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How does the space of production wells influence the heat extraction efficiency?-defined by a 3D modeling work in enhanced geothermal system (EGS)

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The space of production well plays a crucial role in the heat extraction performance of the Enhanced Geothermal System (EGS), which have the potential to meet the growing global energy demand as a reliable energy source. Nonetheless, there has been insufficient research and attention focused on comprehending the impact of the space of production well on the efficiency and effectiveness of EGS. In this work, a series of numerical simulations were conducted to assess the impact of the space of production well on heat extraction efficiency in EGS. Three different cases were considered: Case 50 with 50 m production well spacing, Case 100 with 100 m production well spacing, and Case 150 with 150 m production well spacing. At the X-Y plane and Y-Z plane, the simulation results indicated that there were slight differences in temperature variation among the Case 50, Case 100 and Case 150. And the cooling area decreasing as the spacing of production wells decreased. Moreover, the delivery of cooling water via the injection well and its subsequent distribution to various reference points lead to a decline in temperature at each point, albeit with varying degrees of variation. Besides, the established efficiency (ef) for Case 50 is smaller than the Case 100 and Case 150 during the last 15 years. These findings contribute valuable insights to the exploration and exploitation of EGS systems and can serve as a guide for further research in this field.

KEYWORDS

space of production well, heat extraction efficiency, enhanced geothermal system (EGS), numerical simulation, engineering scale

1 Introduction

Nowadays, as a matter of fact, energy consumption has recently increased significantly due to the world economy's rapid growth, which is also causing the intended low-carbon and green process—that is, a high-speed and green development—to go more quickly (Olasolo et al., 2016; Liu et al., 2017a; Liu et al., 2017b; Zheng et al., 2019; Hao et al., 2021;

Lin et al., 2021; Zhao et al., 2022). With this context in mind, unconventional oil/gas resources like tight sandstone gas and shale oil, as well as renewable energy sources like solar, wind, and enhanced geothermal systems (EGS), emerge on the scene (Zheng et al., 2018; Kumari and Ranjith, 2019; Cheng et al., 2021; Steffen et al., 2021). Therein, the EGS is thought to have the capacity to supply the growing energy needs for the reason that it has an endless supply of resources that can be obtained for less money than traditional fuels from almost any place in the globe (Lu, 2018; Abdelhafiz et al., 2023; Zhao et al., 2023). Undoubtedly, a highly effective development pertaining to this type of renewable energy source is required to support this issue. Due to its emerging nature and potential to generate clean, low-carbon energy, EGS is receiving a lot of attention. There are also efforts underway to steer EGS toward a commercially viable platform through the use of costcutting measures, enhanced performance, and technology validation (Lu, 2018). As a result, many initiatives were carried out and demonstrated successful and acknowledged some accomplishments (Olasolo et al., 2016).

Actually, EGS is now commonly known as an engineered geothermal system, which has replaced the previous phrases used in earlier research, such as hot dry rock and hot sedimentary aquifers (Christ et al., 2017). Rudimentary studies on the creation of artificial geothermal reservoirs, and heat exchange and transport were arranged in relation to the EGS-related investigations (Feng et al., 2012; Zhang and Jiang, 2012; Li and Lior, 2014). Additionally, the Songliao Basin in Northeast China was the subject of a study on the 30-year heat extraction process in an EGS system, which examined the most significant variable factors involved (Huang et al., 2015). Besides, predictions were made regarding the diverse effects on EGS results during extended periods of operation under varying geological situations (Chen et al., 2013a; Chen et al., 2013b). Moreover, Gan et al. (2021) and Spycher and Pruess (2010) conducted research on an EGS system that employed CO₂ as the working fluid instead of water. Furthermore, based on the utilization of fracture network simulation techniques for an EGS system, the study also analyzed the hydraulic fracturing process (Wang and Zhang, 2011). While these mentioned literature sources may have different focuses, they share a common aspect, which is the utilization of numerical modeling. Furthermore, upon reviewing past accomplishments in relation to EGS systems, it is apparent that the impact of the space of production well on heat extraction efficiency has not been adequately addressed. This lack of consideration may restrict the optimal positioning of production wells for heat extraction.

