

Prediction of Drag Reduction in Slickwater Fracturing by Two General Models

Pengfei Chen*, Honggang Chang, Yongqiang Fu, Yongfan Tang, Xuesong Huang and Weichu Yu

Research Institute of Natural Gas Technology, PetroChina Southwest Oil and Gas Field Company, Chengdu, China

Drag reduction (DR) is critical to the success of hydraulic fracturing operations with slickwater, and it is a challenge to accurately predict DR due to the problem of high injection rates. Although a practical pipe diameter model is frequently used to predict the field DR based on laboratory experimental data, there exist many limitations. This study, on account of dynamic similarity, shows two novel general models for the prediction of field DR, and such two models can give reliable predictions when the laboratory and field Reynolds numbers (*Re*) are the same. For general model 1, the DR can be predicted by using the laboratory volumetric flow rate, pipe diameter and pressure drop, and the field volumetric flow rate, with a deviation ranging from -10 to 10%. For general model 2, it is simpler than general model 1, and the DR can be predicted by using the laboratory pipe diameter and the field volumetric flow rate, with a deviation ranging from -6 to 6%. The two novel general models can be used for more scenarios than the existing reported ones.

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*Correspondence: Pengfei Chen chenpengfei@petrochina.com.cn

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INTRODUCTION

As a widely distributed clean and efficient energy source, shale gas has been highly valued by the international energy market and various countries (Peng et al., 2019a). Since the early 21st century, the boom in shale gas development in the United States has significantly advanced the global shale gas development. Since then, the energy landscape in the world has changed gradually (Yuan et al., 2015; Shi et al., 2020). As far as we know, China has abundant shale gas resources and an increasing demand for natural gas. Therefore, a large-scale commercialized development (Ma et al., 2018; Pang, 2018) of hydraulic fracturing shale reservoirs can be expected. Hydraulic fracturing is a necessary technology for developing shale gas (Yu et al., 2020). During the hydraulic fracturing process, the proppant-carrying fracturing fluid is injected into a well at a high pressure and rate to fracture the reservoir rocks. To reduce the friction loss due to tubular roughness, slickwater with drag reducers is often used since it has desirable hydraulic features (Barbot et al., 2013; Shaffer et al., 2013; Al-Muntasheri, 2014; Engle and Rowan, 2014). Numerous studies have focused on various parameters affecting drag reduction (Gallego and Shah, 2009; Chai et al., 2019). Four main parameters can affect the drag reduction of slickwater, including the drag-reducing agent concentration, fluid flow Reynolds number (Re), relative pipe roughness, and water quality (Yang et al., 2019). Meanwhile, two main underlying mechanisms of drag reduction have been identified (Habibpour and Clark, 2017; Habibpour et al., 2017). The first mechanism introduced by Lumley is based on the elongation of coiled polymer molecules, hence increasing the thickness of the viscous sublayer; the other mechanism is the elastic properties of polymers.

Although we have known a lot of the slickwater parameters affecting drag reduction, accurate prediction of drag reduction still remains a challenge. Allahdadi Mehrabadi and Sadeghy (2008) obtained a good drag reduction prediction model called the κ - ϵ turbulence model for low Reynolds numbers. Al-Sarkhi et al. (2011) developed two correlations to predict the effect of drag-reducing polymers on the friction factor of the two-phase flow for any pipe diameter. Karami and Mowla (2013) presented a general model for predicting drag reduction in crude oil pipelines. Recently, Zhou et al. (2011) proposed a practical pipe diameter model by introducing an effective pipe radius ($r_{\rm eff}$) which is defined as follows:

$$r_{eff} = \frac{v_f \rho}{\mu},$$

where ρ is the fluid density, μ is the fluid viscosity, and v_f is the flow velocity calculated as $(d\Delta p/4\rho l)^{1/2}$. To upscale the laboratory experimental results of the drag reduction of slickwater to the field application, two conditions must be met: first, the r_{eff} value of the field pipe (tubing/casing) must be in the same range as the laboratory pipe; second, the slope of r_{eff} versus v of the field must match the laboratory setup, where v is the average fluid velocity. We found that as long as the slopes of effective pipe radius versus velocity from the laboratory pipe and the field pipe are numerically close, field drag reduction can be predicted by a modified correlation between DR and velocity established in the laboratory (Zhao et al., 2018; Chen et al., 2021). In this study, two novel general models are proposed using the Reynolds criterion to simplify the prediction condition.

