the thermal conductivity  $k$  and specific heat  $C_p$  are set to very low values to minimize the heat transfer from active to quiet elements, as follows:

$$k_{quiet} = \alpha_k k \tag{13}$$

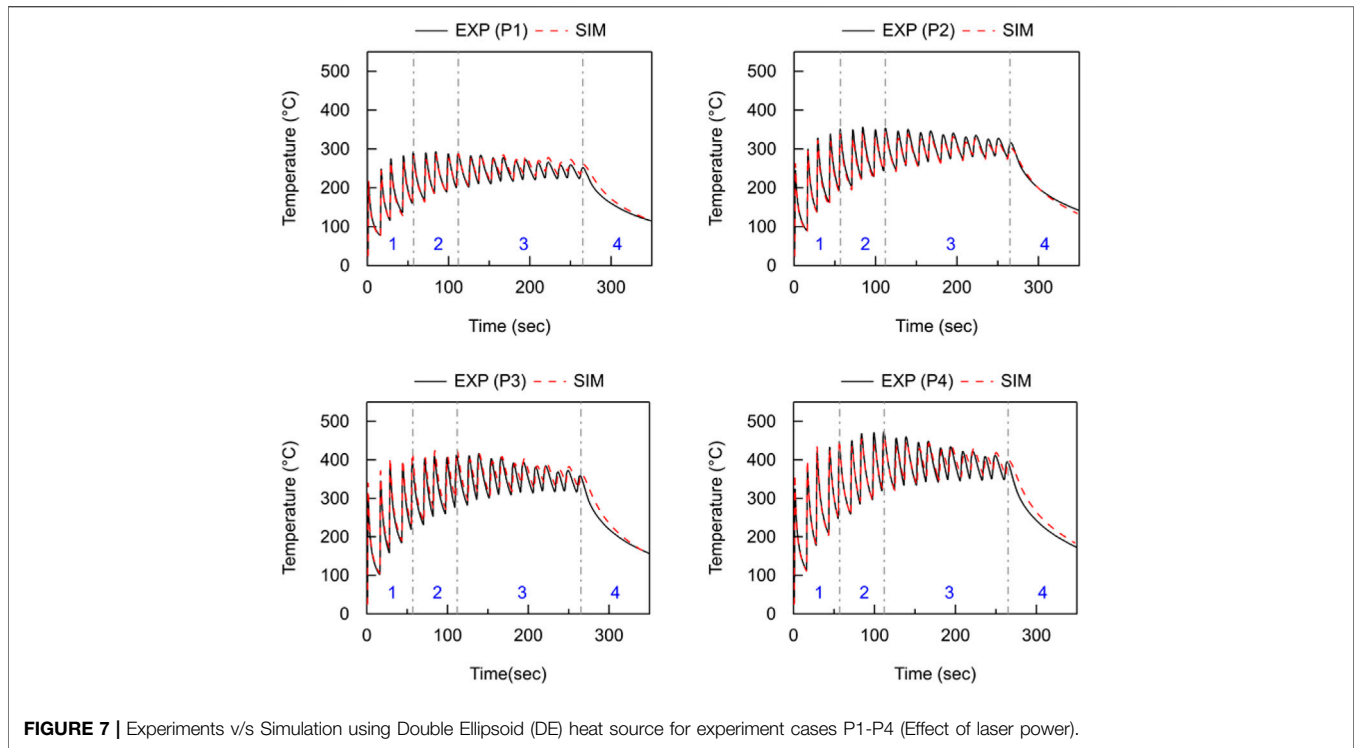
$$C_{p_{quiet}} = \alpha_{C_p} C_p \tag{14}$$

where  $k_{quiet}$  and  $C_{p_{quiet}}$  are the respective thermal conductivity and specific heat for the quiet elements;  $\alpha_k$  and  $\alpha_{C_p}$  are the appropriate

**TABLE 4** | Classification of layers deposition process according to temperature gradient for experiment cases D1-D4 (Effect of dwell time).

Experiment Case	Number of layers		
	Zone 1	Zone 2	Zone 3
D1	18	2	0
D2	9	4	7
D3	5	4	11
D4	4	2	14

scaling factors that are usually assigned very small values, for instance,  $10^{-4}$  is employed in the present work. The properties of an element are switched from “quiet” to “active” when any Gauss



**FIGURE 7 |** Experiments v/s Simulation using Double Ellipsoid (DE) heat source for experiment cases P1-P4 (Effect of laser power).

**TABLE 5 |** Classification of layers deposition process according to temperature gradient for experiment cases P1-P4 (Effect of laser power).

Experiment case	Number of layers		
	Zone 1	Zone 2	Zone 3
P1	5	4	11
P2	5	4	11
P3	5	4	11
P4	5	4	11

point of the element is consumed by the heat source volume (Eq. 15).

$$\exp\left(-\frac{3(x + v_s t)^2}{a_{f,r}^2} - \frac{3y^2}{b^2} - \frac{3z^2}{c^2}\right) \geq 5\% \quad (15)$$

This means that if the evaluated heat source value at any Gauss point is greater than 5% of the peak intensity, the element will be activated by switching material from quiet to active.

