



Research on Countercurrent Flow Limitation in Reactor Hot Leg at the Loss of Coolant Accident-Thermohydraulic Calculation With System Code RELAP5

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Since the counter current flow limitation (CCFL) of the two-phase flow has an important influence on the safety analysis of the reactor, it is also of great significance to study some characteristics and flow pattern transitions of the two-phase flow. In order to explore the characteristics and influencing factors of two-phase counter current flow in the hot leg under the condition of the Loss of Coolant Accident, this research, using RELAP5 simulation calculation, analyzed the effect of the lengths of the flow path of the hot leg, the shapes of the upper outlet of the inclined pipe, and the shapes of the elbow connecting the horizontal pipe and the inclined pipe on the characteristics of the CCFL. This research also evaluated the limitation of the model. The simulation founded that under the same liquid flow rate, the longer the horizontal pipe, the lower the gas flow rate at the beginning of the counter current flow; the effect of the elbow on the CCFL was not great; compared with the sharp outlet shape, the J_f of the gentle circle was slightly larger. The method established in this study is suitable for the prediction of CCFL in the hot leg during Loss of Coolant Accident.

Keywords: two-phase flow, RELAP5, counter current flow limitation, hot leg, LOCA

INTRODUCTION

The Loss of Coolant Accident, a break in the loop boundary or a stuck safety valve, etc., causes the coolant from the loop to flow out of the loop area. Flow erosion in piping systems is a concern for many industrial practices. The driving pressure head of natural circulation depends on the density difference between the up and down sections of the loop. In this process, if the main pump of the first loop of the nuclear power station stops idling, the natural circulation begins to change into a single phase, and the system pressure gradually drops. At this time, a lot of gas phase may be generated by flashing at the top of the core due to the high pressure, and thus the density difference increases in the pipeline, and ultimately the ability of natural circulation increases. But the resulting steam flow into the heat transfer pipe may be condensed, and soon the pressure in the primary loop is reduced to the pressure in the secondary loop. If the natural circulation is interrupted, CCFL will occur, and the

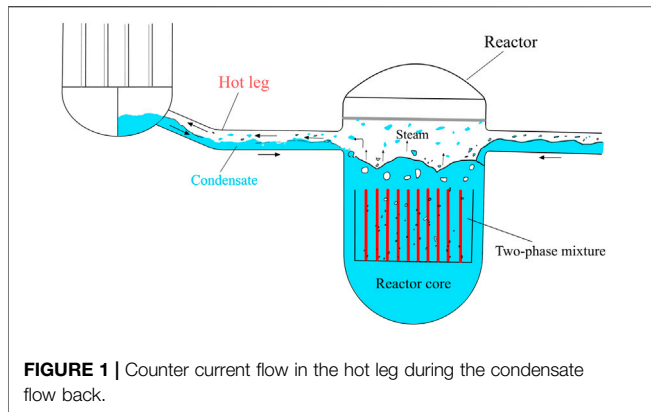


FIGURE 1 | Counter current flow in the hot leg during the condensate flow back.

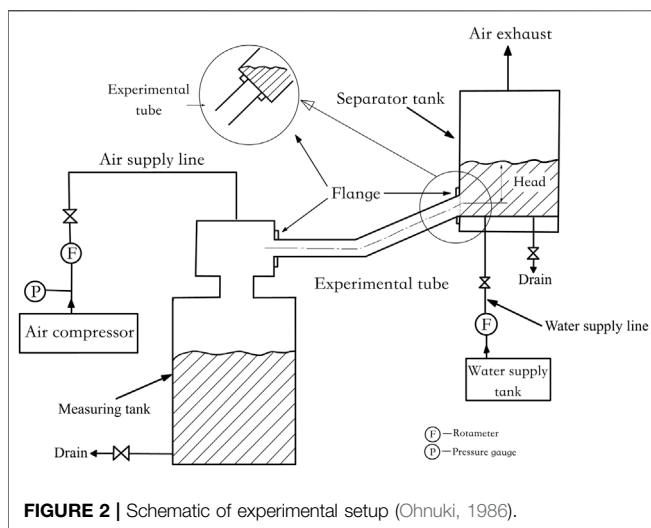


FIGURE 2 | Schematic of experimental setup (Ohnuki, 1986).

steam generated in the core will rise to the steam generator in the hot leg, be cooled by the secondary circuit, and then flow back down the pipe. Such counter current flow may take away some of the heat, as shown in **Figure 1**. In recent years, international scholars have carried out research on the characteristics of two-phase countercurrent flow in different pipeline systems. Among them, the two-phase countercurrent flow in the hot leg was studied by Siddiqui et al. (1986), Ardron and

TABLE 1 | Experiment section dimensions description.

| No | $\theta/^\circ$ | D/m | l/m | H/m | h/m | Type |
|------|-----------------|-------|------|------|------|------|
| I | 40 | 0.026 | 0.10 | - | 0.10 | A |
| II | 40 | 0.026 | 0.30 | - | 0.10 | A |
| III | 40 | 0.026 | 0.10 | 0.01 | 0.10 | B |
| IV | 40 | 0.026 | 0.19 | 0.02 | 0.10 | B |
| V | 40 | 0.026 | 0.19 | 0.05 | 0.10 | B |
| VI | 40 | 0.026 | 0.19 | 0.08 | 0.10 | B |
| VII | 40 | 0.026 | 0.19 | 0.08 | 0.10 | B |
| VIII | 40 | 0.026 | 0.10 | 0.01 | 0.10 | C |
| IX | 40 | 0.026 | 0.19 | 0.08 | 0.10 | D |

Banerjee (1986). Wongwises (1998) conducted a study of two-phase counter current flow in the hot leg of a PWR. Kang et al. (Hewitt et al., 1965) and Yu et al. (Chun et al., 2000), specifically studied the effect of liquid level in the hot leg on its CCFL characteristics. Murase et al. (2012) used the Fluent software to numerically simulate the CCFL in the hot leg, and to model the full-scale hot leg model and the 1/15 reduced-scale hot leg test piece respectively. For the full-scale hot leg model, the numerical calculation results were dimensionless. And they were in good agreement with the experimental data. But for the reduced size hot leg test piece model, the numerical calculation model underestimated the discharge fluid flow. Researchers have devoted great efforts to provide credible thermal-hydraulic system codes for the analyses of transients and accidents scenarios in nuclear power plants (Wang et al., 2021). And this paper mainly used RELAP5 for simulation calculation, and evaluated the CCFL model. Simultaneously, this paper studied the effect of the lengths of horizontal pipe, the shapes of the upper exit of inclined pipe, and the shapes of elbow connecting horizontal pipe with inclined on the characteristics of the CCFL.