Indeed, numerical approaches have become extensively utilized in research on geological resources in recent years, particularly in endeavors aiming to simulate complex engineering scenarios that are impractical to conduct in a traditional experimental setup (Sun et al., 2013; Chen et al., 2019; Li and Elsworth, 2019; Liu et al., 2021; Yang et al., 2023). This study employs numerical modeling to simulate the process of heat extraction from an EGR system. Herein, the production well spacing is varied to assess its impact on heat extraction efficiency. Moreover, the study also contrasts the heat extraction effectiveness under various operational circumstances to conduct a quantitative analysis of how the space



of production well affects heat extraction efficiency from an EGS system. This numerical study is carried out on an engineering scale about how the well space influence the heat extraction performance, providing a new viewpoint and possibly providing some guidance for the exploration and exploitation of EGS-related issues.

2 Numerical model description

For this numerical study conducted at an engineering scale, a hot dry rock (HDR) cubic model with edge length of 400 m was employed, which is a further work on the basis of our previous investigations (Wang et al., 2016a; Wang et al., 2016b; Hu et al., 2022; Ke et al., 2022; Wang et al., 2022; Wang et al., 2023). Within this model, the EGS system was situated at its center with dimensions of 250 m in length, 250 m in width, and 150 m in height (Figure 1). The simulated reservoir in this study has a depth of 600 m from the surface to the bottom, while the roof of the reservoir is located at a burial depth of 300 m. In addition, to examine the impact of the space of production well on heat extraction efficiency, EGS system comprises of injection well and production wells. Three distinct situations are established, wherein each model features a single injection well with a length of 50 m with coordinates midpoint at X: -100, Y: 0, Z: 0 (Figure 2), and the origin of coordinates situates at the central position of the EGS system (Figure 1). The spacing of production wells in Case 50, Case 100, and Case 150 is 50, 100, and 150 m, respectively (Figure 2).

3 Governing equations for numerical model establishment

3.1 Model hypotheses

In this work, a 3D THM coupling model was created to simulate the procedure of the heat extraction from HDR system. The model incorporates a few hypotheses pertaining to heat transmission and fluid flow to effectively simulate the process (Zimmermann et al., 2009; Ye et al., 2021; Huang et al., 2023; Wang et al., 2023; Liu et al., 2024).



TABLE 1 Reservoir physical properties.

Parameter	Value	Unit
Initial pressure in seepage field	30	MPa
Injection rate in seepage field	10	kg/s
Production pressure in seepage field	20	MPa
Upper and lower boundaries in seepage field	Impermeable	
Initial temperature in temperature field	473.15	K
Injection temperature in temperature field	303.15	K
Upper and lower boundaries in temperature field	Thermal insulation	
Matrix density	2,700	kg/m ³
Matrix porosity	0.2	-
Matrix permeability	5e-15	m ²
Matrix heat capacity	950	J/(kg·K)
Matrix thermal conductivity	2.8	W/(m·K)
Fluid compressibility	1e-8	1/Pa
Biot coefficient	1	-

- (1) The working fluid is water, which exists as a liquid in the pores throughout the heat extraction procedure.
- (2) It is assumed that the initial EGS system brims with water, and the fluid in the matrix follows Darcy's law as laminar flow.
- (3) The heat transmission process within the matrix is governed by Fourier's law. Calorific balance can be achieved locally between the rock mas and working fluid.

These hypotheses are commonly used as reasonable conditions in numerical studies related to EGS systems (Lu, 2018; Zhao et al., 2023).

3.2 Governing equations

Therefore, based on the above assumptions, the main control equations of this heat extraction simulation process are as follows

(Sun et al., 2019; Tan et al., 2021; Zinsalo et al., 2021; Zhao et al., 2023).

When the working fluid flow in the porous medium a, the mass conservation law in the seepage field can be described as:

$$S\frac{\partial p}{\partial t} + \nabla \cdot q = -Q_f \tag{1}$$

Furthermore, according to the Darcy's law, the expressions of q is defined as:

$$q = -\frac{k}{\mu_f} \nabla \cdot \left(p + \rho_w g z \right) \tag{2}$$

The local thermal balance is the main feature of the heat exchange between the cryogenic fluid and the rock surface in the temperature field. During this heat exchange process, the liquid and the solid have the same temperature at any location. Hence, the energy conservation law can be expressed as:

$$\left(\rho c_p\right)_m \frac{\partial T}{\partial t} + \nabla \cdot \left(\rho_w c_{p,w} qT\right) - \nabla \cdot \left(\lambda_m \nabla T\right) = -Q_{f,E}$$
(3)

$$\left(\rho c_p\right)_m = (1 - \varphi)\rho_s c_{p,s} + \varphi \rho_w c_{p,w} \tag{4}$$

$$\lambda_m = (1 - \varphi)\lambda_s + \varphi\lambda_w \tag{5}$$

3.3 Effect of temperature on water properties

In addition, the temperature can determine the density (ρ_w) , the thermal conductivities (λ_w) , the heat capacity $(c_{p,w})$, and the dynamic fluid viscosity (μ_f) of water. The governing equations are described as (Sun et al., 2019; Aliyu and Archer, 2021; Zhou et al., 2022):

 $\mu_{f} = \begin{cases} 1.3799 - 0.0212T + 1.3604 \times 10^{-4}T^{2} - 4.6454 \times 10^{-7}T^{3} + 8.9043 \times 10^{-10}T^{4} \\ -9.0791 \times 10^{-13}T^{5} + 3.8457 \times 10^{-16}T^{6} & 273.15K \leq T \leq 413.15K \\ 0.004 - 2.1075 \times 10^{-5}T + 3.8577 \times 10^{-8}T^{2} - 2.3973 \times 10^{-11}T^{3} & 413.15K \leq T \leq 573.15K \end{cases}$ $c_{p,w} = 12010 - 80.4T + 0.3T^{2} - 5.4 \times 10^{-4}T^{3} + 3.6 \times 10^{-7}T^{4} \\ 273.15K \leq T \leq 573.15K \end{cases}$ (7)



$$\lambda_{w} = 7.9754 \times 10^{-9}T^{3} - 1.5837 \times 10^{-5}T^{2} + 0.0089T - 0.8691$$

$$273.15K \le T \le 573.15K$$

$$\rho_{w} = 838.4661 + 1.4005T - 3 \times 10^{-3}T^{2} - 3.7182 \times 10^{-7}T^{3}$$
(8)

(9)
$$273.15K \le T \le 573.15K$$

In this work, the boundary/initial conditions and main reservoir physical parameters for the numerical simulation model referred from previous research (Han et al., 2020; Yu et al., 2021; Zhou et al., 2022) are listed in Table 1. Moreover, during the modeling process, all simulation cases run in 30 years.

4 Mathematical results and discussion

The assessment of heat extraction performance in an EGS system heavily relies on temperature as a fundamental parameter (Rodriguez et al., 2013; Guo et al., 2018; Yang et al., 2021). Consequently, this study focuses on comparing the heat extraction efficiency of an EGS system utilizing water as the working fluid by analyzing temperature. Firstly, the overall temperature distribution throughout the EGS system is investigated, followed by setting up three reference points to track the temperature variation during variable simulated cases. Subsequently, three operation cases (Case 50, Case 100 and Case 150) were compared in terms of the temperature changes across the entire system.

4.1 Overall trend of temperature variation for different operation cases in the EGS system

In this work, the trend of overall temperature change from various directions for different numerical simulation cases are analyzed from three planes, which include plane X-Z (Y = 0), plane Y-Z (X = 0) and plane X-Y (Z = 0)–according to the coordinate system in Figure 1.

4.1.1 X-Z plane

The X-Z plane represents a cross-section that passes through both the injection well and the center of the EGS system (Figure 1). In Figure 2, the Y-coordinate is set to 0 on this plane. Basically, the area of cooling place gets larger from the injection well to production well during the water injection process (Figures 3–5). Nevertheless, in the X-Z plane, the temperature variation tendency seems to be similar for different production well spacing. Herein, the reason for the insignificant difference among all numerical simulation cases might be analyzed from two aspects. On the one hand, during the simulated process, the water is injected from one injection well with stable injection rate of 10 kg/s. On the other hand, the water (relatively low temperature) can scarcely transport on a large scale in the low permeability system, which makes the water extraction from production well influenced the water seepage. In general, the overall temperature variation tendency in X-Z plane of EGS system is less affected by the well spacing.