THEORETICAL BASIS

The models presented in this study are based on the similarity principle so as to upscale the laboratory experimental results to the field application. Due to laboratory setting constraints that are different from those in the field application, the experimental results can be upscaled to the field application only under some specific conditions. The laboratory measurements can accurately represent the flow dynamics of the corresponding prototype. Therefore, the similarity between the model and the prototype must be satisfied. There are three types of similarity principles: geometric similarity, kinematic similarity, and dynamic similarity. It is relatively easy to achieve the geometric and kinematic similarities, while the dynamic similarity requires an equal ratio of forces acting on the two systems. Usually, dynamic similarity can be achieved by equating such flow dynamic dimensionless variables as Froude number and Reynolds number. In this study, we chose the Reynolds number as the inherent parameter because it can be easily measured in both the laboratory condition and the field application condition. Reynolds number (Re) is a dimensionless number that is commonly used to characterize flow patterns in different fluid flow situations (Peng et al., 2019b). Because of the similarity of flow viscosity between the prototype and the model, the Reynolds number of the field fracturing flow model should be equal to that



of the lab drag reduction experiment model, according to the similarity of flow dynamics.

In previous studies, we have developed a modified Virk's correlation to accurately characterize the friction factor (λ) under different Reynolds numbers in the turbulent flow pattern for the polyacrylamide polymer drag-reducing agent using laboratory data. The unified model is shown in **Eq. 1**.

$$\frac{1}{\sqrt{\lambda}} = \alpha D^{\beta} lg \Big(Re \sqrt{\lambda} \Big) + \gamma D^{\theta}.$$
(1)

This equation can be changed to Eq. 2.

$$\operatorname{Re}_{lab} = a \mathcal{D}_{lab}^m \lambda_{lab}^n, \tag{2}$$

where Re_{lab} is the Reynolds number based on the laboratory data, D_{lab} is the lab pipe diameter, and λ_{lab} is the friction factor from the laboratory data. For a given pipe diameter in the field, a similar relationship between the field Re (Re_{fie}) and the field λ (λ_{fie}) can be obtained by **Eq. 3**.

$$\operatorname{Re}_{fie} = b\lambda_{fie}^{n}.$$
(3)

The Reynolds numbers of the lab and field can be equated by **Eq. 4**:

$$\lambda_{fie} = \left(\frac{aD_{lab}^m}{b}\right)^{1/n} \lambda_{lab}.$$
 (4)

For a slickwater solution, the λ_{lab} and λ_{fie} can be determined by the Hagen–Poiseuille equation, respectively, as follows:

$$\lambda_{lab} = \frac{\pi^2 \mathcal{D}_{lab}^5 \Delta P_{lab}}{8\rho \mathcal{L}_{lab} \mathcal{Q}_{lab}^2},\tag{5}$$

$$\lambda_{fie} = \frac{\pi^2 \mathcal{D}_{fie}^5 \Delta P_{fie}}{8\rho \mathcal{L}_{fie} \mathcal{Q}_{fie}^2},\tag{6}$$

where ΔP_{lab} is the pressure drop in the laboratory experiment, ρ is the density of slickwater, L_{lab} is the pipe length in the laboratory experiment, Q_{lab} is the volumetric flow rate in the laboratory experiment, ΔP_{sie} is the field friction of slickwater, L_{fie} is the well depth, and Q_{fie} is the field volumetric flow rate.