EXPERIMENT INTRODUCTION

When LOCA occurs in a PWR, during the reflooding phase, the water collected at the steam generator inlet may flow back through the hot leg against the steam from the upper plenum, which creates a steam blockage in the primary loop. Among them, the counter current flow of water from the steam generator affects the accumulation of water and its horizontal distribution in the

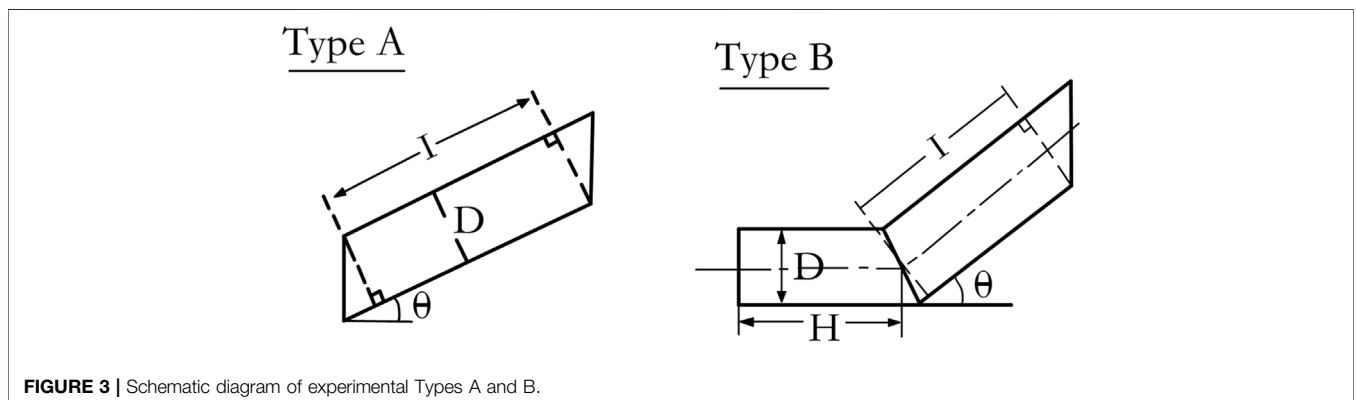


FIGURE 3 | Schematic diagram of experimental Types A and B.

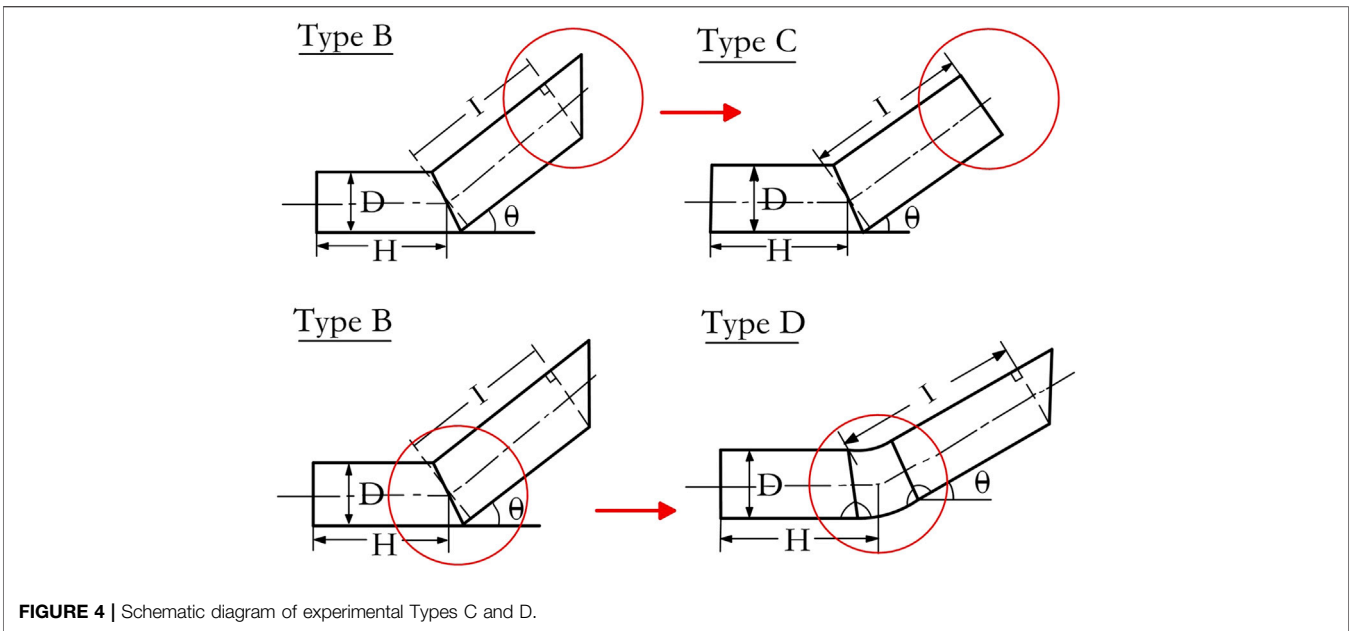


FIGURE 4 | Schematic diagram of experimental Types C and D.