For the elongated ellipsoid line heat input model, material activation is done in the same way of heat source intensity as explained above, but in this case only the activation domain is larger over the deposition path. If the evaluated heat source value at any Gauss point is greater than 50% of the peak intensity, the element turned to active state. The activation criteria of 50% in the deposition direction is explained in the further sections.

$$\exp\left(-\frac{3\left(x + v_s\left(t_0 + \frac{1}{2}\Delta t\right)\right)^2}{\hat{a}^2}\right) \geq 50\% \quad (16)$$

### Model Calibration and Boundary Conditions

To simplify the model, deposited wall build is considered to be a flat rectangular shape with constant layer height and width. This is contrary to the experiments as layer height changes in the few first layers and then reaches a uniform layer thickness. As discussed previously, to develop an accurate thermal model, certain input parameters have to be calibrated against experiment results. The laser absorption efficiency ( $A$ ) is taken as 0.4 with reverse calibration of fitting simulated temperature field to experiment results at thermocouple location iteratively as suggested in (Michaleris, 2014). Also, for all experiment cases, as laser spot size is 2.2 mm, laser spot radius is taken as 1.1 mm. The double ellipsoid heat source dimensions parameters are dependent on experiment cases but follow the same rule i.e., front ellipsoid length  $a_f = W/2$  and  $a_r = 2a_f$ ,  $b = W/2$  and  $c = H + MP_D$ , that represent the double ellipsoid dimensions to be almost equal to experiment melt-pool dimensions for each experiment case. Emissivity ( $\epsilon$ ) is temperature dependent, but mean value of 0.6 is taken as widely reported in the literature (Biegler et al., 2018b). The convective heat transfer ( $h$ ) coefficient is temperature dependent, but a mean value is taken at the substrate  $h = 5 (W.m^{-2}.K^{-1})$  and to consider the effect of forced convection due to the powder carrying argon gas at the wall builds, average convective heat transfer coefficient of  $h = 25 (W.m^{-2}.K^{-1})$  is taken as widely reported in the literature that it can be in the range of  $h = 20 - 60 (W.m^{-2}.K^{-1})$  (Heigel et al., 2015). At the clamped surfaces, the substrate is in contact with metallic fixtures clamps shown in Figure 1 (d). In the numerical model, to reduce the computation time, fixtures clamp is not included, therefore for these surfaces, a higher heat loss is modelled through conductive to convective equivalent heat loss expression similar to Newton’s law



**TABLE 6** | Cases examined for Thermal Model Validation (Double Ellipsoid).

case	Run Time (min)	% Error	
		TC 1	TC 2
1	98	2.49	1.55
2	135	3.41	3.37
3	175	3.09	3.41
4	324	3.57	3.97
5	175	3.09	3.41
6	177	3.16	2.42
7	180	3.93	2.57
8	182	2.81	2.53

**TABLE 7** | Elongated Ellipsoid (EE) heat source parameters used in the present work.

$K_E$	Computation time step FEM ( $\Delta t$ ) (s)	EE length ( $\hat{a}$ ) (mm)	No. of sub-tracks per layer (Wall Length/ $\hat{a}$ )
0.5	0.066	1.142	52
1	0.132	2.288	26
4	0.538	9.1538	8
8	1.057	18.308	4
27	3.564	61.788	1

of cooling Eq. 10. The thermal conductance at the contact surfaces is thus  $h = 150 (W.m^{-2}.K^{-1})$  to model the conductive heat transfer from substrate to the metallic fixture clamps as suggested in (Lu et al., 2019).

## MODELING RESULTS AND DISCUSSION

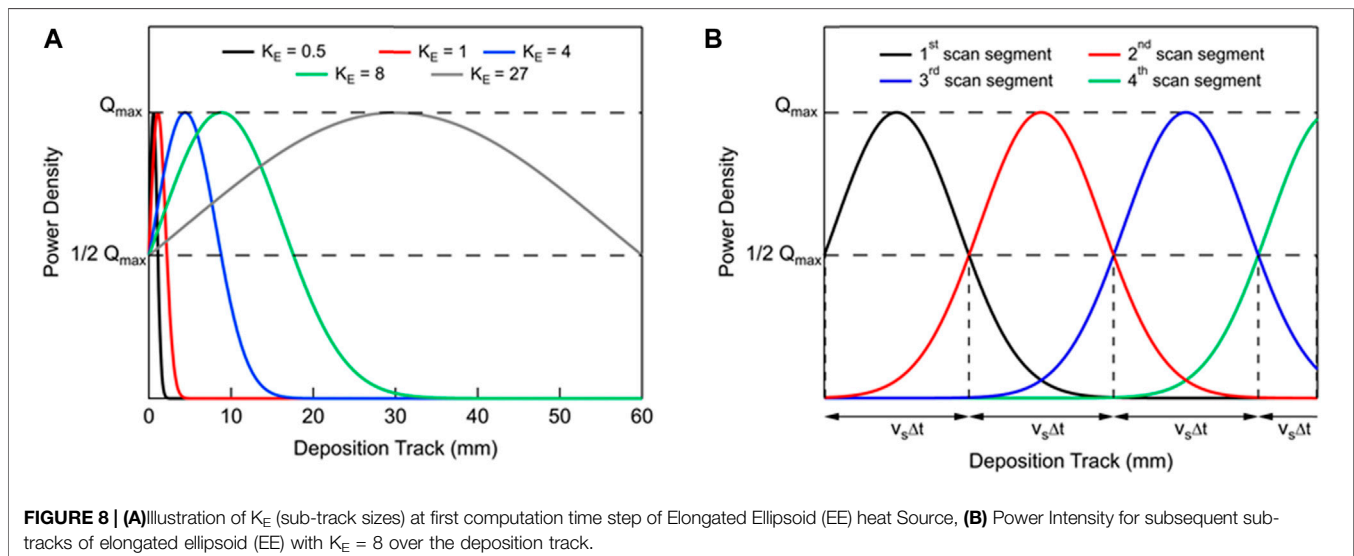
### Model with Double Ellipsoid Heat Source