Eq. 5 and **6** are substituted into **Eq. 4** to get the predicted ΔP_{fie} by **Eq. 7**:

$$\Delta P_{fie} = \left(\frac{a}{b} D_{lab}^{m}\right)^{1/n} \frac{D_{lab}^{5} \Delta P_{lab} L_{fie} Q_{fie}^{2}}{L_{lab} Q_{lab}^{2} D_{fie}^{5}}.$$
 (7)

For the given L_{fie} , Q_{fie} , and D_{fie} , ΔP_{fie} can be calculated according to the process shown in **Figure 1**.

The field friction of pure water (ΔP_{pw}) can be calculated by **Eq.** 8, which is obtained by the friction-gradient flow rate diagram of water, as shown in Supplementary Figure S1 in Supporting Information.

$$\Delta P_{pw} = 1.385 \times 10^6 D_{fie}^{-4.8} Q_{fie}^{1.8} L_{fie}.$$
 (8)

The drag reduction (DR) is defined as follows:

$$DR = 100 - \frac{\Delta P_{fie}}{\Delta P_{pw}} \times 100.$$
(9)

Eq. 7 and 8 are substituted into Eq. 9 to get the DR:

$$DR = 100 - 7.22 \times 10^{-5} \left(\frac{a}{b} D_{lab}^{m}\right)^{1/n} \frac{D_{lab}^{5} \Delta P_{lab} Q_{fie}^{0.2}}{L_{lab} Q_{lab}^{2} D_{fie}^{-9.8}} \times 100.$$
(10)

EXPERIMENT

Materials

The drag-reducing agent, having an average molecular weight of 1.05×10^7 and a hydrolysis degree of 21.5%, was provided by Chengdu Ringt Technology Development Co., Ltd. In the laboratory experiment, an aqueous solution of 0.1 wt% drag-reducing agent and deionized water as the solvent were used. The experimental setup used to study the drag reduction performance is shown in Supplementary Figure S2 in Supporting Information. The laboratory flow loop was composed of a mixing tank (20 L), a



cavity progressive pump (XBY, model: G FG 25-2, range: $0-2 \text{ m}^3 \text{h}^{-1}$), a steel pipe (relative roughness: 3.2), a heater, and a flow meter (SINCERITY, model: DMF-1-M, range: $0-35 \text{ kg} \text{ min}^{-1}$, accuracy: 0.10-0.20% relative error of the flow meter). It also had two pressure sensors (Sailsors Instruments Ltd., model: V4Db7E, range: -700 to 700 kPa, output: 4-20 mA DC, accuracy: $\pm 0.5\%$ full scale).

Drag Reduction Experiments in the Laboratory

The drag reduction experiments in the laboratory were conducted, following the procedures detailed in the literature (Zhao et al., 2018).

1) Drag-reducing agent and experimental equipment

The drag-reducing agent used in the experiment was the high molecular weight polyacrylamide emulsion prepared by inverse emulsion polymerization. It was a linear polymer with a molecular weight of 1.05×10^7 and a concentration of 31.2 wt%. The dosage of the linear polymer was 0.1wt%. All experiments were carried out at room temperature. The loop included a 50-L container, and it can circulate at different rates up to 50 L/min under the action of the pump. First, the liquid flowed from the mixing tank through the flow meter having a maximum capacity of 10 t/h and an accuracy of ±0.1%. Then, the liquid flowed through the pipeline, and the pressure was measured by using the pressure sensors with a range of 0-14 MPa. A 30-m long pipe with a diameter of 0.08 m was used in the system. A 50-L container was used to prepare the drag-reducing agent solution, and the solution can be recycled. Then, experiments were conducted at different flow rates.

- 2) Experimental procedures
- 1) The water was added to a mixing tank and allowed to circulate for 5 min, followed by filling the pipeline with water and

TABLE 1 | Predicted DRs and deviations obtained by using general model 1.