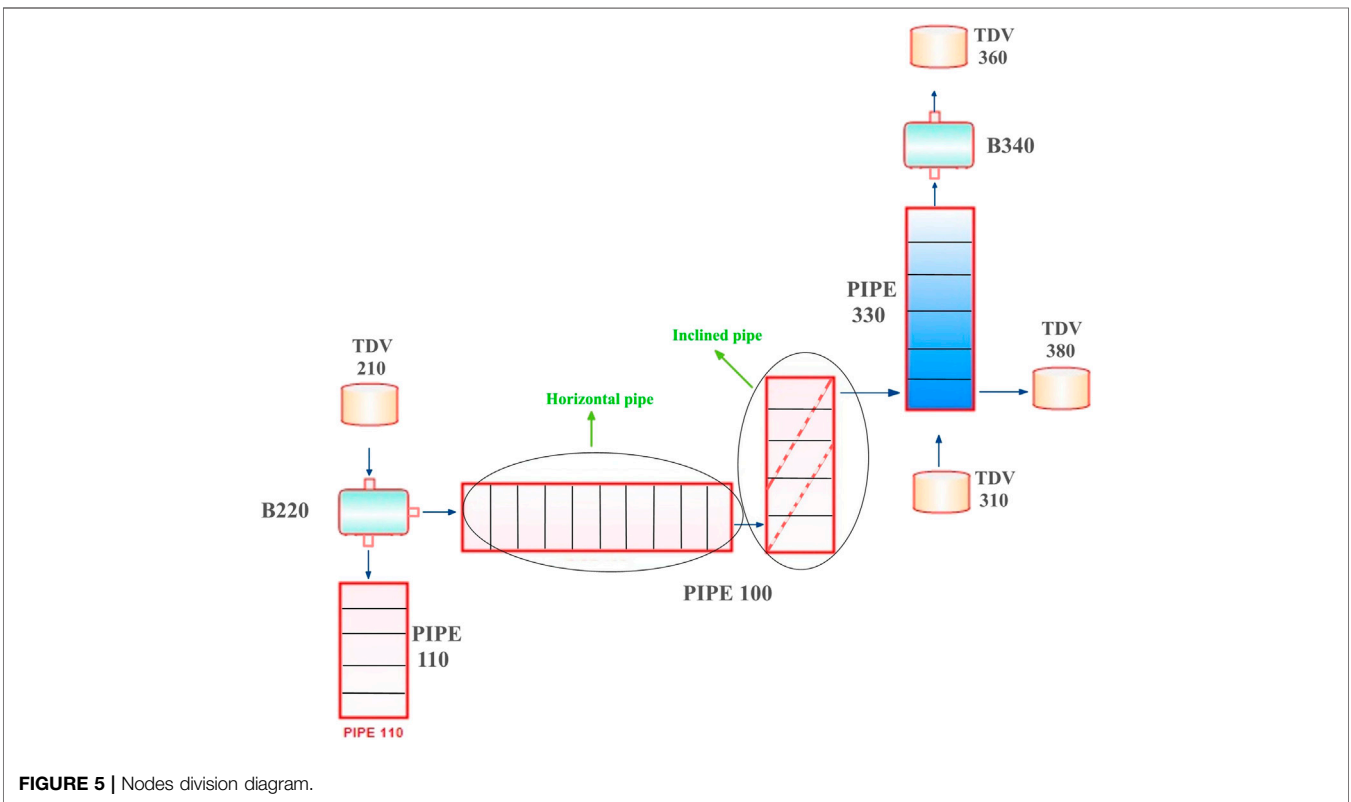


FIGURE 5 | Nodes division diagram.

upper plenum, thereby affects the cooling of the multi-dimensional core, and the flow rate of water depends on the CCFL characteristics in the hot leg. In 1986, Akira (Ohnuki, 1986) conducted an experiment research in order to study the

characteristics of CCFL in the model flow channel. This paper conducted a simulation study based on this experiment. The experimental system as shown in Figure 2. The cross section of the separator tank was a square of 0.21 m × 0.21 m and the height

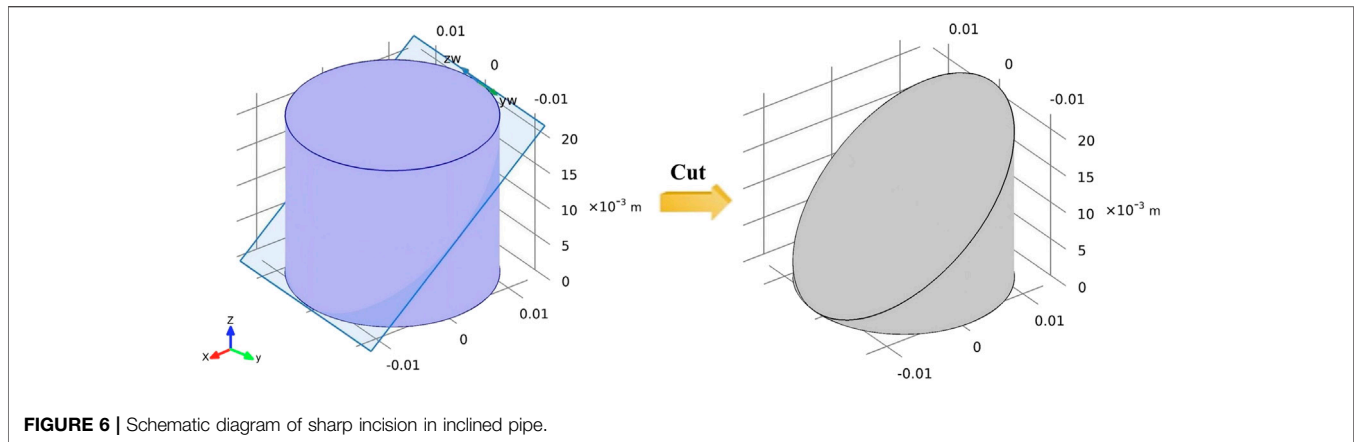


FIGURE 6 | Schematic diagram of sharp incision in inclined pipe.