The Quiet/Active material activation is well implemented in the model as can be seen in Figure 5. Indeed, during the deposition of

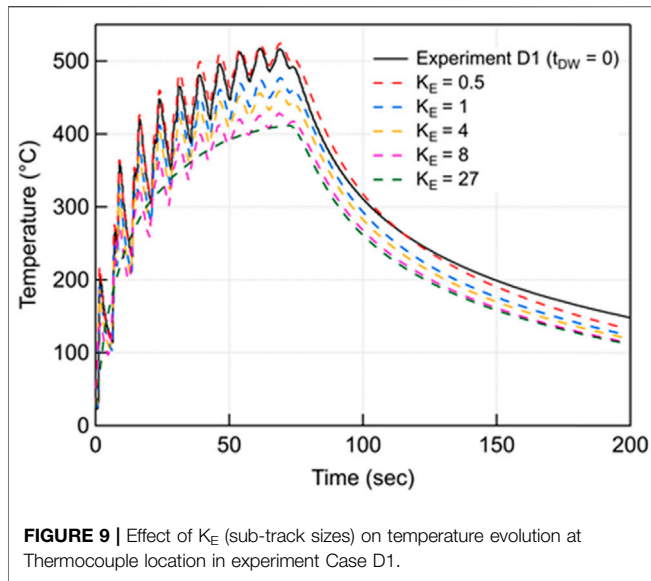
the 12th layer, quiet elements of the layer and elements above are not activated and hence does not contribute to the heat transfer, as the material properties of quiet elements are assigned to a dummy material. The current deposition elements and previously deposited layer elements are correctly activated from quiet to active once the activation criteria are satisfied that depends upon the laser travel (Eq. 15). The input heat source absorptivity and heat losses parameters are kept same for all experiment cases as discussed in previous section Model Calibration. Double Ellipsoid heat source dimensions are dependent on experiment cases and is taken following the rule explained in the previous section Model Calibration. The thermal response of the workpiece is calculated by the Double Ellipsoid model and compared to the experimental measurements.

Figure 6 shows the comparison between experimental results, as measured by thermocouples 1 and 2, and the numerical results at the corresponding nodes in FEM analysis for experiment set 1 (effect of dwell time). As explained in the previous section, due to the experiment set-up, because thermocouple 1 and 2 are at different locations on the substrate but along the deposition line, they record almost same thermal histories with a time offset. Therefore, only Thermocouple 1 results are presented. The thermal response can be classified in 4 different zones depending upon the thermal history. In zone 1, peak temperature for successive build layers increases as layers are building up, in zone 2, peak temperature for successive layers stabilises and there is no further increase of temperature, and in zone 3, peak temperature for successive build layers starts to decrease due to the fact that heat source is moving vertically away from the thermocouple locations and fabricated material starts to increase that also conducts heat leading to the less heat transfer or temperature gradient. In zone 4, once the process is finished, temperature rapidly reduces with respect to time.

In the experiment set 1, analysis of effect of dwell time is studied, as shown in Figure 6. Peak temperature starts to decrease as dwell time increases from 0 to 30 s. Longer dwell times results



**FIGURE 8** | (A) Illustration of  $K_E$  (sub-track sizes) at first computation time step of Elongated Ellipsoid (EE) heat Source, (B) Power Intensity for subsequent sub-tracks of elongated ellipsoid (EE) with  $K_E = 8$  over the deposition track.



**FIGURE 9** | Effect of  $K_E$  (sub-track sizes) on temperature evolution at Thermocouple location in experiment Case D1.

in lower peak temperatures of 230°C with the 30 s dwell time (D4), while with 0 s dwell time (D1) exceeding 500°C. Therefore, dwell time has a strong influence on temperature evolution as well as the peak temperatures obtained during the deposition process.

For experiment set 1, analysis of effect of dwell time, it is observed that thermal response is changing drastically, not just in terms of peak temperatures, but also in terms of thermal evolution. **Table 4** that counts the number of peaks (or layers) observed in the different zones (Z1 to Z4), the number of layers under Zone 1 are decreasing with an increase of dwell time, depicting peak temperature is stabilising at much earlier deposition stage (number of layers). But, the number of layers under Zone 2 and 3 are increasing with an increase of dwell time, depicting that stabilised peak temperature are maintained for longer duration of deposition. Numerical model with calibrated parameters captures the temperature evolution trend and peak temperatures correctly with change of dwell time.