4.1.2 Y-Z plane

The Y-Z plane is a slide that across the EGS center, where X =0 in the coordinate system in Figure 1. Herein, as the water continuously injected from injection well, the area of cooling place becomes grater (Figures 6-8). When comparing the temperature variations among the different numerical cases, a noticeable difference can be observed when examining the X-Z plane (Figures 3-5) as mentioned earlier. It becomes evident that having smaller production well spacing leads to a smaller cooling area during heat extraction. This observation provides a perceptual understanding of the relationship between the spacing of production wells and the corresponding cooling area. Additionally, considering the inconspicuous difference observed in the X-Z plane, it can be hypothesized that the variation in temperature resulting from the space of production well primarily affects the Y-Z plane. Therefore, during heat extraction process, this variation is likely to contribute to the volume difference.

4.1.3 X-Y plane

In this study, the X-Y plane is perpendicular to the wellbore of production/injection well (Figure 1). It is utilized to examine the horizontal temperature variation within the EGS system. In general, across all operational cases, the cooling area within the EGS system gradually expands as water is injected from the injection well. It is observed that this expansion occurs in a manner where the cooling area extends from the injection well towards the production well (Figures 9–11). Similar to the temperature variation tendency of the X-Z plane (Figures 3–5), the difference among three numerical cases is not apparent in the X-Y plane and the stable injection rate and low permeability system might be the main factors.







4.2 Temperature variation in reference points for different operation cases in the EGS system

In order to more comprehensively illustrate the temperature change among distinct simulation cases, the dynamic alterations in temperature are quantitatively evaluated based on three reference points. These representative points are located at coordinates (X = 100, Y = -50, Z = 25), (X = 100, Y = 0, Z = 25), and (X = 100, Y = 50, Z = 25), as illustrated

in Figure 1. The temperature change at the three reference points is visually depicted in Figures 12–14, respectively.

In general, at each reference point, the temperature undergoes only slight variations during the initial ~3"years across all simulation cases (as shown in Figures 12–14). This is attributed to the fact that the cooling water injected into the system has not yet reached these points, and also due to the fact that the heat extraction from the production well has a relatively mild impact on the EGS. Subsequently, the temperature at these points begins to decrease as the cooling water is continually







injected and reaches the reference points, exhibiting varying degrees of amplitude in its variation. For Case 50, the temperature variation at the designated coordinates (X = 100, Y = 50, Z = 25) is gradual and steady throughout the entire heat extraction process. This can be attributed to

the water pressure discrepancy between the injection well and the production well, which stimulates the movement of cooled water in the direction of the production well. However, for Case 100 and Case 150, the injected cooling water eventually reaches the location at (X = 100,





Y = 50, Z = 25) due to the seepage induced by the hot water extraction from the production well. For Case 150, a larger quantity of cooling water flows towards this specific point compared to Case 100. This discrepancy arises because the injected cooling water exhibits a tendency to flow towards the production well, driven by the disparity in fluid pressure.

In regard to the point at (X = 100, Y = 0, Z = 25), the temperature fluctuation demonstrates a comparable pattern and inclination in Case 50 and Case 100 (as shown in Figure 13). This is likely due to the fact that, the cooling water encounters analogous seepage space and flow conditions in its path towards this reference point. Moreover, the disparity in fluid pressure between the injection well and the location at (X = 100, Y = 0, Z = 25) is comparable for both Case 50 and Case 100, as demonstrated in this work. Throughout the entire heat extraction period in Case 150, the temperature at the reference coordinates (X = 100, Y = 50, Z = 25) exhibits a consistently smooth variation. This can be also

attributed to the discrepancy in water pressure between the injection well and the production well, which propels the flow of cooling water towards the production well and ensures a continuous provision of cooling water to the reference points. Additionally, an observation can be made at the location of (X = 100, Y = -50, Z = 25), where there is a comparable temperature variation tendency (as seen in Figure 12) to that observed at point (X = 100, Y = 50, Z = 25) (as depicted in Figure 14). It can be inferred that the underlying mechanism responsible for the temperature variation at both of these points is similar to the aforementioned process.

4.3 Attenuation process of temperature in the whole EGS system

In this study, an efficiency metric called heat extraction efficiency (marked as *ef*) has been introduced (Zhao et al., 2023).