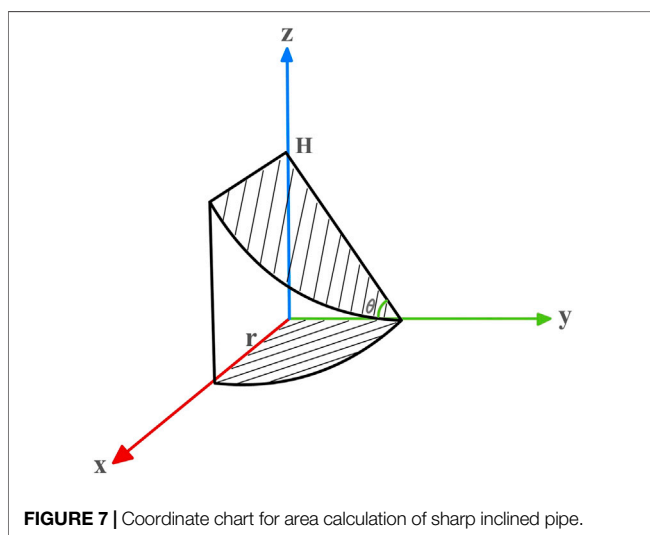


FIGURE 7 | Coordinate chart for area calculation of sharp inclined pipe.

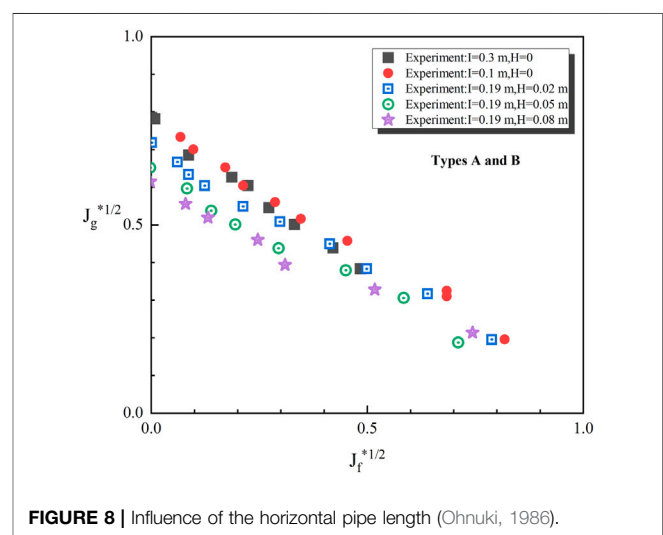


FIGURE 8 | Influence of the horizontal pipe length (Ohnuki, 1986).

is 0.38 m. And the 0.03 m inner diameter water supply pipe located at the bottom of the measuring tank. The experimental section was visualized, and the separator tank was opened to the atmosphere. During the experimental procedure, to obtain an initial holdup of water in the separator tank, a continuous high flow rate of air was first supplied into the measuring tank. The water level h in the separator tank was kept constant during the experiment. When the air quality was constant Q_g , the volume flow Q_f of falling water was calculated from the rising velocity of the water level in the measuring tank. After measured the Q_f for the specified Q_g , the water in the measuring tank was drained. And the procedure mentioned above was repeated.

Four pipe configurations were used in this study (Type A, Type B, Type C and Type D). Figure 3 to Figure 4 were schematic diagrams of the experimental pipes. In fact, the shape of upper exit of the PWR hot leg is a circular section. The shapes of elbow connecting the horizontal pipe and the inclined are smooth, and this section diameter is constant.

Type A was composed of inclined pipes only; Type B was composed of horizontal pipes connected to inclined pipes.

Therefore, an improved pipe type was used instead of the Type B to study the effects of the shapes of upper exit and elbow. The shape of upper exit of Type C was a circular section, which was an improvement on Type B; Type D had a double-edged connection with the inclined pipe, which was an improvement to Type B, as shown in Figure 4. And Table 1 shows the dimensions of the experimental section.

Condition I, II and IV to VI carry out an analysis of the effect of lengths of horizontal pipe. Condition III and VIII pay attention to the effect of shapes of upper exit of inclined pipe. Furthermore, condition VII and IX focus on the effect of shapes of bend connecting horizontal pipe with inclined pipe.

SIMULATION METHOD

The nodes division is shown in Figure 5. The inlet boundary, 210 and 310 components adopted the “TDV”; and the outlet boundary, 360 and 380 also adopted the “TDV”. Among them, 210 was used to simulate the air injection device, 310 was used