In experiment set 2, analysis of effect of laser power is studied, as shown in **Figure 7**. Peak temperature keeps on increasing as laser power increases from 800 to 1400 W. higher laser power results in higher peak temperatures of 480°C with 1400 W laser power (P4), and only 280°C with 800 W laser power (P1). Therefore, laser power has a strong influence on peak temperature, but does not influence the trend of temperature evolution obtained during the deposition process.

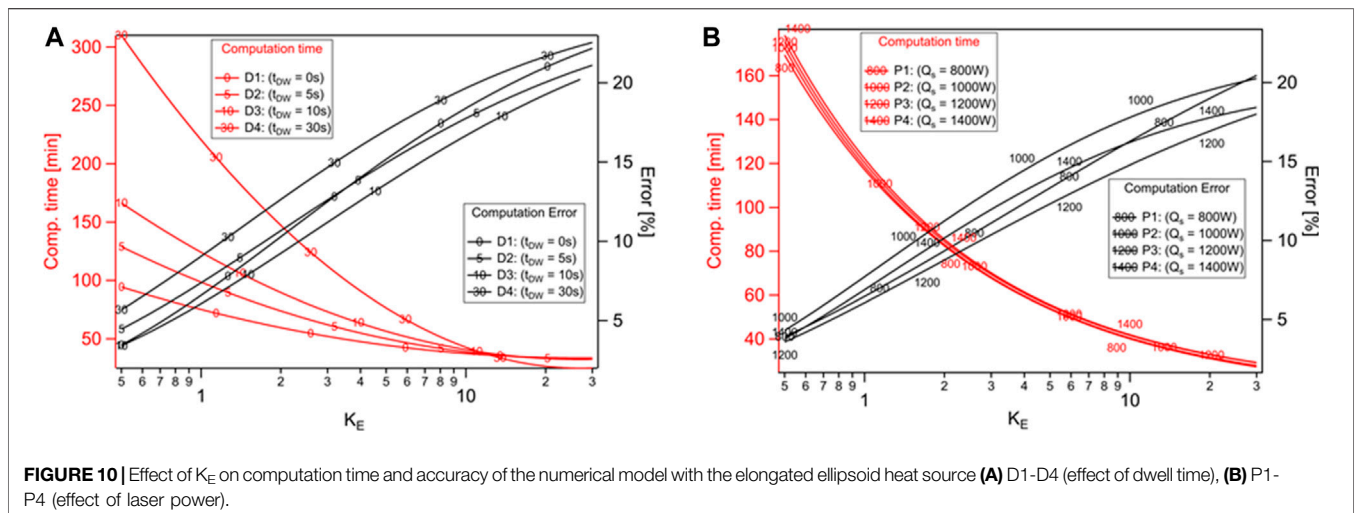
For experiment set 2 (analysis of effect of laser power), it is observed that thermal response is changing peak temperatures drastically, but does not influence the thermal gradient. As it can be seen in **Table 5**, number of layers under Zone 1, 2 and 3 remains same depicting that temperature trend is lifted upwards (increase of peak temperature). Higher peak temperatures are recorded because of the increase of laser power, but the thermal gradient remains the same. As can be seen in **Table 5**, peak temperature is stabilised at fifth layer, stabilises for next 4 layers and then starts to decrease as heat source is moving vertically away from thermocouple locations for all experiment cases (P1-P4).

The results of the transient thermal analyses are in close agreement with the experimental results as can be seen in **Figure 6** and **Figure 7**. Errors between experiment and simulation results are calculated by comparing instances in time.