Well Number of section ΔP _{ne} (MPa) section DR (%) (m/s) Average velocity (m/s) 7.8 mm 10 mm 7.8 mm 10 mm W1 1 25.24 69.40 22.84 23.28 25.15 71.77 3.30 69.51 0.16 3 23.86 65.20 20.23 17.71 19.12 74.18 12.10 72.11 9.56 4 29.79 66.95 24.39 25.37 27.40 71.86 11.56 69.60 8.70 10 25.9 70.17 26.02 28.47 30.75 69.48 3.12 67.04 -0.41 10 25.9 70.17 26.02 26.71 28.85 69.23 -1.35 66.77 -5.09 11 25.01 70.77 26.02 25.74 27.80 69.48 5.49 67.04 2.04 13 27.97 66.54 26.00 23.73 25.63 69.49 5.62 67.04 2.18 1.4 28.16
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13 25.22 64.06 22.78 19.70 21.28 71.92 10.93 69.68 8.06
14 23.5 71.62 26.01 25.24 27.26 69.52 -3.02 67.08 -6.77
15 26.83 60.36 22.79 19.01 20.53 71.91 16.07 69.66 13.36
16 22.74 70.07 24.48 21.55 23.27 71.64 2.19 69.37 -1.01
17 28.66 63.49 26.01 23.94 25.86 69.50 8.65 67.06 5.33
18 25.13 67.47 26.00 23.55 25.43 69.52 2.95 67.09 -0.58
19 23.43 69.19 26.04 23.24 25.10 69.44 0.36 66.99 -3.28
20 20.38 72.76 26.04 22.87 24.70 69.43 -4.80 66.98 -8.63
21 18.85 74.39 26.06 22.53 24.33 69.39 -7.21 66.94 -11.13
22 16.18 73.15 22.75 16.88 18.23 72.00 -1.60 69.76 -4.87
23 19.67 72.32 26.01 21.67 23.40 69.50 -4.05 67.06 -7.84







checking the flow meter and pressure sensor to ensure proper and stable range.

- 2) The water was circulated for 5 min, and the pressure difference (ΔP_0) per second and the flow rate were recorded.
- 3) A drag-reducing agent was added to the water and allowed to circulate for 5 min, followed by recording the pressure difference (ΔP_1) per second and the flow rate.
- 4) The system was cleaned with water.
- 5) The drag reduction was calculated.
- 6) The drag reduction performance was evaluated by Reynolds number, shear rate, friction coefficient, and average velocity.

RESULTS AND DISCUSSION

TABLE 2 | Data used for parameter determination

Construction and Validation of General Model 1

Figure 2 shows the changes of Re_{lab} with λ_{lab} of slickwater in pipes with a diameter of 7.8 and 10.15 mm, respectively. We applied numerical regression to the experimental data and

obtained the correlation **Eq. 11** with coefficients of a = 0.0011, m = 1.76, and n = -2.8.

$$\operatorname{Re}_{lab} = 1.1 \times 10^{-3} \operatorname{D}_{lab}^{1.76} \lambda_{lab}^{-2.8}.$$
(11)

Figure 3 shows the changes of Re_{fie} with λ_{fie} of slickwater in the pipe with a diameter of 114.3 mm. For this pipe, the correlation was obtained by **Eq. 12**.

$$\operatorname{Re}_{fie} = 8.8 \times 10^{-4} \lambda_{fie}^{-2.8}.$$
 (12)

Eq. 11 and **12** should be substituted into **Eq. 7** and **10** to obtain the predicted ΔP_{fie} and drag reduction (by general model 1), respectively, by **Eq. 13** and **14**:

$$\Delta P_{fie} = \frac{0.66}{D_{lab}^{0.55}} \frac{D_{lab}^5}{D_{fie}^5} \frac{L_{fie}}{3} \frac{Q_{fie}^2}{(Q_{lab}/60 \times 1000)^2} \Delta P_{lab}, \quad (13)$$

$$DR = 100 - 7.22 \times 10^{-5} \left(1.25 D_{lab}^{1.76}\right)^{-0.36} \frac{D_{lab}^5 \Delta P_{lab} Q_{fie}^{0.2}}{L_{lab} Q_{lab}^2 D_{fie}^{-9.8}} \times 100.$$
(14)

For the pipe with a diameter of 114.3 mm, the field data, predicted $\Delta P_{\rm fie}$, and predicted DR are listed in **Table 1**. Since $\Delta P_{\rm fie}$ predicted using D_{lab} of 10 mm was higher than that of D_{lab} of 7.8 mm, the DR predicted using D_{lab} of 10 mm was lower than that of D_{lab} of 7.8 mm. **Figure 4** shows the DR prediction deviations. In general, the prediction deviation produced by using D_{lab} of 7.8 mm was lower than that of D_{lab} of 10 mm, and the deviation was -10–10%.