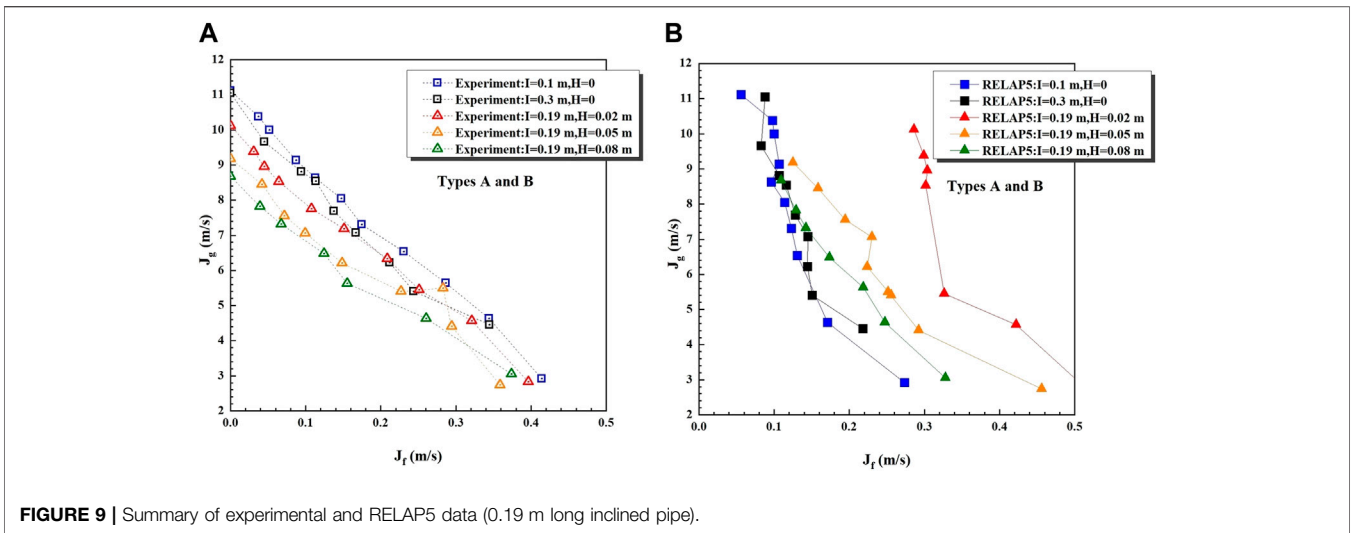


FIGURE 9 | Summary of experimental and RELAP5 data (0.19 m long inclined pipe).