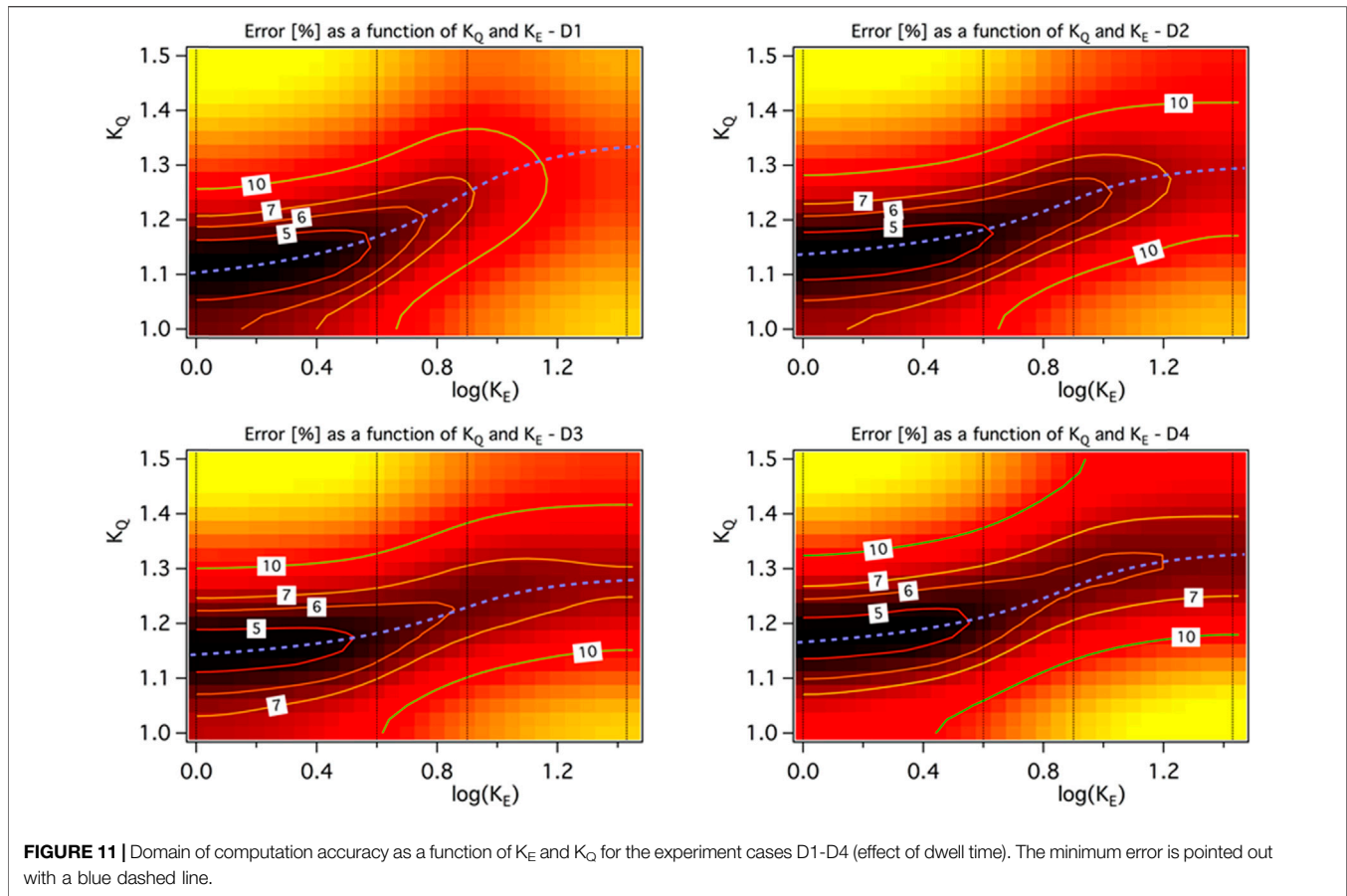
$$\% \text{ Error} = \frac{100 \sum_{i=1}^n \left| \frac{(T_{exp})_i - (T_{sim})_i}{(T_{exp})_i} \right|}{n} \quad (17)$$

where  $n$  is the total number of simulation time increments between the beginning and end of the deposition,  $i$  is the current time increment,  $T_{sim}$  is the simulated temperature, and  $T_{exp}$  is the measured temperature. The largest error at thermocouple is found to be 3.91%.

**Table 6** shows the computation time and percent error at both thermocouples TC1 and TC2 for all cases. For experiment set 1 (D1 to D4), it can be clearly seen that with the increase of dwell time, that leads to increase in number of time increments leading to increase in computation time. For experiment set 2 (P1 to P4), it can be observed



**FIGURE 10** | Effect of  $K_E$  on computation time and accuracy of the numerical model with the elongated ellipsoid heat source **(A)** D1-D4 (effect of dwell time), **(B)** P1-P4 (effect of laser power).



that with an increase in laser power, that leads to an increase in simulation peak temperature at the melt-pool region i.e., higher thermal gradient, that leads to slight increase of computation time.