Construction and Validation of General Model 2

As shown in **Figure 1** and **Eq. 10**, the DR prediction using general model 1 is very complicated. Many parameters in the model cannot be measured directly. Therefore, general model 1 should be further simplified. When L_{fie} and D_{fie} are known, ΔP_{fie} is a function of Re_{lab} , D_{lab} , and V_{fie} , as shown in **Eq. 7**. Therefore, **Eq. 10** can be changed to **Eq. 15**as follows:

D _{fie} (mm)	Q _{fie} (m ³ /min)	V _{fie} (m/s)	D _{<i>lab</i>} = 7.8 mm		D _{lab} = 10 mm	
			d (ΔP_{lab} /V)/d (ΔP_{lab})	Calculated DR	d (∆P _{lab} /V)/d∆p _{lab}	Calculated DR
114.3	1	1.63	0.0087	60.32	0.0147	60.43
114.3	2	3.25	0.0087	63.99	0.0147	64.09
114.3	3	4.88	0.0087	65.98	0.0147	66.07
114.3	4	6.50	0.0087	67.32	0.0147	67.41
114.3	5	8.13	0.0087	68.33	0.0147	68.41
114.3	7	11.38	0.0087	69.79	0.0147	69.87
114.3	9	14.63	0.0087	70.83	0.0147	70.91
114.3	10	16.25	0.0087	71.26	0.0147	71.34
114.3	12	19.50	0.0087	71.98	0.0147	72.06
114.3	14	22.75	0.0087	72.58	0.0147	72.65
114.3	16	26.00	0.0087	73.09	0.0147	73.16
114.3	18	29.25	0.0087	73.53	0.0147	73.60
114.3	20	32.50	0.0087	73.92	0.0147	73.99



$$DR = 100 - x \operatorname{Re}_{lab}^{y} D_{lab}^{z} V_{fie}^{q}.$$
 (15)

Re_{*lab*} is also a parameter that cannot be measured directly. Our laboratory experimental results showed that there was a linear relationship between $\Delta P_{lab}/V_{lab}$ and Re_{*lab*} (Figure 5) within our experiment range. In addition, we also found that there was a linear relationship between $\Delta P_{lab}/V_{lab}$ and ΔP_{lab} (see Figure 6). As shown in Figure 5 and Figure 6, Re_{*lab*} and ΔP_{lab} also have a linear relationship. In other words, d ($\Delta P_{lab}/V$)/d (ΔP_{lab}) and Re_{*lab*} have multiple relationships. As a result, Eq. 15 can be converted to Eq. 16as follows:

$$DR = 100 - x' \times \left[\frac{d\left(\Delta P_{lab}/V_{lab}\right)}{d\left(\Delta P_{lab}\right)}\right]^{y'} \times D^{z}_{lab}V^{q}_{fie}.$$
 (16)

When V_{fie} is changed from 1.63 m/s to 32.5 m/s, the DR was calculated for different pipe diameters by adjusting the parameters in **Eq. 16**. The data used for parameter determination are shown in **Table 2**. Using numerical regression, we obtained general model 2, as shown in **Eq. 17**.

$$DR = 100 - 56 \times \left[\frac{d(\Delta P_{lab}/V_{lab})}{d(\Delta P_{lab})}\right]^{0.028} \times D_{lab}^{-0.07} \times V_{fie}^{-0.14}.$$
 (17)

When the pipe diameter was 114.3 mm, the field data and the predicted ΔP_{fie} and DR were obtained using general model 2. **Figure 7** shows the DR prediction with a deviation ranging from -6 to 6%.