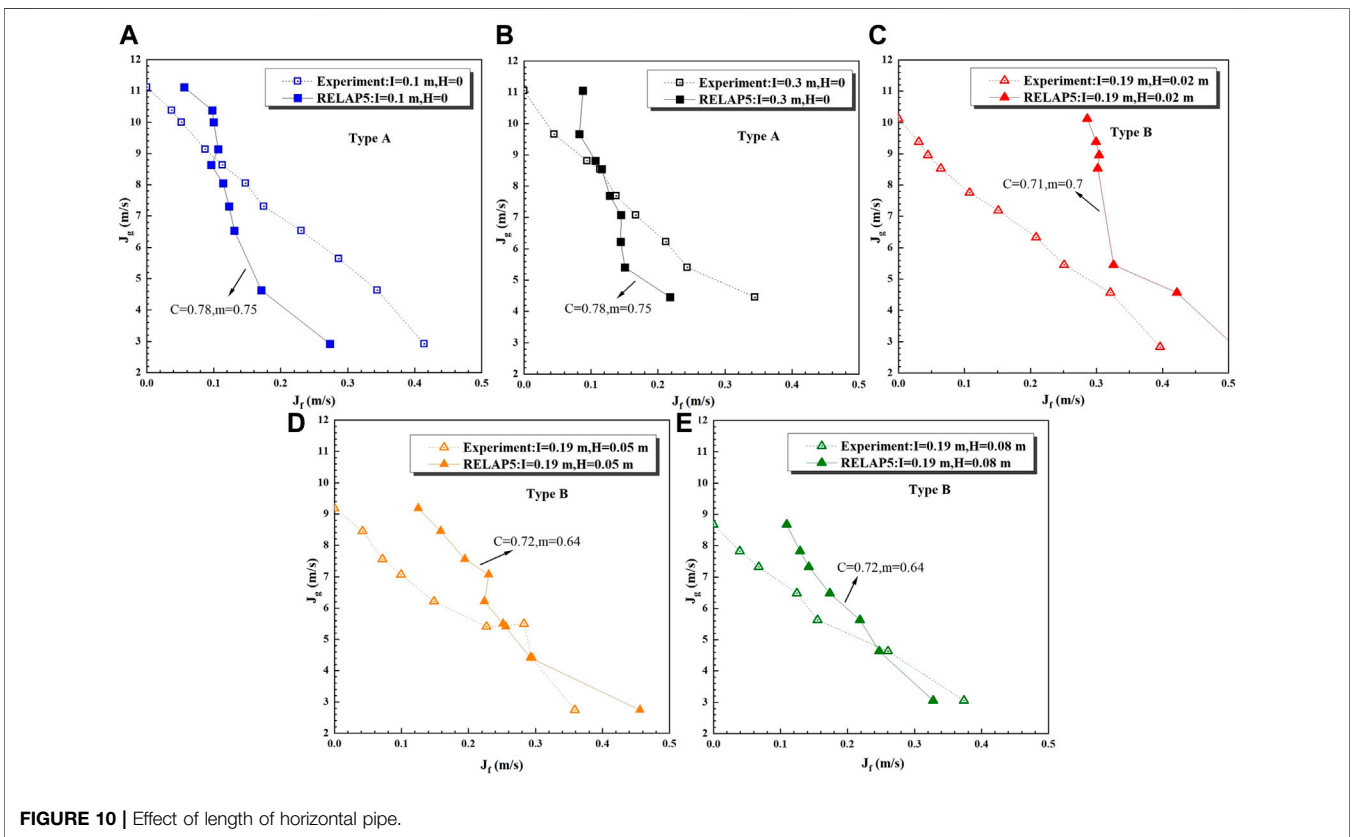


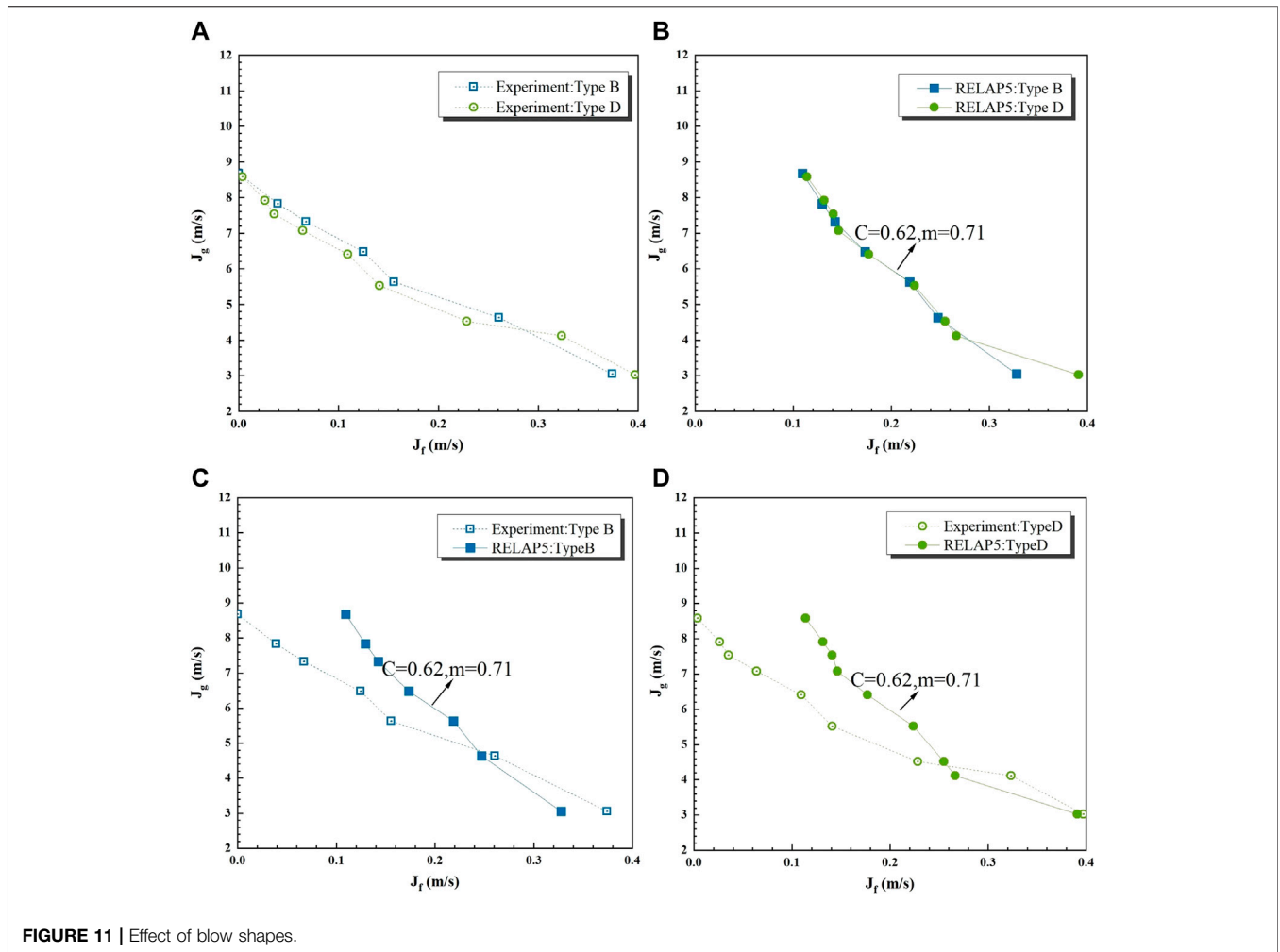
FIGURE 10 | Effect of length of horizontal pipe.

to simulate the liquid injection, 360 was the exhaust port, and the 380 was used to maintain a constant water head in the separator. The horizontal and inclined pipe were simulated by “PIPE” 100.

Noteworthy, in the modeling process, considering the difference in shapes of upper exit of inclined pipe, the simulation was carried out by changing the cross-sectional area of the exit, that was, Type A, Type B, and Type D were all

calculated by elliptical area, and Type C was calculated by circular area, as shown in the Figure 6. As the Type D had a smooth bend connecting horizontal pipe with inclined, the nodes were refined to achieve a better simulation effect. At the same time, the influence of the coefficient of local resistance was considered at the joints of different elbows.

According to Figure 7, the area of the ellipse formed by the oblique cut is calculated as follows (in this study, $\theta = 40^\circ$):



$$\frac{S}{4} = \int_0^r \frac{\sqrt{r^2 - x^2}}{\cos \theta} dx = \frac{1}{\cos \theta} \cdot \frac{\pi r^2}{4} \tag{1}$$

$$S = \frac{\pi r^2}{\cos \theta} \tag{2}$$

In the RELAP5/MOD3.3 program, if the CCFL model not turned on at the place where the gas-liquid counter current flow occurs, the “addition” and “subtraction” momentum equations are directly calculated simultaneously. Meanwhile, the program simulation do not take into consideration the effect of momentum loss caused by the instability of the liquid film; if the CCFL model turned on, this factor is considered.

The earliest Hewitt and Wallis (1963) passed the liquid down in a vertical acrylic resin tube with a diameter of 1.25 inch, and made the air flow up through the tube, in the opposite direction of the liquid, and then determined the minimum air flow rate required to cause the liquid to flow in the opposite direction upward. (that was, the flooding point), so obtain the empirical correlation related to CCFL.

Wallis number is a dimensionless number that characterizes the equilibrium relationship between momentum and static force. The following mainly analyzes the effect of the length H of the

horizontal pipe, connecting the inclined pipe on the relationship between $J_g^{*1/2}$ and $J_f^{*1/2}$, where J_g^* and J_f^* parameters are based on the Wallis correlation:

$$J_g^{*1/2} + m J_f^{*1/2} = C \tag{3}$$

where C is a geometrical parameter, the factor m and the C vary with the change of the test conditions such as the structure of the gas-liquid phase inlet and outlet of the vertical pipe, the size of the vertical pipe, and the physical properties of the fluid.