### Model with Elongated Ellipsoid Heat Source

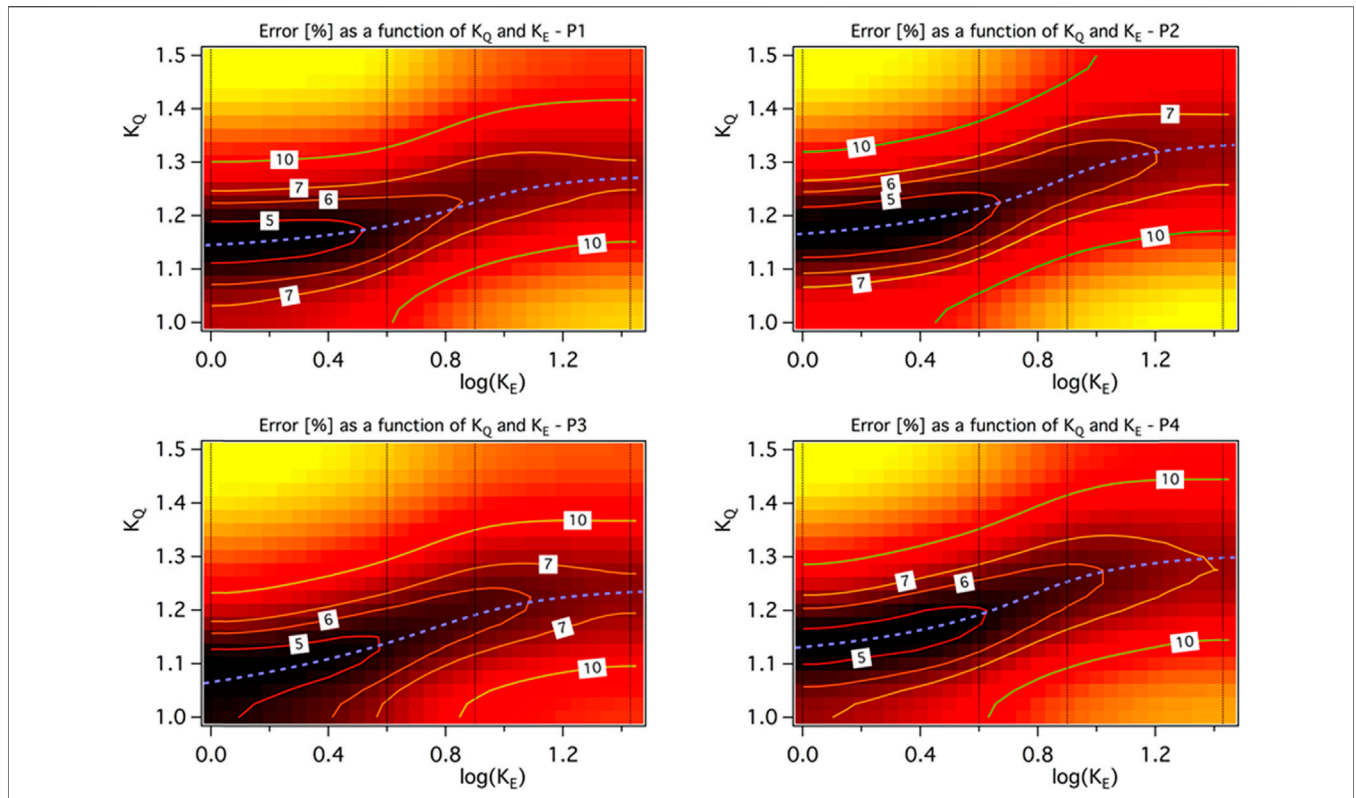
To reduce the computation time, elongated ellipsoid model is used to reduce the number of simulation time steps by dividing the complete track in number of linear sub-tracks, with each sub-track is solved in one simulation time step. Different track size (sub-track) is chosen for material activation presented in **Table 7** and a comparison is done with the experiment results to make a compromise between computation speed and accuracy.

$$K_E = \frac{v_s \Delta t}{a} \quad (18)$$

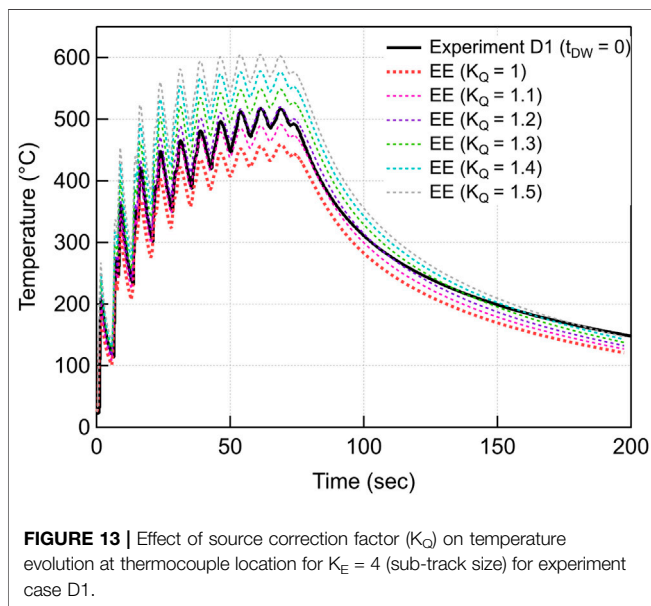
As shown in **Figure 8A**, 5 different simulation time increments are chosen, where  $K_E = 0.5$  represents heat source movement of  $R$  with a time step of  $R/v_s$ , hence the ellipsoid is not elongated and that is same as in the case of double ellipsoid heat source,  $K_E = 4$  represents heat source averaging over a segment of  $8R$  with a simulation time step of  $8R/v_s$  and  $K_E = 27$  represents heat source averaging over a segment of  $54R$  with a simulation time step of  $54R/v_s$ , that is averaging over the complete track, activating the complete track and performing heat transfer analysis in 1 simulation time increment.

The movement of the elongated ellipsoid heat source is such that there is a smooth distribution of power intensity over the successive scan segments. The power intensity at the start and end of each scan segment is half of its peak value, resulting in a smooth distribution over the successive segments of a track as shown in **Figure 8B**. And this is the reason why activation criteria for elongated ellipsoid (EE) heat source model is 50% that accounts for the half of its peak value in the deposition direction. Of course, increasing the factor  $K_E$  helps in reducing the total computation time, but it also leads to the increase of computation error. Thus, effect of sub-track size activation on temperature evolution at thermocouple location is shown in **Figure 9**.  $K_E$  of 0.5 ( $\Delta t = R/v_s$ ) gives the same accurate results as we observed with Double Ellipsoid (DE) heat source with a simulation time step of  $R/v_s$ . As explained in the section further, computation error increase with  $K_E$ , while computation time drops, as shown in **Figure 10**.