The purpose of this metric is to examine the attenuation process during the temperature variation in the EGS. The heat extraction efficiency is derived by dividing the heat recovery by the total heat stored in the EGS, providing valuable insights into the overall effectiveness of heat extraction (Zhao et al., 2023),

$$f = \frac{\iiint_{V_s} \rho_s c_{p,s} (T_0 - T) dV}{\iiint_{V_s} \rho_s c_{p,s} (T_0 - T_{inj}) dV}$$
(10)

Figure 15 presents the *ef* performance for each numerical case, as determined by Eq. 10. The calculations reveal slight variations among the different simulation cases, as shown in Figure 15. In general, it can be noted that for three numerical simulation cases, there is a swift surge in the *ef* during the first 10 years, succeeded by a relatively sluggish progression in the

final 20 years. Throughout the heat extraction process, the variation of ef for Case 50 exhibits a slightly lower trend compared to Case 100 and Case 150. Therefore, it could be concluded that the larger the well spacing, the higher the value of ef during the last 20 years.

5 Conclusion

When investigating the temperature variation during the heat extraction process, there are minimal disparities observed among the three operation cases on the X-Y plane and Y-Z plane. Furthermore, a noticeable observation is that smaller production well spacing leads to a smaller cooling area during heat extraction.

Based on the investigation conducted on the points (X = 100: Y = 0: Z = 25), (X = 100: Y = 50: Z = 25), and (X = 100: Y = -50: Z = 25), it is observed that the temperature is piecemeal reduce in each reference point for the unceasing injection of cooling water and its arrival at the representative locations at (X = 100, Y = 0, Z = 25), (X = 100, Y = -50, Z = 25), and (X = 100, Y = 50, Z = 25), albeit with varying amplitude of variation. Notably, among the three numerical cases, the temperature change at points (X = 100: Y = -50: Z = 25) and (X = 100: Y = 50: Z = 25) exhibits a significant difference. Conversely, for Case 50 and Case 100, the temperature change at the point (X = 100: Y = 0: Z = 25) appears less pronounced.

As for the *ef*, it is noteworthy that the variation of the *ef* for Case 50 exhibits a slightly lower trend compared to Case 100 and Case 150. This finding suggests that the spacing of production wells has impact on the *ef* in an EGS system during the last 15 years. Therein, it could be concluded that the larger the well spacing, the higher the value of *ef*.

Data availability statement

The original contributions presented in the study are included in the article/Supplementary material, further inquiries can be directed to the corresponding author.

Author contributions

ZW: Conceptualization, Writing-original draft. BZ: Conceptualization, Methodology, Writing-original draft. XZ: Data curation, Methodology, Writing-original draft. LY: Investigation, Project administration, Writing-original draft. YF: Conceptualization, Methodology, Writing-review and editing. HY:

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Conflict of interest

Authors ZW, BZ, XZ, LY, YF, and HY were employed by China Three Gorges Corporation. Author BZ was employed by China Three Gorges Corporation.

The remaining authors declare that the research was conducted in the absence of any commercial or financial relationships that could be construed as a potential conflict of interest.

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Nomenclature

S	storage coefficient of the rock matrix (Pa ⁻¹)
Р	pore pressure (Pa)
Т	time (s)
q	is the Darcy velocity (m/s)
q _f	the Darcy velocity in the fracture (m/s)
Q _f	the source (1/s)
k	permeabilities of the rock matrix (m ²)
μ _f	dynamic fluid viscosity (Pa·s)
$\rho_{\rm w}$	fluid density (kg/m ³)
g	gravitational acceleration (m/s ²)
z	unit vector in the vertical direction
Т	temperature (K)
c _{p,w} is	the heat capacity of the fluid (J/(kg·K))
Q _{f,E}	heat source (W/m ³)
$(\rho c_p)_m$	effective volumetric heat capacities of the matrix $(J/(m^{3}{\boldsymbol{\cdot}}K))$
$\lambda_{\rm m}$	effective thermal conductivities of the matrix $(W/(m{\cdot}K))$
Φ	porosities of the matrix
ρ_s	solid density (kg/m ³)
c _{p,s}	solid heat capacity (J/(kg·K))
λ_s	thermal conductivities of the solid $(W/(m{\cdot}K))$
λ_{w}	thermal conductivities of the fluid $(W/(m{\cdot}K))$
Vs	heat extraction zone in the EGS
T ₀	initial temperature
T _{inj}	injection temperature of the fluid (namely, cooling water)