The currently developed CCFL model has three main forms in the program: Wallis correlation, Kutateladze correlation, and the form between Wallis and Kutateladze correlation (Bankoff et al., 1981).

Classical relation:

$$H_g^{1/2} + m H_f^{1/2} = C \tag{4}$$

where H_f is the dimensionless liquid flux, H_g is the dimensionless gas flux, C is the gas intercept (value of when $H_f = 0$, i.e., complete flooding), and m is the “slope”, that is the gas intercept divided by the liquid intercept (the value of when $H_g = 0$).

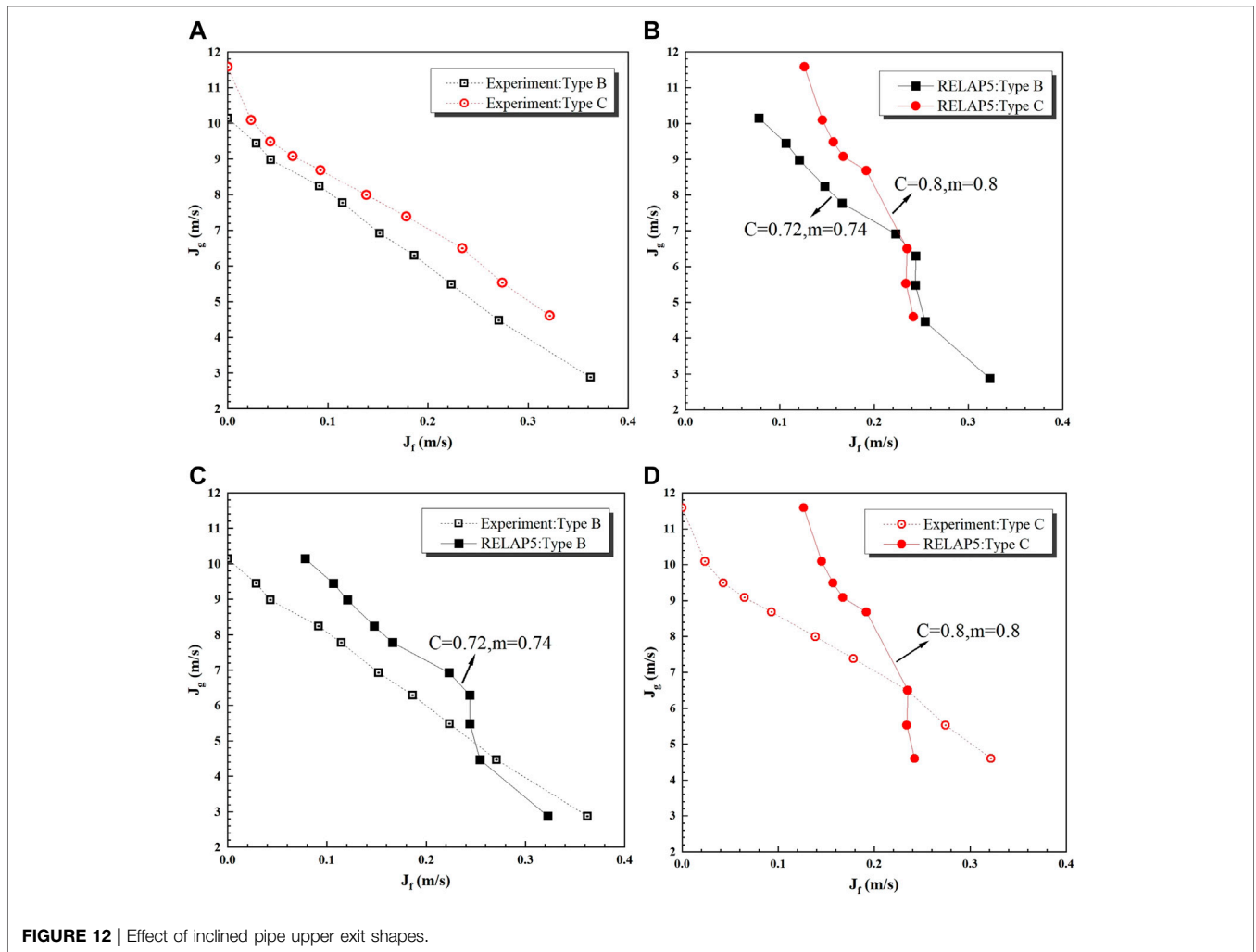


FIGURE 12 | Effect of inclined pipe upper exit shapes.

The dimensionless fluxes of gas and liquid have the following forms:

$$H_f = j_f \left[\frac{\rho_f}{g w (\rho_f - \rho_g)} \right]^{1/2} \tag{5}$$

$$H_g = j_g \left[\frac{\rho_f}{g w (\rho_f - \rho_g)} \right]^{1/2} \tag{6}$$

$$j_f = \alpha_f \gamma_f \tag{7}$$

$$j_g = \alpha_g \gamma_g \tag{8}$$

where j_f is the liquid superficial velocity, j_g is the gas superficial velocity, ρ_f is the liquid density, ρ_g is the gas density, α_f is the liquid volume fraction, α_g is the gas volume fraction, g is the gravitational acceleration.

w is the feature length, given by the following expression, (where β is a user input constant)

$$w = D_j^{1-\beta} L^\beta \tag{9}$$

D_j is the junction hydraulic diameter, and L is the Laplace capillary constant, given by the following form:

$$L = \left[\frac{\sigma}{g(\rho_f - \rho_g)} \right]^{1/2} \tag{10}$$

where σ is the surface tension, and in Eq. 9, β can be a number from 0 to 1.0.

By default, the Wallis correlation is used for small diameters and the Kutateladze correlation is used for large diameters. But the study of Tien et al. (Tien et al., 1979) believed that the correlations' conditions should be more stringent and flooding data obtained from measurements of nuclear power plant geometry. Then, an appropriate CCFL model for the input cards is generated.