For all experiment cases, simulation is performed to find out the effect of  $K_E$  on computation time and accuracy. **Figure 10A** shows the results for D1-D4 (effect of dwell time). For case D1, with increase in  $K_E$  from 0.5 to 27, computation time reduces drastically from 90 to 35 min, but leads to an increase of error from 2 to 22%. For case D2, with respect to increase of  $K_E$ , computation time reduction is more drastic from 130 to 36 min and computation error increases



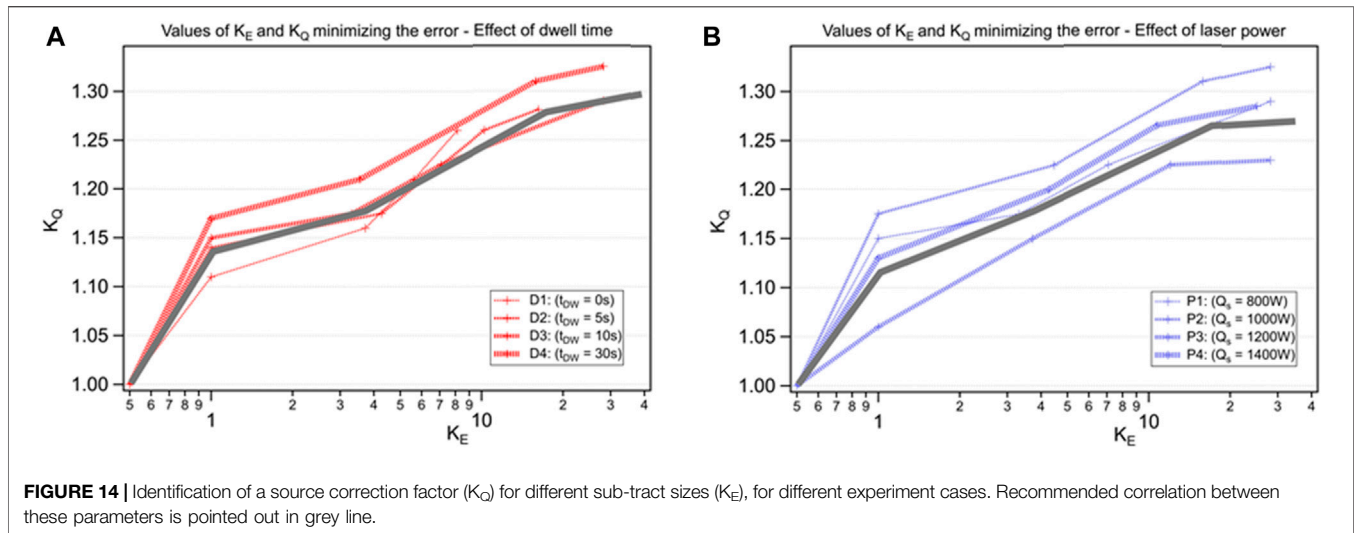
**FIGURE 12 |** Domain of computation accuracy as a function of  $K_E$  and  $K_Q$  for the experiment cases P1-P4 (effect of laser power). The minimum error is pointed out with a blue dashed line.



**FIGURE 13 |** Effect of source correction factor ( $K_Q$ ) on temperature evolution at thermocouple location for  $K_E = 4$  (sub-track size) for experiment case D1.

from 3 to 21%. For case D3 and case D4 as well, computation time reduction is more as dwell time is increasing but computation error is also increasing.

**Figure 10B** shows the results for all cases P1-P4 (effect of laser power). For case P1-P4, with increase in  $K_E$  from 0.5 to 27, computation time reduces from 175 to 25 min, but leads to an increase of error from 2 to 21%. Computation time reduction is exponential (3–4 times reduction) when  $K_E$  is increased from 0.5 to 4. But then the trend in computation time becomes linear when  $K_E$  is increased from 4 to 27 (not even half). Computation error increases linearly with an increase in  $K_E$ . So, an intelligent compromise needs to be done to reduce the computation time but also keeping in mind the computation error as well. In this work, the objective is that computational error should be less than 10%, that is well accepted in the scientific and industrial community. Keeping this in mind,  $K_E = 4$  seems to satisfy both the objectives of reducing the computation time drastically but also keeping computation error less than 10%. Further increase of  $K_E$  results in computation time reduction, but computation error exceeds 20% when  $K_E = 27$ . However, with the introduction of elongated ellipsoid (EE) heat source, it is noticed that there is a drop-down of the temperature at melt-pool scale (local scale) as well as part scale (global scale) e.g., thermocouple locations that is far away from the deposition region as shown in **Figure 9**. This modification of the thermal behaviour (especially, melt-pool scale) will surely impact the local mechanical response of the sample. Therefore, same analysis needs to be performed to verify if the optimised value of  $K_E$  found in thermal analysis is also



valid for mechanical response e.g., distortion and residual stresses.