Deviation Analysis

This study presents models from regression and upscaling laboratory data to predict the field performance of slickwater. The potential deviation from the developed model comes from three different sources: 1) uncertainties of the laboratory measurements and their propagation to the regression correlation. To overcome the system deviation of measurement, multiple measurements on rates, diameters, and pressures were conducted, and the averaged values were used in the laboratory experimental results; 2) numerical regression on a laboratory experiment. To obtain the quantitative relationships between measured quantities and overcome the potential deviations, we applied weighted least squares with data uncertainties determined from experiments; 3) propagation of deviation during upscaling laboratory data to field applications. In this upscaling, we did not consider the potential impact of other differences such as different water quality and changes in fluid flow dynamics or thermodynamic properties, and all of them can contribute to the increased inaccuracy in the upscaling process.

Additionally, Eq. 16 shows that the friction resistance value predicted for the field is related to the linear relationship between the ratio of pressure difference to linear velocity and the differential pressure measured in the laboratory experiment, hydrodynamic radius, and linear velocity designed in the field. The calibration of the hydrodynamic radius is based on the water experiment, and the friction coefficient of water conforms to the Prandtl-Karman law in the hydraulic smooth area, which requires the accurate measurement of the stable pressure difference at different linear velocities under laboratory experimental conditions. Similarly, to establish an accurate linear relationship between the ratio of pressure difference to linear velocity and differential pressure, it is necessary to measure the stable pressure difference of the drag-reducing agent at different linear velocities. Therefore, the most important factor for accurate prediction of field friction is to obtain the stable pressure difference between water and drag-reducing agent solution at different linear velocities under laboratory experimental conditions.

CONCLUSION

Two novel general models are established to predict the field drag reduction using laboratory experimental data based on the dynamic similarity theory. When the laboratory and field Reynolds numbers are the same and the laboratory and field volumetric flow rates and pipe diameters are known, general model 1 can be used. When the field volumetric flow rates and the laboratory pipe diameter are known, general model 2 can be used. The two proposed mathematical models for field drag reduction prediction are validated by 42 data points. The validation results showed that both models can give high-accuracy predictions, with a deviation ranging from -10 to 10% using general model 1 and a deviation ranging from -6 to 6% using general model 2.

DATA AVAILABILITY STATEMENT

The original contributions presented in the study are included in the article/Supplementary Material; further inquiries can be directed to the corresponding author.

AUTHOR CONTRIBUTIONS

All authors listed have made a substantial, direct, and intellectual contribution to the work and approved it for publication.

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Conflict of Interest: Authors PC, HC, YF, YT, XH and WY were employed by PetroChina Southwest Oil and Gas Field Company.

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NOMENCLATURE

 $\label{eq:result} \begin{array}{l} \textbf{A:} \mbox{ friction factor, dimensionless;} \\ \textbf{Re:} \mbox{ Reynolds numbers, dimensionless;} \\ \textbf{Re_{lab}:} \mbox{ recalculated by using laboratory data, dimensionless;} \\ \textbf{D_{lab}:} \mbox{ pipe diameter in the laboratory experiment, mm;} \\ \textbf{D_{fie:} pipe diameter in the field application, mm;} \\ \textbf{\lambda_{lab}:} \mbox{ } \lambda \mbox{ calculated by using laboratory data, dimensionless;} \end{array}$

$$\begin{split} &\Delta \mathbf{P}_{lab}\text{:} \text{ pressure drop in the laboratory experiment, MPa;} \\ &\rho\text{: density of slickwater, g/cm}^3\text{;} \\ &\mathbf{L}_{lab}\text{: pipe length in the laboratory experiment, mm;} \\ &\mathbf{Q}_{lab}\text{: volumetric flow rate in the laboratory experiment, m}^3\text{/min;} \\ &\Delta \mathbf{P}_{fie}\text{: field friction of slickwater, MPa;} \\ &\mathbf{L}_{fie}\text{: well depth, m;} \\ &\mathbf{Q}_{fie}\text{: field volumetric flow rate, m}^3\text{/min;} \\ &\mathbf{V}_{fie}\text{: linear velocity, m/s.} \end{split}$$