RESULTS AND DISCUSSION

Effect of Lengths of Horizontal Pipe

This part mainly discusses the influence of horizontal pipe length on CCFL of hot leg. It was found that the case of inclined riser without horizontal pipe, J_f^* did not depend on the length l at the same J_g^* (Ohnuki, 1986), as shown in Figure 8.

As follows, **Figures 9, 10** showed the compares with the experimental data and RELAP5 simulation calculation data, respectively. It can be seen that the results calculated by RELAP5 are consistent with the change trend of the experimental data, but there are certain differences between the predicted data and the experimental data in some working conditions. If the pipe diameter and inclination angle of the inclined pipe remains constant in the hot leg, the initial flow rate keeps invariant, and the force between the gas and liquid phases and their effect maintains invariable. If the length of the horizontal pipe increases and the flow rate of the two phases remains unchanged, the action time and the distance between gas and liquid phases increases. And the gas phase consumes more energy in the liquid phase. Under the same liquid supply flow, a smaller air supply flow is required, the CCFL phenomenon occurs. It is worth noting that the “TDV” is used in the program calculation to specify the boundary conditions for the gas and liquid phase supply. In practice, if the pipe length is longer, the effect of gravity is magnified.

Effect of Shapes of Elbow

This section primarily discusses the effect of shapes of bend connecting horizontal pipe with inclined on CCFL of hot leg. **Figure 11** shows the effect of elbow shapes on the CCFL in experimental. The experimental data shows that the elbow shape had little effect on the CCFL, which is also reflected from the RELAP5 simulation results. Meanwhile, the trend of the simulated data is consistent with the experimental data, which is in good agreement. Due to the changes of the flow pattern of the two-phase flow at the junction of the horizontal pipe and the inclined pipe, the liquid in the elbow channel disturbs greatly at the junction, so it is easily carried by the gas flowing. This phenomenon becomes more obvious with the increase of liquid flow rate, which may affect the occurrence of CCFL phenomenon.

Effect of Shapes of Upper Exit of Inclined Pipe

Figure 12A shows the effect of different upper exit shapes of the inclined pipe on the CCFL experimental. Experimental data suggests that the upper exit shapes have an effect on CCFL, which are also reflected from the RELAP5 simulation results. Chung et al. (Tien et al., 1980) carried out an air-water CCFL test in a group of vertical pipes with the same pipe diameter and different outlet structures at the upper and lower ends. The lowest gas-phase flow rate occurs in the vertical pipe with the sharp-edged structure at the top; in the vertical pipe with the rounded top, the highest gas flow rate when CCFL occurs. **Figures 12B–D** have a similar situation in the inclined pipe. The J_f^* for the pipe with a circular upper exit is slightly larger than that of the sharp one at the same J_g^* .

CONCLUSION

Based on the simulation results in this paper, the following conclusions can be drawn:

- 1) Under the same water flow rate, the longer the horizontal pipe, the lower the gas flow rate at the beginning of the counter current flow. It should be noted that the program starts the

CCFL model, nodes division in the vicinity of the connecting pipe should be as detailed as possible to reduce the program calculation error. And a single component can be added as a transition when simulating “TDV”.

- 2) The simulation results show that the elbow has little effect on the CCFL phenomenon. But with the increase of the liquid flow rate, the liquid disturbance increases, which affects the CCFL in the elbow.
- 3) The results show that the upper outlet shapes have the effect on CCFL characteristics. Compared with the sharp injection method, the gentle injection method is not easy to cause the initial disturbance of the liquid phase in the exit area, and the gas flow rate required to initiate the CCFL initiation under the same conditions is larger.
- 4) The above simulation results show that the CCFL model in Wallis correlation has good applicability, and the influence of pipe lengths and pipe orifice shapes on the characteristics of CCFL can be considered through the parameter fitting of c and m in the CCFL model. As long as the model aims to build at different geometric structures, the CCFL characteristics of different geometric structures can be well considered, but the input data of the determined model parameters are required in advance. It should be noted that the model of Wallis parameters comes from the fitting of experimental data with a small aperture geometry, and the modeling effect of the model needs to be considered for the large aperture experimental section in the nuclear reactor accident analysis.

In order to understand the influencing factors of the CCFL phenomenon in the hot leg more clearly, this paper mainly carried out the simulation of the effect of lengths of horizontal pipe, the effect of shapes of elbow and the effect of shapes of upper exit of inclined pipe. And this paper evaluated the CCFL model in the system program and analyzed the influencing factors. Simultaneously, some special considerations and solutions for modeling hot leg segments were given.

DATA AVAILABILITY STATEMENT

The original contributions presented in the study are included in the article/Supplementary Material, further inquiries can be directed to the corresponding author.

AUTHOR CONTRIBUTIONS

XL contributed the simulation work and also contributed the original writing and revision. LG, CX and TW contributed conceptions. WS provided experimental literature surveys and contributed constructive discussions. WZ, LuZ, ZM, and LoZ contributed review and suggestions. LP contributed the supervision of the study.

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Conflict of Interest: Authors LG, CX and TW are employed by Nuclear Power Technology Research Institute Co., Ltd.

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NOMENCLATURE

g gravitational acceleration ($\text{m}\cdot\text{s}^{-2}$)

D pipe diameter (m)

K nondimensional group depending only on fluid properties

L length of the lower limb of the elbow (m); Laplace capillary constant

D_j junction hydraulic diameter (m)

Greek symbols

ρ_k density ($\text{kg}\cdot\text{m}^{-3}$)

j_k superficial velocity (m/s)

β constant

w feature length

Abbreviations

CCFL Counter current flow limitation

LOCA Loss of Coolant Accident

TDV Time Dependent Volume

RELAP Reactor Excursion and Leak Analysis Program

PWR Pressurized Water Reactor