## Correction Factor

Due to the increase in  $K_E$  (sub-track length), computation time reduces exponentially, but it also leads to increase in computation errors. To compensate those errors, a variable source correction factor  $K_Q$  is introduced that leads to an increase in source intensity in elongated ellipsoid model, as shown in the equation below.

$$Q_{EE} = \frac{6\sqrt{3}APK_Q}{abc\pi\sqrt{\pi}} \exp\left(-\frac{3\left(x + v_s\left(t_0 + \frac{1}{2}\Delta t\right)\right)^2}{\hat{a}^2} - \frac{3y^2}{b^2} - \frac{3z^2}{c^2}\right) \quad (19)$$

With the introduction of  $K_Q$ , distribution and peak intensity over the ellipsoid is increased artificially. Thus, the correct value of  $K_Q$  should be dependent upon the value of  $K_E$ , so calibration of source correction factor is done with respect to different sub-track size for all experiment cases. Then for the correct value of correction factor that should be dependent upon the size of sub-track ( $K_E$ ), Calibration of correction factor is done with respect to sub-track size ( $K_E$ ) for all cases.  $K_E = 0.5$  does not elongate the ellipsoid. So, to analyse the effect of correction factor, values of  $K_E$  starting from 1 to 27 is used, because  $K_E = 0.5$  using Elongated Ellipsoid does averaging over a sub-track of  $R$ , i.e., SE or DE with a simulation time step of  $R/v_s$  i.e., conventional simulation increment.

As it can be seen in **Figure 11** with  $K_E$  values (1, 4, 8 and 27) shown in black vertical lines, for experiment cases D1-D4 (effect of dwell time), with an increase in  $K_E$ , computation error increases as we have seen in the previous section, source correction factor  $K_Q$  value is also increasing to compensate for this increase in computation error with an increase in  $K_E$ . Therefore, there is a direct relation between  $K_E$  and  $K_Q$ , that is helpful in achieving the second objective to reduce the computation error. With the introduction of  $K_Q$ ,  $K_E = 8, 16$  can also be utilised as computation error falls below 10% which was not the case previously. As it can be seen in **Figure 12**, for experiment cases

P1-P4 (effect of laser power), same relation is established between  $K_E$  and  $K_Q$ . In this set of experiments also,  $K_E = 8, 16$  can now be used in heat transfer analysis as computation error falls below 10%. So, with the introduction of  $K_Q$ , computation error is reduced along with reduction of computation time as higher values of  $K_E$  yields desired accurate results. With the introduction of correction factor, there is no significant impact on the computation time of analysis.

As it can be seen in **Figure 13**, for experiment case D1 with  $K_E = 4$ , with the introduction of correction factor  $K_Q$  using EE heat source model, temperature evolution at thermocouple location shifts upwardly due to the increase of heat intensity.  $K_Q$  values of 1.1 and 1.2 seems to be well compensating the effect of  $K_E$  that leads to underprediction of temperature evolution, but with further increase of  $K_Q$  values of 1.3, 1.4 and 1.5, it can be seen that it over-compensates the thermal error generated due to  $K_E$ , and starts to yield higher computation error. Therefore, a correlation between  $K_E$  and  $K_Q$  should be established for different process parameters that yields minimum computation error for elongated ellipsoid (EE) heat source.

## Correlation

For all experiment cases D1-D4 and P1-P4, it can be seen in **Figure 14** that for most experiment cases, with an increase of  $K_E$  from 0.5 to 8, there is also an increase in correction factor from  $K_Q = 1$  to 1.25, that leads to minimum computation error for temperature history at thermocouple location. But with further increase of  $K_E$  from 8 to 27, further increase in values of  $K_Q$  is not required. Therefore, a correlation between  $K_E$  and  $K_Q$  is recommended in this work that yield to minimum computation error.

## CONCLUSION

- DE heat source model with small time increments ( $\leq R/v_s$ ) correctly predicts temperature evolution with an average computation accuracy of more than 96%, but leads to long

computation time due to thousands of simulation time increments.

- EE heat source model with different  $K_E$  values significantly reduces computation times (up to 10 times) while yielding average computation accuracy of more than 75–95%.
- A source correction factor ( $K_Q$ ) for Elongated Ellipsoid (EE) heat source model is presented that reduces the average computation error within 5% at thermocouple location.
- A correlation is then found that is helpful in finding the correlation between simulation time increments, computation time and error. Correlation is shown to be dependent of variables laser power and dwell time.
- Thermal model with DE heat source and EE heat source with different  $K_E$  and  $K_Q$  works efficiently for different process parameters justifying the versatility of the model.

In the future, correlation can also be extended to other materials and process parameters and can be applied to a thermo-mechanical model to prove the efficiency of the Elongated Ellipsoid heat source model in the prediction of residual stress and deformation as well.

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## DATA AVAILABILITY STATEMENT

The original contributions presented in the study are included in the article/Supplementary Material, further inquiries can be directed to the corresponding author.

## AUTHOR CONTRIBUTIONS

VN, MC, and TE contributed to the modeling of the study. VN, LS, and DB organized and performed the experiments. VN, LS, and TE did the data treatment and analysis. VN wrote the first draft of the manuscript. VN, MC, TE, and DB contributed to manuscript revision, read, and approved the submitted version.

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